TECHNICAL REPORT



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Plain bearings — Testing of the tribological behaviour of bearing materials for oil lubrication application — Running-in under mixed lubrication conditions

iTeh STANDARD PREVIEW

Paliers lisses a Essai de propriétés tribologiques des matériaux pour paliers lubrifiés à l'huile – Aptitude au rodage dans des conditions de lubrification en régime mixte

<u>ISO/TR 9993:1989</u> https://standards.iteh.ai/catalog/standards/sist/03d54d72-fbe8-4a01-89f0-25d9bd0be702/iso-tr-9993-1989

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Reference number ISO/TR 9993 : 1989 (E)

Foreword

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The main task of ISO technical committees is to prepare International Standards. In exceptional circumstances a technical committee may propose the publication of a technical report of one of the following types: STANDARD PREVIEW

 type 1, when the necessary support within the technical committee cannot be obtained for the publication of an International Standard, despite repeated efforts;

type 2, when the subject is still under technical development)requiring wider
exposure;
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 type 3, when a technical committee has collected data of a different kind from that which is normally published as an International Standard ("state of the art", for example).

Technical reports are accepted for publication directly by ISO Council. Technical reports of types 1 and 2 are subject to review within three years of publication, to decide whether they can be transformed into International Standards. Technical reports type 3 do not necessarily have to be reviewed until the data they provide are considered to be no longer valid or useful.

ISO/TR 9993, which is technical report of type 2, was prepared by Technical Committee ISO/TC 123, *Plain bearings.*

Annex A of this Technical Report is for information only.

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Introduction

The surfaces of a journal and a bearing in the bearing designed for operation under hydrodynamic lubrication conditions are not always fully separated with a continuous oil film forming a hydrodynamic film. It causes severe wear and could cause bearing seizure. Wear of the bearing and the likelihood of its seizing are determined by a relative period of bearing operation under insufficient lubrication conditions, and by the relation of the friction surface areas operating under fluid and boundary lubrication and under direct contact.

The ability of a bearing to form a hydrodynamic and boundary oil film is ensured by its design and by the bearing material antifriction (tribological) properties, but mainly by their running-in ability. The higher the running-in ability and running-in time, the shorter the period of operation under alternating load during which mixed lubrication takes place and the greater is the range of the loads which ensure hydrodynamic and boundary effects ds.iteh.ai)

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Plain bearings — Testing of the tribological behaviour of bearing materials for oil lubrication application — Running-in under mixed lubrication conditions

1 Scope

This Technical Report specifies a test procedure for evaluating the running-in ability of bearing materials under mixed lubrication conditions.

It does not apply to materials of hardness less than 10 HB and with overlay thickness less than 0,02 mm.

The test procedure helps to compare the running in ability of the tested system, bearing material + mating surface material + lubricating oil, with the other combinations of the above components to help choose a system with the required properties of serviceability.

ISO/TR 9993:

2 Normative reference

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The following standard contains provisions which, through reference in this text, constitute provisions of this Technical Report. At the time of publication, the edition indicated was valid. All standards are subject to revision, and parties to agreements based on this Technical Report are encouraged to investigate the possibility of applying the most recent edition of the standard indicated below. Members of IEC and ISO maintain registers of currently valid International Standards.

ISO 468 : 1982, Surface roughness — Parameters, their values and general rules for specifying requirements.

3 Symbols and units

See table 1.

Symbol	Description	Unit
и	Sliding velocity	m/s
f	Coefficient of friction	-
F _n	Normal force	N
р	Pressure	Pa
T _{oil}	Oil bath temperature	°C
С	Loading rate	Pa/min
C_1	$= 2 \times 10^5$	Pa/min
C2	$= 5 \times 10^2$	Pa/min
R _{HT} , a	Temperature ratio constants	-
R _H	Hydrodynamic index of running-in ability	-

4.3 hydrodynamic running-in ability: The property of the bearing material to restore or to maintain a hydrodynamic oil film during the running-in period.

4.1 running-in: Process of changing the rubbing surfaces

geometry as well as the physical and mechanical properties of material surface layers during the initial period of friction,

usually displaced under constant outside conditions as a

4.2 orunning in ability: The property of the bearing material

to reduce friction force, temperature and wear intensity during

decrease of friction force, temperature and wear intensity.

4.4 hydrodynamic index of running-in ability, $R_{\rm H}$: The ratio

 $\frac{p_2}{p_1}$

Definitions

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the running-in period.

where p_1 and p_2 are pressure values corresponding to the minimum of a function of a coefficient of friction and pressure at loading rates $C_1 = 2 \times 10^5$ Pa/min and $C_2 = 5 \times 10^2$ Pa/min respectively.

It characterizes the ability of materials to form a hydrodynamic oil film during friction under mixed lubrication conditions.

5 Principle

A rotating cylindrical specimen and a stationary specimen made of the tested bearing material are immersed in an oil bath heated up to the preset temperature. A rotating specimen is brought up to the preset angular frequency. A load is applied to the specimen at low and high speeds until seizure occurs. A coefficient of friction is constantly recorded during loading. The running-in ability of the bearing alloys is evaluated from the value of the appropriate coefficient of friction.

Equipment and materials 6

Equipment 6.1

The testing apparatus shall permit

a rotation of a cylindrical specimen at a frequency which gives a linear velocity on its cylindrical surface of 1 m/s with an error not exceeding 5 %;

the specimen to be maintained under an alternating force under a continuous pressure increase at the rates C_1 \pm 10 % and C_2 \pm 10 %, starting at a pressure of 10^{-3} Pa;

continuous measurement of the specimen loading and of the coefficient of friction with the error not exceeding 5 %.

