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**Plain bearings — Symbols —**

**Part 2:**  
Applications

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*Paliers lisses — Symboles —*

*Partie 2: Applications*

*ISO 7904-2:1995*

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## Foreword

ISO (the International Organization for Standardization) is a worldwide federation of national standards bodies (ISO member bodies). The work of preparing International Standards is normally carried out through ISO technical committees. Each member body interested in a subject for which a technical committee has been established has the right to be represented on that committee. International organizations, governmental and non-governmental, in liaison with ISO, also take part in the work. ISO collaborates closely with the International Electrotechnical Commission (IEC) on all matters of electrotechnical standardization.

Draft International Standards adopted by the technical committees are circulated to the member bodies for voting. Publication as an International Standard requires approval by at least 75 % of the member bodies casting a vote.

International Standard ISO 7904-2 was prepared by Technical Committee ISO/TC 123, *Plain bearings*.

ISO 7904 consists of the following parts, under the general title *Plain bearings* — *Symbols*:

- *Part 1: Basic symbols*
- *Part 2: Applications*

# Plain bearings — Symbols —

## Part 2: Applications

### 1 Scope

This part of ISO 7904 specifies practical applications of the general symbols laid down in ISO 7904-1 with regard to the calculations, design and testing of plain bearings.

ISO 7904-1 distinguishes between basic characters and additional signs. Additional signs are subscripts and superscripts. The symbols necessary for plain bearing calculations, design, manufacture and testing are combinations of the above-mentioned signs.

The symbols which have been found necessary for the calculations, design and testing of plain bearings are given in 3.1 and 3.2. They have been combined according to the recommendations given in ISO 7904-1. The list may be enlarged, if necessary.

Angles and directions of rotation are defined positively as rotating left-hand (counter-clockwise); the same applies to rotational frequencies, peripheral and angular velocities.

### 2 Normative reference

The following standard contains provisions which, through reference in this text, constitute provisions of this part of ISO 7904. At the time of publication, the edition indicated was valid. All standards are subject to revision, and parties to agreements based on this part of ISO 7904 are encouraged to investigate the possibility of applying the most recent edition of the standard indicated below. Members of IEC and ISO

maintain registers of currently valid International Standards.

ISO 7904-1:1994, *Plain bearings — Symbols — Part 1: Basic symbols.*

### 3 Symbols and terms

#### 3.1 Symbols (Roman alphabet)

|             |  |
|-------------|--|
| $A$         | heat-emitting surface (bearing housing); elongation at fracture  |
| $A_{lan}$   | land area  |
| $A_{lan}^*$ | relative land area   |
| $A_p$       | oil pocket area  |
| $A_S$       | area of cross-section  |
| $a$         | distance; acceleration; thermal diffusivity  |
| $a_F$       | distance between entrance of the gap and the location of the pivot point   |
| $a_F^*$     | relative distance between entrance of the gap and the location of the pivot point  |
| $a_M$       | off-set of bearing support   |
| $B$         | (breadth); nominal bearing width; effective bearing width at right angles to the direction of motion; diameter of a circular tilting pad |
| $B^*$       | relative width; width ratio  |

