International Standard

INTERNATIONAL ORGANIZATION FOR STANDARDIZATION+MEXDYHAPODHAR OPFAHИЗАЦИЯ ПО CTAHDAPTИЗАЦИИ+ORGANISATION INTERNATIONALE DE NORMALISATION

Plain bearings — Pressed bimetallic half thrust washers — Features and tolerances

Paliers lisses - Demi-flasques de butée bimétalliques découpés à la presse - Caractéristiques et tolérances

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<u>ISO 6526:1983</u> https://standards.iteh.ai/catalog/standards/sist/62886716-96cb-4d05-9f0b-11c162b29639/iso-6526-1983

UDC 621.822.5

Descriptors : bearings, plain bearings, washers (spacers), thrust washers, dimensions, dimensional tolerances.

Foreword

ISO (the International Organization for Standardization) is a worldwide federation of national standards bodies (ISO member bodies). The work of developing International Standards is carried out through ISO technical committees. Every member body interested in a subject for which a technical committee has been authorized has the right to be represented on that committee. International organizations, governmental and non-governmental, in liaison with ISO, also take part in the work.

Draft International Standards adopted by the technical committees are circulated to the member bodies for approval before their acceptance as International Standards by the ISO Council.

iTeh STANDARD PREVIEW International Standard ISO 6526 was developed by Technical Committee ISO/TC 123, Plain bearings, and was circulated to the member bodies in May 1982. s.iteh.ai)

It has been approved by the member bodies of the following countries: 1983

Austria	
Brazil	
Czechoslovakia	
Egypt, Arab Rep. of	
France	

https://standards.iteh.ai/catalog/standards/sist/62886716-96cb-4d05-9f0b-Germany, F.R. India Italy Poland Romania

11c162**Sweden**iso-6526-1983 **United Kingdom** USA USSR

No member body expressed disapproval of the document.

Plain bearings — Pressed bimetallic half thrust washers — Features and tolerances

1 Scope and field of application

This International Standard specifies the main features and lays down tolerances for pressed bimetallic half thrust washers having an outside diameter up to 160 mm.

11c162b29639/iso-6526-1983

NOTES

1 All the linear dimensions and tolerances are expressed in millimetres.

2 The main dimensions for the half thrust washers are not the subject of an International Standard.

2 Symbols

The following symbols are used in this International Standard: RD PREVEW

- D = outside diameter of the washer (standards.iteh.ai)
- d = inside diameter of the washer

 H_{D} = washer height https://standards.iteh.ai/catalog/standards/sist/62886716-96cb-4d05-9f0b-

 $e_{\rm T}$ = total washer thickness

 $E_{\rm D}$ = height at lug top

- $F_{\rm D}$ = height at lug root
- $A = \log width$
- α = groove side angle
- $G_{W} =$ groove width
- $G_{\rm E}$ = wall thickness at the back of the groove
- $G_{\rm X}$ = distance between groove and the washer axis
- r_1 = width of back chamfer or radius
- r_2 = lug and joint face radius and lug fillet radius
- r_3 = width of sliding surface chamfer or radius
- L_{1} = scalloped toe width at joint face
- t = depth of the sliding surface relief
- l = height of the sliding relief
- β = sliding surface relief angle at joint faces
- p =flatness limit



Blanking radius and joint face sliding surface relief

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Figure 1 – Half thrust washers with and without lug



Figure 2 – Blanking fall-away for scalloped toe thrust washers **iTeh STANDARD PREVIEW**

3 General tolerances

(standards.iteh.ai)

For dimensions without tolerance indication, the following values apply:

- linear dimensions: http://www.iteh.ai/catalog/standards/sist/62886716-96cb-4d05-9f0b-
- 11c162b29639/iso-6526-1983 angular dimensions: ± 5° ____

4 Tolerances for diameters and for heights

4.1 Tolerance for the outside diameter, D

 	 	 	 	<u> </u>	

Table 1

	D	Teleropee
Above	Up to and including	Tolerance
_	120	0 - 0,25
120	160	0 - 0,35

4.2 Tolerance for the inside diameter, d

Table 2

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Above	Up to and including	I olerance for a
	120	+ 0,25 0
120	160	+ 0,35

