## INTERNATIONAL STANDARD

ISO 6687

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# Machinery for forestry — Winches — Performance requirements

## iTeh SMateriel forestier D Theuits E Exigences de performance (standards.iteh.ai)

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Reference number ISO 6687:1994(E)

### Foreword

ISO (the International Organization for Standardization) is a worldwide federation of national standards bodies (ISO member bodies). The work of preparing International Standards is normally carried out through ISO technical committees. Each member body interested in a subject for which a technical committee has been established has the right to be represented on that committee. International organizations, governmental and non-governmental, in liaison with ISO, also take part in the work. ISO collaborates closely with the International Electrotechnical Commission (IEC) on all matters of electrotechnical standardization.

Draft International Standards adopted by the technical committees are circulated to the member bodies for voting. Publication as an International Standard requires approval by at least 75 % of the member bodies casting VIEW a vote.

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This second edition cancels and replaces<sup>1010</sup>the<sup>9/i</sup>first<sup>87</sup>edition (ISO 6687:1982), of which it constitutes a technical revision.

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# Machinery for forestry — Winches — Performance requirements

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#### 1 Scope

This International Standard specifies a uniform87:1994

method of defining specifications. imeasurements and sist 2.466 depth 8 of flange  $h_1$ : Radial distance from the drum storage capacity, line pull and line specifications.  $h_1$ : Radial distance from the survinches used in tree harvesting machines. face on the cable drum barrel.

#### 2 Definitions

For the purposes of this International Standard, the following definitions apply (see figure 1 and table 1).

**2.1** barrel diameter,  $D_1$ : Diameter of the cable drum barrel.

**2.2 flange diameter**,  $D_2$ : Diameter of the cable drum flanges.

**2.3 distance between flanges**, *b*: Distance measured between the flanges of the cable drum, measured at half the difference between depth of flange and safety distance:

**2.5** safety distance,  $h_3$ : Outmost periphery of the flange or housing that is left free from cable.

**2.6 throat clearance**,  $h_2$ : Minimum distance from the barrel of the cable drum to the winch housing at any point located between the flanges of the cable drum.

NOTE 1 The throat clearance,  $h_2$ , is greater than the depth of flange,  $h_1$ .

#### **3** Performance requirements

#### 3.1 Symbols and units

See table 1 and figure 1.

Symbol	Definition	Unit	
<i>D</i> <sub>1</sub>	Barrel diameter	mm	
$D_2$	Flange diameter	mm	
b	Distance between flanges	mm	
<i>h</i> <sub>1</sub>	Depth of flange	mm	
h <sub>2</sub>	Throat clearance	mm	
L	Cable length	m	
h <sub>3</sub>	Safety distance	mm	
d	Cable diameter	mm	
F	Line pull	N	
п	Rotational frequency of input shaft	S-1	
Т	Torque on winch input shaft	N∙m	
R	Total gear reduction between the winch input shaft and the cable drum		
η	Efficiency of total gear reduction between input shaft and cable drum at the speed corresponding to the torque used for <i>T</i>		
V	Line speed	m/s	
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Fable 1 -	— Symbols	and units
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#### 3.2 Drum storage capacity

#### 3.2.1 Safety distance

The safety distance,  $h_3$ , to be used in the calculations below shall be equal to two cable diameters:

 $h_3 = 2d$ 

#### 3.2.2 Calculations

The length of cable, L, in metres, that can be stored on the cable drum taking into account the safety distance,  $h_3$ , shall be calculated using the following formula:

$$L = (D_1 + h_1 - h_3) \times (h_1 - h_3) \times b \times K \times 10^{-3}$$

where K is a factor, function of the diameter of cable used (see table 2), calculated as follows:

Table 2 — Factor, K, as function of cable

diameter, d

 $K = \frac{\pi}{\left(1,04d\right)^2}$ 

#### 3.3 Line pull

#### 3.3.1 Bare drum and full drum

Line pull, *F*, in newtons, shall be calculated using the following formulae:

a) Bare drum line pull:

$$F = \frac{2\ 000 \times T \times R \times \eta}{D_1 + d}$$

b) Full drum line pull:

$$F = \frac{2\ 000 \times T \times R \times \eta}{D_2 - (2h_3 + d)}$$

NOTE 2 The breaking strength of the cable used may be exceeded in the specification for the line pull.

#### 3.3.2 Rated line pull of winch

The rated line pull of the winch is the maximum approved bare drum line pull as specified by the manufacturer and calculated in accordance with 3.3.1.

d	K	3.3.3 Maximum calculated line pull of installed		
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9	0,035 (95dfbf04f2c9)	Sdfbf04f2c9iso-668Thegaaximum bare drum line pull shall be calculated		
10	0,029 0	the following conditions.		
11	0,024 0			
12	0,020 2	a) When the torque on the input shaft is influence		
13	0,017 2	by a torque converter, it is calculated for a sta condition while the engine is at full governor cor trol position		
14	0,014 8			
16	0,011 3			
18	0,008 96	b) When the torque on the input shaft is influenced		
19	0,008 05	by the transmission gear ratios, it is calculated f		
20	0,007 26	the gear giving the highest line pull.		
22	0,006 00			
24	0,005 04	c) When the torque on the input shaft is influenced		
26	0,004 30	by a fixed gear ratio only, it is calculated f		
28	0,003 70	maximum engine torque.		
32	0,002 84	NOTE 3 The total winch pull may be limited by the machine on which the winch is mounted.		
35	0,002 37			
36	0,002 24			
38	0.002 01	3.4 Line speed		

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The values of K allow for a normal oversize on cables of 4 %. The formula for calculation of K is based on uniform cable winding and will not give correct figures if cable is wound non-uniformly on the cable drum. 3.4.1 Bare drum and full drum

Line speed, v, in metres per second, shall be calculated using the following formulae:

a) Bare drum line speed

$$v = \frac{n(D_1 + d)}{318.4R}$$

b) Full drum line speed

$$v = \frac{n[D_2 - (2h_3 + d)]}{318,4R}$$

#### 3.4.2 Maximum line speed

The maximum bare drum and full drum line speed shall be calculated in accordance with 3.4.1 using the maximum speed available at the winch input shaft under no-load conditions and stabilized engine speed.

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