|             |  |             |   |
|-------------|--|-------------|---|
| $B_H$       | external width of bearing housing in axial direction   | $D$         | nominal bearing diameter (inside diameter of journal bearing; mean diameter of thrust-bearing carrier ring) |
| $B_{tot}$   | total bearing width at right angles to the direction of motion   | $D_B$       | twice the radius of lobe or pad in a multi-lobed and tilting pad journal bearing                            |
| $b_{ax}$    | width of axial outlet  | $D_{B,max}$ | maximum value of $D_B$  |
| $b_c$       | width of circumferential outlet  | $D_{B,min}$ | minimum value of $D_B$  |
| $b_G$       | width of oil groove; width of bleed groove   | $D_H$       | diameter of bearing housing   |
| $b_p$       | width of oil pocket  | $D_i$       | inside diameter of thrust-bearing carrier ring  |
| $C$         | nominal clearance; concentration; chamfer  | $D_J$       | shaft diameter  |
| $C^*$       | relative bearing clearance (also $\psi$ )  | $D_{J,max}$ | maximum value of $D_J$  |
| $C_B$       | difference between lobe or pad bore radius and shaft radius of a multi-lobed and tilting pad journal bearing | $D_{J,min}$ | minimum value of $D_J$  |
| $C_D$       | bearing clearance; bearing diametral clearance (difference between journal bearing bore and shaft diameter)  | $D_o$       | outside diameter of thrust-bearing carrier ring   |
| $\bar{C}_D$ | mean value of $C_D$  | $d$         | diameter; damping coefficient   |
| $C_{D,eff}$ | effective bearing diametral clearance  | $d_{cp}$    | diameter of capillaries   |
| $C_{D,max}$ | maximum value of $C_D$   | $d_L$       | oil hole diameter   |
| $C_{D,min}$ | minimum value of $C_D$   | $E$         | modulus of elasticity   |
| $C_{man}$   | clearance range due to machining tolerances of a multi-lobed journal bearing                                 | $E^*$       | parameter of elasticity   |
| $C_{max}$   | maximum clearance of multi-lobed bearing   | $E_B$       | modulus of elasticity of bearing material   |
| $C_{min}$   | minimum clearance of multi-lobed bearing   | $E_J$       | modulus of elasticity of rotor material (sliding surface)   |
| $C_R$       | bearing radial clearance (difference between journal bearing radius and shaft radius)                        | $E_{rsl}$   | resultant modulus of elasticity   |
| $\bar{C}_R$ | mean value of $C_R$  | $e$         | eccentricity (eccentricity between the axis of the shaft and the bearing axis)                              |
| $C_{R,eff}$ | effective bearing radial clearance   | $e^*$       | relative eccentricity (also $\varepsilon$ )   |
| $C_{R,max}$ | maximum value of $C_R$   | $e_B$       | eccentricity of sliding surfaces (segments) of a multi-lobed and tilting pad journal bearing                |
| $C_{R,min}$ | minimum value of $C_R$   | $e_F$       | eccentricity of shaft in direction of load of a multi-lobed journal bearing                                 |
| $C_{wed}$   | wedge depth of a multi-taper land bearing ("thrust bearing clearance")                                       | $F$         | bearing force (nominal load)  |
| $c$         | specific heat capacity; stiffness coefficient  | $F^*$       | bearing force parameter   |
| $c_J$       | flexural stiffness of shaft  | $F_E$       | bearing force (with EHD influence)  |
| $c_p$       | specific heat capacity (with $p$ constant)   | $F_E^*$     | bearing force parameter (with EHD influence)  |
|             |  | $F_{E,tr}$  | bearing force (with EHD influence) at the limit of boundary lubrication                                     |

