# INTERNATIONAL STANDARD



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## Earth-moving machinery — Hydraulic excavators — Lift capacity

## iTeh Engins de terrassement P. Pelles hydrauliques – Capacité de levage (standards.iteh.ai)

<u>ISO 10567:1992</u> https://standards.iteh.ai/catalog/standards/sist/57fb2b14-e112-4f5a-a2d0ee012aa36c97/iso-10567-1992



#### Foreword

ISO (the International Organization for Standardization) is a worldwide federation of national standards bodies (ISO member bodies). The work of preparing International Standards is normally carried out through ISO technical committees. Each member body interested in a subject for which a technical committee has been established has the right to be represented on that committee. International organizations, governmental and non-governmental, in liaison with ISO, also take part in the work. ISO collaborates closely with the International Electrotechnical Commission (IEC) on all matters of electrotechnical standardization.

Draft International Standards adopted by the technical committees are circulated to the member bodies for voting. Publication as an International Standard requires approval by at least 75% of the member IEW bodies casting a vote.

#### (standards.iteh.ai)

International Standard ISO 10567 was prepared by Technical Committee ISO/TC 127, Earth-moving machinery, Sub-Committee SC 2, Safety requirements and human factors. https://standards.iteh.ai/catalog/standards/sist/57fb2b14-e112-4f5a-a2d0-

Annex A forms an integral part of this International Standard. Annex B is for information only.

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International Organization for Standardization

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### Earth-moving machinery — Hydraulic excavators — Lift capacity

#### 1 Scope

This International Standard provides a uniform method to calculate hydraulic excavator lift capacity and a test procedure for verifying the calculations. It covers both hydraulic lift capacity limits and machine tipping limits and establishes the rated lift capacity for hydraulic excavators as defined in ISO 7135.

#### axis of rotation to the vertical hoist line or tackle. iTeh STANDARD See figure 1.

#### 2 Normative references

(standards.it35h balance point: Moment acting to overturn the machine with a specific load and point lift radius The following standards contain provisions which,

which is equal to the moment of the machine availthrough reference in this text, constitute provisions 67:199 able to resist overturning. of this International StandardarAtrtheetimeaofopublitards/sist/ 'tb2b14-e112 cation, the editions indicated were valid, All stanfiso-1053.619 tipping load: Static load at the balance point.

figure 1.

dards are subject to revision, and parties to agreements based on this International Standard are encouraged to investigate the possibility of applying the most recent editions of the standards indicated below. Members of IEC and ISO maintain registers of currently valid International Standards.

ISO 6015:1989, Earth-moving machinery – Hydraulic excavators — Methods of measuring tool forces.

ISO 7135:-1), Earth-moving machinery – Hydraulic excavators - Terminology and commercial specifications.

#### **3 Definitions**

For the purposes of this International Standard, the following definitions apply.

3.1 load: External force, including the weight of the attached equipment, applied at the lift point.

3.2 lift point: Location on the bucket or the bucket mounting bracket, specified by the manufacturer, to which a load may be attached, or the centre-line of the bucket pivot mounting pin on the arm. For bucket

3.7 rated tipping load: 75 % of the static tipping load.

or bucket mounting bracket load attachment, the

3.3 lift point height: Vertical distance from the

ground reference plane to the lift point. See

3.4 lift point radius: Horizontal distance from the

bucket cylinder is fully extended. See figure 1.

#### 3.8 Hydraulic pressures

3.8.1 working circuit pressure: That nominal pressure applied to the specific circuit by the pump(s).

3.8.2 holding circuit pressure: Maximum static pressure in a specific circuit, limited by a relief valve at a flow no greater than 10 % of rated circuit flow.

3.9 hydraulic lift capacity: Load that can be lifted from the lift point by the boom cylinders with the bucket in rated lift bucket position and the excavator physically restrained from tipping.

3.9.1 boom hydraulic lift capacity: Load that can be lifted by applying working circuit pressure to the boom cylinder(s) without exceeding holding circuit pressure in any other circuit.

3.9.2 arm hydraulic lift capacity: Load that can be lifted by applying working circuit pressure to the arm

<sup>1)</sup> To be published.

cylinder(s) without exceeding the working circuit pressure in the boom cylinders or the holding circuit pressure in any other circuit.

3.10 rated hydraulic lift capacity: 87 % of the smaller of boom or arm hydraulic lift capacity at specific lift point positions.

3.11 rated lift capacity: Smaller of either rated tipping load (3.7) or rated hydraulic lift capacity (3.10).

