INTERNATIONAL STANDARD



INTERNATIONAL ORGANIZATION FOR STANDARDIZATION ORGANISATION INTERNATIONALE DE NORMALISATION MEЖДУНАРОДНАЯ ОРГАНИЗАЦИЯ ПО СТАНДАРТИЗАЦИИ

Synchronous belt drives - Automotive belts

Transmissions synchrones par courroies - Courroies pour la construction automobile

iTeh STANDARD PREVIEW (standards.iteh.ai)

ISO 9010:1987 https://standards.iteh.ai/catalog/standards/sist/6e0a5be8-b177-4754-9cb4c68f0c891add/iso-9010-1987

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41

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Foreword

ISO (the International Organization for Standardization) is a worldwide federation of national standards bodies (ISO member bodies). The work of preparing International Standards is normally carried out through ISO technical committees. Each member body interested in a subject for which a technical committee has been established has the right to be represented on that committee. International organizations, governmental and non-governmental, in liaison with ISO, also take part in the work.

Draft International Standards adopted by the technical committees are circulated to the member bodies for approval before their acceptance as International Standards by the ISO Council. They are approved in accordance with ISO procedures requiring at least 75 % approval by the member bodies voting.

iTeh ST International Standard ISO 9010 was prepared by Technical Committee ISO/TC 41, Pulleys and belts (including veebelts).

Users should note that all International Standards undergo revision from time to time and that any reference made herein to any other International Standard implies its 77-4754-9cb4 latest edition, unless otherwise stated and ards.iteh.ai/catalog/standards/sist/6e0a5be c68f0c891add/iso-9010-1987

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Synchronous belt drives — Automotive belts

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1 Scope and field of application

Both types of belt are characterized by their tooth dimensions (profile), the pitch $p_{\rm b}$ being 9,525 mm¹).

This International Standard specifies the principal characteristics of synchronous endless bets for use in automotive applications such as engine camshaft drives. $\frac{150,9010;1987}{charac-}$

The principal characteristics include

- a) nominal tooth dimensions;
- b) pitch spacing;
- c) width tolerances;
- d) pitch length tolerances;
- e) pitch length measuring specifications.

2 Belt types

Two belt types for synchronous drives for automotive application are standardized :

- type ZA light-duty automotive belt;
- type ZB heavy-duty automotive belt.

A belt is designated by a series of numbers and letters as follows :

- a) the first set of numbers indicates the number of teeth;
- b) the first letter indicates tooth pitch;
- c) the second letter indicates tooth profile;
- d) the second set of numbers indicates the width in millimetres.

Example:

	<u>80</u>	Z	A	<u>19</u>
Number of teeth				
Tooth pitch				
Tooth profile				
Width (mm)				

1) Carried to the third decimal place because belt pitch is a defined value.

Dimensions and tolerances 4

4.1 Belt tooth dimensions

The nominal belt tooth dimensions are shown in figure 1 and given in table 1.

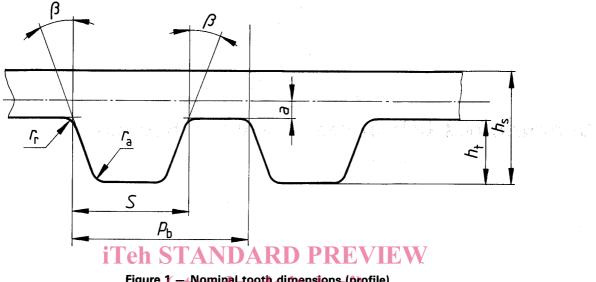


Figure 1 - Nominal tooth dimensions (profile)

	ISO 9010:1987	
https://standards.lien.ai/ca	Nominal tooth dimensions	177-4754-9cb4-

Term	Symbol	Nominal profile			
	Gymbol	Type ZA	Type ZB		
Pitch	p _b	9,525	9,525		
Tooth angle	2β	40	40		
Height	h _s	4,1	4,5		
Pitch line differential	а	0,686	0,686		
Root radius	r_r	0,51	1,02		
Tip radius	ra	0,51	1,02		
Tooth height	h _t	1,91	2,29		
Tooth width	S	4,65	6,12		

c68f(Dimensions in millimetres 7 angle in degrees

4.2 Belt pitch length and tolerances

The number of teeth, z, i.e., the belt pitch length, L_{p} , shall be agreed between the parties concerned. Pitch length tolerances are given in table 2.