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| $F_{E, tr}^*$ | bearing force parameter (with EHD influence) at the limit of boundary lubrication      | $h^*$               | relative local lubricant film thickness (relative film thickness)                                  |
| $F_{eff}^*$   | effective bearing force parameter  | $h_{en}$            | lubricant film thickness at entrance   |
| $F_f$         | friction force   | $h_{ex}$            | lubricant film thickness at exit   |
| $F_f^*$       | friction force parameter   | $h_G$               | depth of oil groove  |
| $F_n$         | normal force; normal to sliding surface  | $h_{lim}$           | minimum permissible lubricant film thickness during operation                                      |
| $F_{rot}$     | proportion of bearing force absorbed by the rotation of the rotor (wedge action)       | $h_{lim}^*$         | minimum permissible relative lubricant film thickness during operation                             |
| $F_{sc}$      | static load  | $h_{lim, tr}$       | minimum permissible lubricant film thickness at transition to boundary lubrication                 |
| $F_{sq}$      | proportion of bearing force absorbed by displacement due to squeezing (squeeze action) | $h_{lim, tr}^*$     | minimum permissible relative lubrication film thickness at transition to boundary lubrication      |
| $F_{st}$      | bearing force at start ( $N \approx 0$ )   | $h_{min}$           | minimum lubricant film thickness (minimum film thickness)  |
| $F_{stp}$     | bearing force at stop ( $N \approx 0$ )  | $h_{min}^*$         | relative minimum lubricant film thickness (relative minimum film thickness)                        |
| $F_{tr}$      | bearing force (without EHD influence) at the limit of boundary lubrication             | $h_{min, tr}$       | minimum lubricant film thickness at transition to boundary lubrication                             |
| $F_{tr}^*$    | bearing force parameter (without EHD influence) at the limit of boundary lubrication   | $h_{min, tr}^*$     | relative minimum lubricant film thickness at transition to boundary lubrication                    |
| $f$           | coefficient of friction; function  | $h_p$               | depth of oil pocket  |
| $f^*$         | friction parameter   | $h_{wav}$           | waviness of sliding surface  |
| $f_h$         | fluid friction coefficient (in the area of boundary lubrication)                       | $h_{wav, eff}$      | effective waviness of sliding surface  |
| $f_{min}$     | coefficient of friction on minimum of Stribeck curve                                   | $h_{wav, eff, lim}$ | maximum permissible effective waviness of sliding surface  |
| $f_s$         | coefficient of friction of a solid   | $h_0$               | local lubricant film thickness with $\varepsilon = 0$  |
| $f_{tr}$      | coefficient of friction at transition to boundary lubrication                          | $h_0^*$             | relative local lubricant film thickness with $\varepsilon = 0$                                     |
| $G$           | shear modulus  | $h_{0, max}$        | maximum lubricant film thickness with $\varepsilon = 0$  |
| $g$           | acceleration due to gravity  | $h_{0, max}^*$      | lubricant film thickness ratio (relative maximum lubricant film thickness with $\varepsilon = 0$ ) |
| $H$           | nominal height   | $K_w$               | coefficient of wear  |
| $H_H$         | height of bearing housing  | $k$                 | heat transmission coefficient  |
| HB            | Brinell hardness   | $k^*$               | heat transmission parameter  |
| HRB           | Rockwell hardness (ball)   | $k_A$               | outer heat transmission coefficient (reference area A)   |
| HRC           | Rockwell hardness (cone)   |                     |  |
| HV            | Vickers hardness   |                     |  |
| $h$           | local lubricant film thickness (film thickness)  |                     |  |

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| $k_i$        | inner heat transmission coefficient (oil film)   | $P_{th,f}$       | heat flow rate based on frictional power   |
| $L$          | nominal length; length of sliding surface in direction of motion; length of pad in circumferential direction | $P_{th,L}$       | heat flow rate in the lubricant  |
| $L_H$        | length of bearing housing at right angles to the axis  | $P_{tot}$        | total power ( $P_p + P_f$ )  |
| $l_{ax}$     | axial land length  | $P_{tot}^*$      | total power parameter  |
| $l_c$        | circumferential land length  | $p$              | local lubricant film pressure, e.g. specific load  |
| $l_{cp}$     | length of capillaries  | $\bar{p}$        | specific load, e.g. load per unit of projected area  |
| $l_G$        | length of oil groove   | $\bar{p}_{dyn}$  | dynamic specific load  |
| $l_p$        | length of oil pocket   | $p_{en}$         | lubricant feed pressure  |
| $l_{wed}$    | length of wedge  | $p_{en}^*$       | lubricant feed pressure parameter  |
| $M$          | moment; mixing factor  | $p_{lim}$        | maximum permissible lubricant film pressure  |
| $M_F$        | loading moment   | $\bar{p}_{lim}$  | maximum permissible specific bearing load  |
| $M_f$        | friction moment  | $p_{max}$        | maximum lubricant film pressure  |
| $m$          | mass   | $p_{max}^*$      | maximum lubricant film pressure parameter  |
| $N$          | rotational frequency (revolutions per time unit)   | $p_p$            | lubricant pressure in pockets  |
| $N^*$        | rotational frequency parameter   | $p_{sc}$         | static specific load   |
| $N_B$        | rotational frequency of the bearing  | $\bar{p}_{st}$   | specific load at start ( $N \approx 0$ )   |
| $N_{cr}$     | critical rotational frequency of the rigidly supported shaft   | $\bar{p}_{st}^*$ | specific load at stop ( $N \approx 0$ )  |
| $N_F$        | rotational frequency of the bearing force  | $Q$              | lubricant flow rate; volume flow rate  |
| $N_J$        | rotational frequency of the shaft  | $Q^*$            | lubricant flow rate parameter  |
| $N_{lim,tr}$ | maximum permissible transition rotational frequency  | $Q_{cl}$         | cooling oil flow rate  |
| $N_{min}$    | rotational frequency at minimum of friction of Stribeck curve  | $Q_p$            | lubricant flow rate based on feed pressure   |
| $N_{rsn}$    | resonance rotational frequency of the shaft assembled in a plain bearing                                     | $Q_p^*$          | lubricant flow rate parameter based on feed pressure   |
| $N_{tr}$     | transition rotational frequency  | $Q_0$            | reference lubricant flow rate  |
| $P_{cl}$     | cooling capacity; additional cooling   | $Q_1$            | lubricant flow rate at the inlet to lubrication clearance gap (circumferential direction)            |
| $P_f$        | frictional power   | $Q_1^*$          | lubricant flow rate parameter at the inlet to lubrication clearance gap (circumferential direction)  |
| $P_p$        | pumping power  | $Q_2$            | lubricant flow rate at the outlet of lubrication clearance gap (circumferential direction)           |
| $P_{th}$     | heat flow rate   | $Q_2^*$          | lubricant flow rate parameter at the outlet of lubrication clearance gap (circumferential direction) |
| $P_{th,amb}$ | heat flow rate to the ambient  |                  |  |