#### Calculations 4

#### **Tipping load calculations** 4.1

A series of calculations at various lift radii is made to determine the load required to achieve the balance point as defined in 3.5. Sufficient lift radii shall be considered to develop the rated lift capacity chart (see annex A). Lift point positions shall be included above and below the ground reference plane, over the ends and the sides of the machine, and with the machine in the configuration that results in the lowest moment available to resist overturning.

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#### 4.1.1 Machine configuration for calculations

standai oscillating axle shall be a line through the axle pivot 4.1.1.1 Because of the large number of attachment point and one other rigid support point (see options and machine variations available, the figure 3). manufacturer shall publish revised load rating s/sist/57fb2b14-e112-4f5a-a2d0-4.1.3.4.1.0lf ratings are based upon a blocked or noncharts if these variations would decrease the maaa36c97/iso chine rated lift capacity by more than 5 %.<sup>ee</sup>

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4.1.1.2 Lift capacities shall be calculated with the machine on a firm level supporting surface.

#### 4.1.2 Calculations for balance point for end tipping line

4.1.2.1 The tipping line to be used for balance point calculations over the front/rear of machines with track-type undercarriage shall be a line connecting the centre-line of support idlers or sprockets (see figure 2). The linkage shall be positioned over the front/rear in the least stable position for these calculations.

**4.1.2.2** The tipping line to be used for calculations over the front/rear of machines with rubber-tyred undercarriage shall be the axle centre-line, the bogie axle centre-line, or a line connecting the outrigger pads as shown in figure 3.

**4.1.2.3** The tipping line for pivoted outrigger pads shall be a line at the ground reference plane, connecting the point on the pads directly below the centre-line of the pivot. For rigid outrigger pads, the tipping line shall be a line connecting the centroid of the contact area between the pads and the ground reference plane.

4.1.2.4 A backfill blade, properly attached to the machine and capable of supporting the machine as an outrigger, may be considered an outrigger.

4.1.2.5 For machines equipped with outriggers, calculations shall be made both without the outriggers applied and with the outriggers applied in their most favourable position.

#### 4.1.3 Calculations for balance point for side tipping line

4.1.3.1 The tipping line to be used for side tipping balance point calculations on machines with tracktype undercarriages shall be defined by the pivot points between support rollers and track elements (such as links or guides) as shown in figure 4.

**4.1.3.2** The tipping line to be used for calculations for the balance point of machines with rubber-tyred undercarriage with blocked or non-oscillating axles shall be a line connecting the centre of contact of the tyres (midpoint between dual tyres) on the same side of the machine, at the ground reference plane (see figures 3 and 4).

4.1.3.3 The tipping line for an excavator with an

oscillating axle, this condition shall be clearly defined on the load rating charts and diagrams.

4.1.3.5 When outriggers are used, the position of the tipping line shall be as specified in 4.1.2.3.

#### 4.2 Hydraulic lift capacity calculations

A series of calculations at various lift points is made to determine the load that can be lifted with the force generated by the boom or the arm hydraulic lift capacity (as defined in 3.9.1 and 3.9.2). Sufficient excavator linkage position calculations shall be made, including lift points above and below the ground reference plane, to develop the rated lift capacity chart shown in annex A.

#### 5 Verification testing

#### 5.1 Test site

#### 5.1.1 Dead weight test site (immovable weight)

A dead weight test site shall consist of a firm and level horizontal surface arranged so that a load cell can be connected between the lift point and the dead weight. The dead weight may be either a horizontal rail with a movable attachment device or a fixed point dead weight with the excavator moving to obtain the various lift points (see figures 5 and 6).

#### 5.1.2 Live weight test site (movable weight)

A live weight test site shall consist of a firm and level horizontal surface arranged so that a weight attached to the lift point can be moved without obstructing the limit of the excavator's tipping load or hydraulic capacity. See figure 7 for a typical test arrangement. The live weight should be kept within 0,5 m of the surface from which it was raised to minimize the possibility of the machine overturning.

#### 5.2 Test equipment

Instrumentation accuracy shall be as defined in ISO 6015.

**5.2.1 A load cell** of sufficient capacity (if a dead weight test site is used).

5.2.2 Weights of known mass (if a live weight test R site is used).

## 5.2.3 A means of measuring the lift point position

relative to the axis of rotation of the excavator

https://standards.iteh.ai/catalog/standards/si 5.2.4 A means of measuring perpendicularity6.b97iso-103 tween the load line and the ground reference plane when using the dead weight test site.

**5.2.5** A means to monitor the pressure in all hydraulic circuits which will be under pressure during the actual lift capacity verification tests.

#### 5.3 Test procedure

**5.3.1** The excavator shall be thoroughly cleaned and in normal working condition with fuel tanks filled to capacity and all other fluids at their prescribed levels and at normal operating temperature.