NOTE – The difference D-d should be greater than 7 $\times e_{\rm T}$

4.3 Tolerances for heights H_D and F_D

Table 3

D Above Up to and including		Tolerance for <i>H</i> _D	$F_{\rm D} = H_{\rm Dmin} - (r_{\rm 2max} + 0.5)$		
	120	0 0,20	0		
120	160	0 - 0,25	- 0,5		

5 Total thickness, e_T

Table 4

	ס	e _⊤ Preferred dimensions (original size)				Tolerance for <i>e</i> ⊤
Above	Up to and including	1,75	2	2,5	3	
- ,	80	x	x			0 - 0,05
80	120		x	x		0 - 0,06
120	160			x	x	0 - 0,07

NOTE — For over-sizes it is recommended to increase the total thickness by a 0,10 step to which the same tolerance as for the corresponding original size is applied.

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6 Locating lug

6.1 Lug width, A

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	D	. A		
Above	Up to and including	Preferred dimension	Tolerance	
—	80	8		
80	120	10	0,25	
120	160	12	-,	

6.2 Notch recess

The notch recesses to be mostly manufactured with a tolerance $J_s 13$.

6.3 Lug length

The length of the lug is determined by dimension $E_{\rm D}$ given in table 6.

Table 6

	D	ED		
Above	Up to and including	Preferred dimension*		
-	80	$H_{\rm D}$ + 5		
80	160	<i>H</i> _D + 8		

* Dimension $E_{\rm D}$ is left without a tolerance because it is the difference of two dimensions for which the normal tolerance of \pm 0,25 mm would apply.

NOTE — Lug design is usually as shown in figure 1 b), but washers can also be provided with an offset locating lug in order to avoid incorrect assembly.

7 Grooves

7.1 Groove width, G_W

	D	G _W		
Above	Up to Preferred dimension		Tolerance	
	60	3,5	+ 0.50	
60	160	4,5	0	

Table 7

7.2 Wall thickness at the back of the groove, $G_{\rm E}$

Tolerance for G_{E} : $\begin{array}{c} 0\\ -0,30 \end{array}$

7.3 Groove position (with respect to the axis), G_X



Joint faces 8

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8 Joint faces https://standards.iteh.ai/catalog/standards/sist/62886716-96cb-4d05-9f0b-Joint face forms are shown in figure 1 and also in figure 2 for scalloped to where $L_{Jmin} = \frac{D-d}{4}$ or 3 mm whichever is the wider.

9 Fillet radii and chamfers

9.1 Radius on lug and joint faces and lug fillet radius, r_2

Table 9

e	Т	Preferred maximum radius
Above	Up to and including	^r 2max
- 2,59		1
2,59	-	1,5

9.2 Joint face relief (Figure 1, section D-D)

It can be either a blanking radius or a relief the depth of which, t, should not exceed 30 % of the total thickness e_T. Another design is shown in figure 1 (centre, section D-D).

The angle β should not exceed 30°.

9.3 Chamfer or radius between the sliding surface and side faces, r_3

е	T	Maximum width of chamfer or radius
Above	Up to and including	on sliding surface, ^r 3 _{max}
÷	2,59	$0,1 \times \frac{(D-d)}{2}$
2,59	. –	$0,15\times\frac{(D-d)}{2}$

Table 10

9.4 Chamfer or radius between back and external side face, r_1

It can be either a blanking radius or a chamfer whose sharp edges shall be free of burrs. The latter can be at 45° and its width range can be $0.3 < r_1 < 0.6$ with a tolerance of ± 0.20 .

NOTES

- 1 Tool scoring due to chamfering operation is permissible on the lug and its depth can be equal to 0,15 mm over the maximum chamfer height.
- 2 The chamfer between the back and the inside face is not specified. It shall only be free of burrs.

10 Flatness

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Half washers shall slide (under gravity) between parallel plates set at $e_{\text{Tmax}} + p$ where p is given in table 11.

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_	80	0,10	
80	120	0,12	
120	160	0,15	

11 Surface roughness

No mention is made of surface roughness due to the wide range of materials used.