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| $Q_3$        | lubricant flow rate due to hydrodynamic pressure development                        | $s$          | wall thickness   |
| $Q_3^*$      | lubricant flow rate parameter due to hydrodynamic pressure development              | $s_{A,rsn}$  | displacement amplitude of rotor vibration at resonance   |
| $R$          | bearing inside radius of journal bearing  | $T$          | temperature  |
| $R_a$        | mean value of surface finish C.L.A.   | $T_{amb}$    | ambient temperature  |
| $R_{a,B}$    | mean value of surface finish C.L.A. of bearing sliding surface                      | $T_B$        | bearing temperature  |
| $R_{a,J}$    | mean value of surface finish C.L.A. of shaft mating surface                         | $T_{eff}$    | effective temperature of the lubricant   |
| $R_B$        | lobe or pad radius of a multi-lobed journal bearing and tilting pad journal bearing | $T_{en}$     | lubricant temperature at bearing entrance  |
| $R_{cp}$     | flow resistance of capillaries (hydrostatic bearing)                                | $T_{ex}$     | lubricant temperature at bearing exit  |
| $R_J$        | shaft radius  | $T_g$        | glass temperature (plastic testing)  |
| $R_{lan,ax}$ | flow resistance of one land in axial direction (hydrostatic bearing)                | $T_J$        | shaft temperature  |
| $R_{lan,c}$  | flow resistance of one land in circumferential direction (hydrostatic bearing)      | $T_L$        | lubricant temperature  |
| $R_p$        | flow resistance of one pocket (hydrostatic bearing)                                 | $T_{lim}$    | maximum permissible bearing temperature  |
| $R_z$        | average peak-to-valley height   | $T_1$        | lubricant temperature in pockets   |
| $R_{z,B}$    | average peak-to-valley height of bearing sliding surface                            | $T_2$        | lubricant temperature at the exit of the bearing clearance gap   |
| $R_{z,J}$    | average peak-to-valley height of shaft mating surface                               | $t$          | time   |
| $Re$         | Reynolds number   | $U$          | peripheral speed; sliding velocity (related to the journal bearing shaft diameter or the mean thrust bearing carrier ring) |
| $Re_{cr}$    | critical Reynolds number  | $U_B$        | peripheral speed of bearing  |
| $r$          | repeatability   | $U_J$        | peripheral speed of shaft  |
| $S_F$        | security against boundary lubrication due to excessive loading                      | $U_{lim,tr}$ | maximum permissible transition peripheral speed  |
| $S_N$        | security against boundary lubrication due to lower frequency of rotation            | $\bar{U}_R$  | average velocity of flow at pre-restrictor of hydrostatic bearing  |
| $So$         | Sommerfeld number   | $U_{tr}$     | transition peripheral speed  |
| $So_{rot}$   | Sommerfeld number (rotation)  | $u$          | velocity component in $x$ -direction; deformation in $x$ -direction; uncertainty of measurement                            |
| $So_{sq}$    | Sommerfeld number (displacement due to squeezing)                                   | $V$          | volume; surface velocity in $y$ -direction; displacement velocity  |
| $So_{tr}$    | Sommerfeld number at transition to boundary lubrication                             | VG           | viscosity grade  |
|              |   | VI           | viscosity index  |
|              |   | $v$          | velocity component in $y$ -direction; deformation in $y$ -direction  |
|              |   | $W$          | surface velocity in $z$ -direction; work (energy)  |