**5.3.2** The excavator shall be fitted with working equipment and counterweight as specified by the manufacturer for the calculated lift capacity chart being verified.

**5.3.3** Tyres on rubber-tyred undercarriage machines shall be inflated to the manufacturer's recommended values.

**5.3.4** Track tension on machines with track-type undercarriage shall be adjusted to the manufacturer's recommendations.

**5.3.5** The hydraulic pressure shall be checked. This will include the working circuit pressure and the holding circuit pressure to ensure that the system is set at the manufacturer's recommended nominal published value.

**5.3.6** The test personnel shall conduct the tests in a safe manner and follow the operating instructions, operator's manuals, safety rules, etc., furnished by the manufacturer of the excavator and of the test equipment.

**5.3.7** A means shall be provided for preventing the excavator from overturning during the test procedure.

#### 5.4 Tests

**5.4.1** Tipping load measurements shall be carried out at lift specific radii to determine the force that achieves the balance point defined in 3.5.

Tests for machines with outriggers shall be conducted both without the outriggers applied and with the outriggers applied in their most favourable position.

5.4.2 Hydraulic lift capacity measurements shall be carried out at specific lift points to verify hydraulic lards/sistift capacity calculations. These measurements are iso-105 made without exceeding the working circuit pressure in the boom cylinders or the holding circuit pressure in any other circuit.

**5.4.3** The number of verifying points obtained shall include at least the following four points:

- a) tipping over the end and side: position the linkage over the end and the side to obtain tipping load;
- b) hydraulic limited lift capacity above and below the ground reference plane.

#### 5.5 Test results

Measured lift forces, lift point heights and lift point radii for tipping loads and hydraulic lift capacities shall be recorded.

#### 6 Validation of calculated values

The measured values should be within 95 % of the calculated values. If not, the lift capacity chart shall be adjusted based on the correction factor determined by the measured values.

#### 7 Rated lift capacity chart

7.1 The format for the rated lift capacity chart is presented in annex A.

**7.2** The rated lift capacity chart shall show the lift capacity (see 3.11) at specific lift point radii. The chart shall note if the values are limited by hydraulic lift capacity.

**7.3** Rated lift capacity values shall be tabulated for intersections of the lift point with a 0,5 m, 1 m, or 2 m vertically and horizontally spaced grid placed over the excavator's working range with bucket attitude maintained in the rated lift bucket position. The maximum and minimum lift radii shall also be included. The origin of the grid shall be at the intersection of the ground reference plane and axis of rotation.

**7.4** A rated lift capacity chart shall be mounted inside the excavator cab and be legible from the control position.

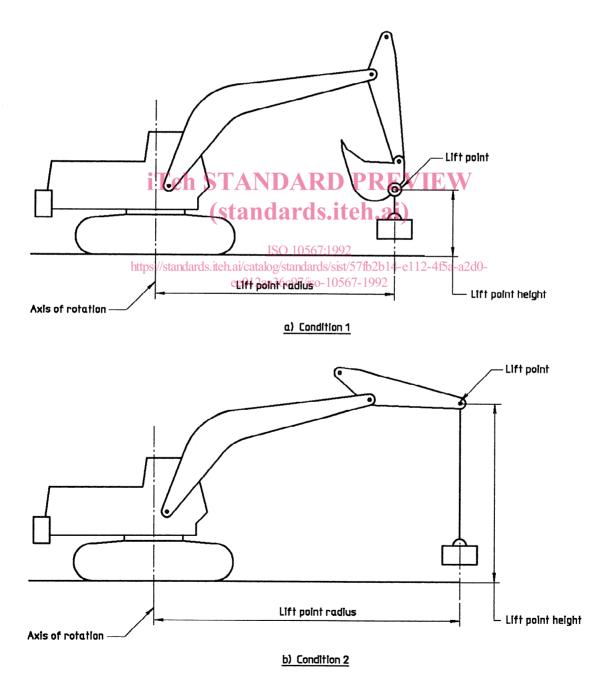
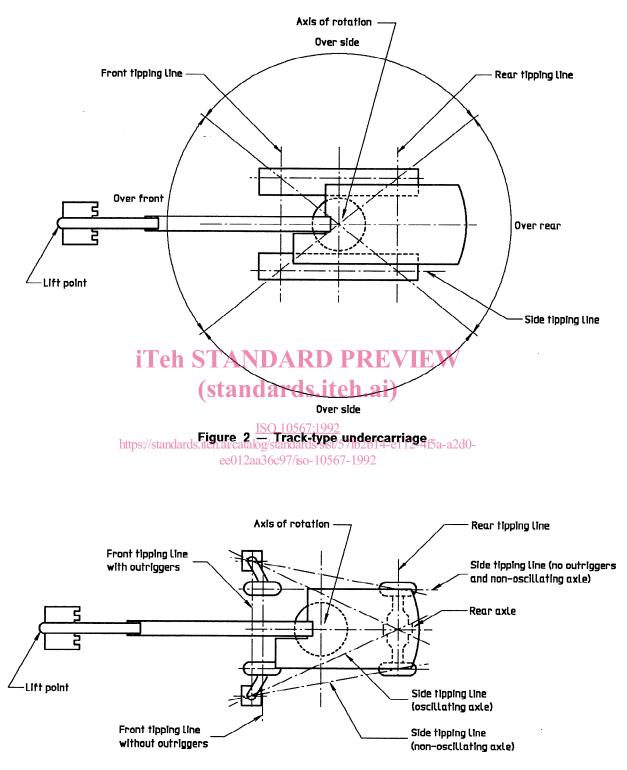