Number of teeth	Belt pitch length $L_{ m p}$					
Z	Range	Tolerance				
<i>z</i> < 40	L _p < 381	± 0,45				
41 < <i>z</i> < 53	$390,525 \le L_p \le 504,825$	± 0,5				
$54 \leq z \leq 80$	$514,35 \le L_p \le 762$	± 0,6				
81 < <i>z</i> < 104	$771,525 < L_p < 990,6$	± 0,65				
105 < <i>z</i> < 128	$1\ 000,125 \le L_p \le 1\ 219,2$	± 0,75				
129 < <i>z</i> < 160	$1\ 228,725\ <\ L_{\rm p}\ <\ 1\ 524$	± 0,8				
161 <i>≤ z ≤</i> 187	$1533,525 \le L_p \le 1781,175$	± 0,85				
188 < <i>z</i> < 213	$1790.7 \le L_p \le 2028.825$	± 0,9				
214 < <i>z</i> < 240	$2\ 038,35 \le L_{\rm p} \le 2\ 286$	± 0,95				
241 < z < 267	$2\ 295,525 \le L_p \le 2\ 543,175$.	± 1				

Table 2 — Pitch length tolerances

Dimensions and tolerances in millimetres

4.3 Belt widths and tolerances STANDARD PREVIEW

The belt width, b_s , shall be agreed between the parties concerned. Width tolerances are given in table 3.

Table 3 TWidth tolerances

https://standards.iteh.ai/catalog/standardsDimensionsland tolerances.in millimetres

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	Tolerances				
Range	Range of belt	pitch lengths			
nunge	$L_{\rm p} < 840$ (z < 88)	$L_{\rm p} \ge 840$ $(z \ge 89)$			
	$(z \leq 88)$	$(z \ge 89)$			
b _s < 40	± 0,8	± 0,8			
$b_{\rm s} > 40$	± 0,8	+ 0,8 - 1,3			

NOTE - For special applications, smaller tolerances can be adopted.

5 Pitch length measurement

5.1 Measuring fixture (see figure 3)

The pitch length of a synchronous belt shall be determined by placing the belt on a measuring fixture composed of the following elements.

5.1.1 Two pulleys of equal diameter, as specified in table 4, of the proper belt type and having standard tooth space dimensions. These pulleys should be made to the tolerances shown in table 4 and have the proper clearance, $C_{\rm m}$, between the pulley tooth space and the theoretical belt tooth width as specified in table 4 (see figure 2). One pulley shall be free to rotate on a fixed-position shaft, while the other shall be free to rotate on a movable shaft to permit the centre distance to change.

5.1.2 Means of applying a total measuring force to the movable pulley.

5.1.3 Means of measuring the centre distance between the two pulleys with the necessary degree of accuracy to check

the allowed tolerances (tolerances for centre distance measurement should be one-half of the allowed length tolerances in table 2).

5.2 Total measuring force

The total measuring force, F_t , to be applied for measuring belts shall be calculated as follows:

$$F_{\rm t} = (b_{\rm s} \times 29) - 100$$

where

 $b_{\rm s}$ is the width in millimetres;

 $F_{\rm t}$ is the total force, in newtons.

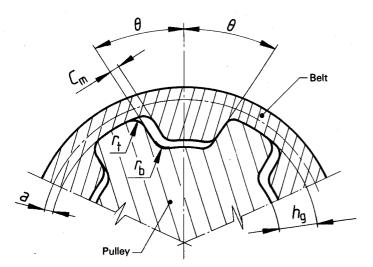
5.3 Procedure

In measuring the pitch length of a synchronous belt, the belt should be rotated at least two revolutions to seat it properly and to divide the total force equally between the two lengths of the belt.

The pitch length shall be calculated by adding the pitch circumference of one of the pulleys to twice the measured centre distance.

Table 4 – Belt pitch length measuring pulleys

Pitch	Number of teeth	Pitch circumference	Outside a diameter	Run	out	Clear- ance	al)				
code	z	$p_{\rm b} imes z$	d _o	radial	axial	$_7 C_{\rm m}$	h _g	r _b	r _t	2 <i>a</i>	θ
		mm	mm	mm		mm	mm	mm	mm	mm	degrees
ZA	20	190,5	59,266 ± 0,013	0,013	0,025	0,33	2,68 ± 0,1	0,85 ± 0,1	0,85 ± 0,1	1,372	20 ± 1,5
ZB	20	190,5	59,266 ± 0,013	0,013	0,025	0,38	3 ± 0,1	1,23 ± 0,1	1,23 ± 0,1	1,372	20 ± 1,5



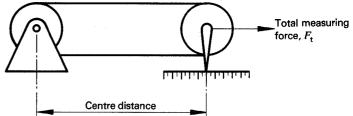


Figure 3 — Pitch length measuring fixture



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