|           |   |              |  |
|-----------|---|--------------|--|
| $w$       | velocity component in $z$ -direction; deformation in $z$ -direction   | $\bar{\eta}$ | mean dynamic viscosity of the lubricant at clearance gap |
| $w_{amb}$ | velocity of air surrounding the bearing housing   | $\eta_{eff}$ | effective dynamic viscosity of the lubricant             |
| $x$       | coordinate parallel to the sliding surface, in circumferential direction  | $\kappa$     | resistance ratio   |
| $y$       | coordinate normal to the sliding surface  | $\lambda$    | thermal conductivity                                     |
| $Z$       | number of sliding surfaces (pads) or pockets per bearing; necking after fracture  | $\mu$        | relative stiffness of the bearing                        |
| $z$       | coordinate parallel to the sliding surface, normal to circumferential direction (for journal bearings in axial direction, for thrust bearings normal to the shaft axis) | $\nu$        | kinematic viscosity of lubricant; Poisson's ratio        |
|           |   | $\nu_B$      | Poisson's ratio (bearing)                                |
|           |   | $\nu_J$      | Poisson's ratio (shaft)                                  |
|           |   | $\xi$        | restrictor ratio (hydrostatic bearing)                   |

**3.2 Symbols (Greek alphabet)**

|                 |  |                |   |
|-----------------|--|----------------|---|
| $\alpha$        | heat transfer coefficient  | $\Pi$          | product; parameter  |
| $\alpha_l$      | linear heat expansion coefficient  | $\pi$          | Ludolf's number ( $\pi = 3,141592 \dots$ )                      |
| $\alpha_{l,B}$  | linear heat expansion coefficient of the bearing   | $\rho$         | density   |
| $\alpha_{l,J}$  | linear heat expansion coefficient of the shaft   | $\sigma$       | normal stress; standard deviation                               |
| $\alpha_V$      | cubic heat expansion coefficient   | $\tau$         | shearing stress   |
| $\beta$         | attitude angle (angular position of shaft eccentricity related to the direction of load), temperature viscosity exponent | $\Phi$         | sliding surface utilization factor                              |
| $\beta_{h,min}$ | angle between direction of load and position of minimum lubricant film thickness   | $\phi$         | angular coordinate in circumferential direction                 |
| $\gamma$        | angular position of bearing load (bearing load in vertical direction: $\gamma = 0$ )                                     | $\psi$         | relative bearing clearance (also $C^*$ )                        |
| $\Delta$        | difference; Laplace operator   | $\bar{\psi}$   | mean relative bearing clearance                                 |
| $\delta$        | angular position of smallest lubrication clearance gap   | $\psi_{eff}$   | effective relative bearing clearance                            |
| $\delta_B$      | angle of misalignment of the bearing (angular deviation of bearing)  | $\psi_{man}$   | relative manufacturing clearance of multi-lobed journal bearing |
| $\delta_J$      | angle of misalignment of the shaft (angular deviation of the shaft)  | $\psi_{max}$   | maximum value of $\psi$   |
| $\varepsilon$   | relative eccentricity (also $e^*$ ), relative elongation   | $\psi_{min}$   | minimum value of $\psi$   |
| $\zeta$         | hydraulic resistance coefficient   | $\psi_{20}$    | relative bearing clearance at 20 °C (journal bearing)           |
| $\eta$          | dynamic viscosity of the lubricant   | $\Omega$       | angular span of bearing sliding surface (segment)               |
|                 |  | $\omega$       | angular velocity ( $\omega = 2 \cdot \pi \cdot N$ )             |
|                 |  | $\omega_B$     | angular velocity of bearing                                     |
|                 |  | $\omega_h$     | angular velocity (hydrodynamic)                                 |
|                 |  | $\omega_J$     | angular velocity of shaft                                       |
|                 |  | $\omega_{rel}$ | relative angular velocity                                       |



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