NOTE - Recording devices may be used for continuous recording of the friction forces moment and normal load; the error should not exceed 5 %.

Figure 1 shows the mounting of the specimens. The fixing mechanism for the stationary specimen shall position it as shown in figure 1.

The apparatus shall include an oil bath of not less than 400 cm³ capacity with a device for heating the oil and with a temperature control in the range from 20 °C to 120 °C to within ± 2 °C.

6.2 Materials

The flushing fluids shall be petrol and acetone.

The dimensions of the stationary specimen are shown on figure 2. Surface roughnesses R_a and R_z should correspond as far as possible to the surface roughness of the tested bearing components.

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7 Preparation for the tests

7.1 Give a suitable finish to the stationary specimen surface using a special tool to give a surface fit to the rotating specimen in the apparatus over not less than 90 % of the nominal contact area. Control the fit by checking the mark of the contact.

 NOTE — The surfaces may be worked with emery paper to give a proper fit.

7.2 Remove burrs from the friction surface of the rotating and the stationary specimens. Blunt the sharp edges and chamfer the ends. Flush the surfaces successively with petrol and acetone. Measure the surface roughness parameters R_a and R_z of the specimens.

7.3 Place the rotating and the stationary specimens in the testing apparatus. Supply oil to the apparatus. There should be enough oil to submerge both specimens completely. Rotate the rotating specimen.

 $\mathsf{NOTE}-\mathsf{For}$ testing, it is preferable to use the oil that will be used in practice for the actual combination of bearing materials.

7.4 Apply a load of 10 MPa. Continue running-in until the coefficient of friction (friction forces moment) is stabilized but for at least 120 min.

8 Test procedure

with 7.4.

temperatures in the oil **8.1** Heat the oil in the oil bath up to 30 °C \pm 2 °C and at the <u>9993:1989</u> same time begin the running-in of the specificial in accordance dards/sist/03d54d72-fbe8-4a0

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8.2 Start the loading device to ensure a continuous increase of load at rate C_1 . Constantly record the value of the coefficient of friction.

 $\mbox{NOTE}-\mbox{A}$ constant recording of the friction force and the load instead of the coefficient of friction is allowed.

8.3 Continue testing in accordance with 7.1 and 7.2 until the coefficient of friction rises sharply or scratches, hollows or build-ups not more than 5 μ m deep (or high) appear on the surface of the specimens.

8.4 Replace the specimens and prepare the new specimens in accordance with clause 7 and 8.1. Start the loading device to ensure a continuous increase of load at rate C_2 . Constantly record the value of the coefficient of friction. Continue the test in accordance with 8.3.

8.5 Repeat the operations given in 8.1, 8.2, 8.3 and 8.4 at temperatures in the oil bath of 60 °C and 90 °C.

8.6 Repeat the operations given in 8.1 to 8.5 not less than three times using new specimens.

8.7 Replace the specimens and prepare the new specimens for the tests given in 8.6. Heat the oil in the oil bath up to the

temperatures given in 8.5. Start the loading device to ensure a continuous increase of load at rate C_2 . Constantly record the values of surface temperature and of the coefficient of friction. Continue the tests until seizure occurs.

9 Independent test variables

In the process of comparative testing, the following independent test variables should be kept constant:

 method of machining and finishing of the harder (usually steel) rotating specimen material;

- initial surface roughness (R_{ar} , $R_{a, max}$) of the rotating specimen.

10 Expression of results

10.1 From the results of each of the tests of 8.2, 8.4 and 8.6 on not less than three specimens, calculate the arithmetic mean of the pressures corresponding to the minimum coefficient of friction at each loading velocity and at temperatures T_1 , T_2 and T_3 .

10.2 From the value found in 10.1, determine the

hydrodynamic index of running-in ability, *R*_H, for each of the oil temperatures in the oil bath. <u>R 9993:1989</u> adards/sist/03d54d72-fbe8-4a01-89f0-

10.3 Linearize the graph of the hydrodynamic index of running-in ability, $R_{\rm H}$, as a function of the oil temperature in the oil bath using the equation

$$R_{\rm H} = R_{\rm HT} + a T_{\rm oil}$$

NOTE — The $R_{\rm HT}$ and *a* parameters are used as running-in indices. The higher the values of $R_{\rm HT}$ and the lower is the value of *a*, the higher is the running-in ability of the material.

11 Description of materials, oil and test conditions

Unless agreed otherwise, for description of materials, oil, test conditions and test results, the following data shall be supplied.

11.1 Bearing material

Туре

Chemical composition, in percent

Heat treatment

Parameters of surface roughness, R_a and R_z

Hardness

Method of applying the surface layer

11.2 Material of mating component

Туре

Chemical composition, in percent

Heat treatment

Parameters of surface roughness, R_a and R_z

Hardness

Method of applying the surface layer

11.3 Oil

Type (including information on viscosity and, if possible, on additives)

11.4 Cover gas

Туре

Relative humidity, in percent

11.5 Operating variables

Loading rate, C, in pascals per minute Sliding velocity, u, in metres per second Temperature of the oil bath, T_{oil}

11.6 Test results

The following parameters shall be supplied and inserted in table 2:

- a) Hydrodynamic index of running-in ability, R_H
- b) Temperature ratio constants, $R_{\rm HT}$ and a

Table 2

	Bearing material	Material of the mating part
Transferred material		
Reaction layer		
Scratches		
— none — several (1 to 3) — many		

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Annex A (informative)

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