Figure 1 — Lift point



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Figure 3 — Rubber-tyred undercarriage

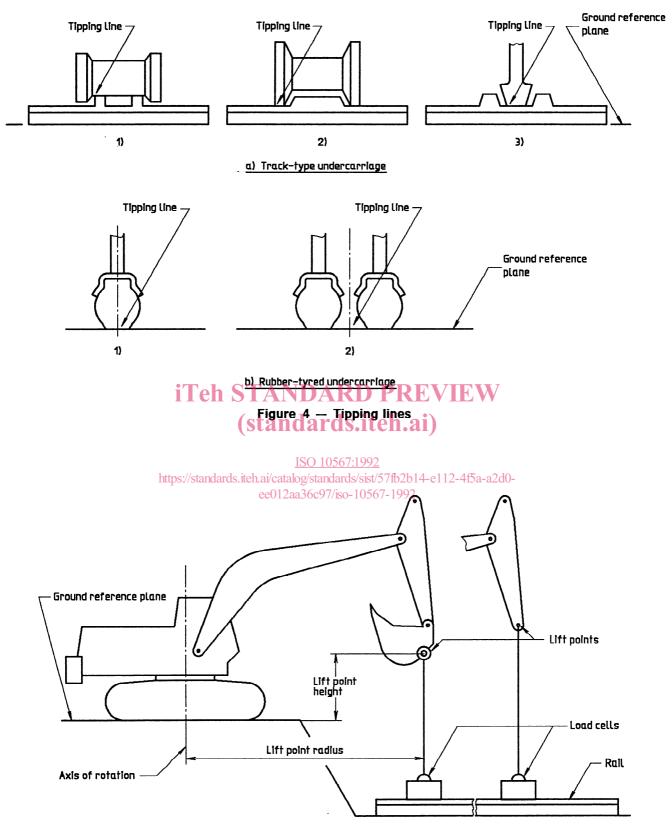


Figure 5 - Self-aligning dead weight

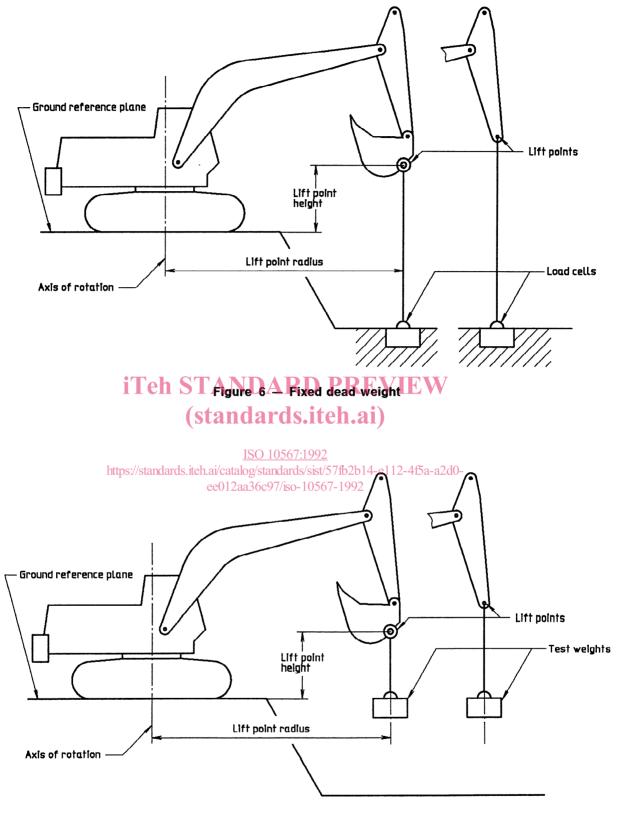


Figure 7 — Live weight