

SLOVENSKI STANDARD

SIST EN 60534-8-3:2011

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Regulacijski ventili za industrijske procese - 8-3. del: Šum - Metoda napovedovanja aerodinamičnega šuma regulacijskega ventila (IEC 60534-8-3:2010)

Industrial-process control valves - Part 8-3: Noise considerations - Control valve aerodynamic noise prediction method (IEC 60534-8-3:2010)

Vannes de régulation des processus industriels - Partie 8-3: Considérations sur le bruit - Méthode de prédiction du bruit aérodynamique des vannes de régulation (IEC 60534-8-3:2010)

[SIST EN 60534-8-3:2011](https://standards.iteh.ai/SIST/60534-8-3-2011)

Vannes de régulation des processus industriels - Partie 8-3: Considérations sur le bruit - Méthode de prévision du bruit aérodynamique des vannes de régulation (CEI 60534-8-3:2010)

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23.060.40	Tlačni regulatorji	Pressure regulators
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EUROPEAN STANDARD
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EN 60534-8-3

January 2011

ICS 17.140.20; 23.060.40; 25.040.40

Supersedes EN 60534-8-3:2000

English version

**Industrial-process control valves -
Part 8-3: Noise considerations -
Control valve aerodynamic noise prediction method
(IEC 60534-8-3:2010)**

Vannes de régulation des processus
industriels -
Partie 8-3: Considérations sur le bruit -
Méthode de prédiction du bruit
aérodynamique des vannes de régulation
(CEI 60534-8-3:2010)

Stellventile für die Prozessregelung -
Teil 8-3: Geräuschbetrachtungen -
Berechnungsverfahren zur Vorhersage
der aerodynamischen Geräusche von
Stellventilen
(IEC 60534-8-3:2010)

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This European Standard was approved by CENELEC on 2011-01-01. CENELEC members are bound to comply with the CEN/CENELEC Internal Regulations which stipulate the conditions for giving this European Standard the status of a national standard without any alteration.

Up-to-date lists and bibliographical references concerning such national standards may be obtained on application to the Central Secretariat or to any CENELEC member.

This European Standard exists in three official versions (English, French, German). A version in any other language made by translation under the responsibility of a CENELEC member into its own language and notified to the Central Secretariat has the same status as the official versions.

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CENELEC

European Committee for Electrotechnical Standardization
Comité Européen de Normalisation Electrotechnique
Europäisches Komitee für Elektrotechnische Normung

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Foreword

The text of document 65B/765/FDIS, future edition 3 of IEC 60534-8-3, prepared by IEC/SC 65B, Devices & process analysis, of IEC TC 65, Industrial-process measurement, control and automation, was submitted to the IEC-CENELEC parallel vote and was approved by CENELEC as EN 60534-8-3 on 2011-01-01.

This European Standard supersedes EN 60534-8-3:2000.

The significant technical changes with respect to EN 60534-8-3:2000 are as follows:

- predicting noise as a function of frequency;
- using laboratory data to determine the acoustical efficiency factor.

Attention is drawn to the possibility that some of the elements of this document may be the subject of patent rights. CEN and CENELEC shall not be held responsible for identifying any or all such patent rights.

The following dates were fixed:

- | | | |
|--|-------|------------|
| <ul style="list-style-type: none"> – latest date by which the EN has to be implemented at national level by publication of an identical national standard or by endorsement | (dop) | 2011-10-01 |
| <ul style="list-style-type: none"> – latest date by which the national standards conflicting with the EN have to be withdrawn | (dow) | 2014-01-01 |

Annex ZA has been added by CENELEC.

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Endorsement notice

The text of the International Standard IEC 60534-8-3:2010 was approved by CENELEC as a European Standard without any modification.

In the official version, for Bibliography, the following notes have to be added for the standards indicated:

- | | | |
|-------------------|------|-----------------------------|
| [1] IEC 60534-2-1 | NOTE | Harmonized as EN 60534-2-1. |
| [2] IEC 60534-8-1 | NOTE | Harmonized as EN 60534-8-1. |

Annex ZA (normative)

Normative references to international publications with their corresponding European publications

The following referenced documents are indispensable for the application of this document. For dated references, only the edition cited applies. For undated references, the latest edition of the referenced document (including any amendments) applies.

NOTE When an international publication has been modified by common modifications, indicated by (mod), the relevant EN/HD applies.

<u>Publication</u>	<u>Year</u>	<u>Title</u>	<u>EN/HD</u>	<u>Year</u>
IEC 60534	Series	Industrial-process control valves	EN 60534	Series
IEC 60534-1	-	Industrial-process control valves - Part 1: Control valve terminology and general considerations	EN 60534-1	-

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INTERNATIONAL STANDARD

NORME INTERNATIONALE

**Industrial-process control valves –
Part 8-3: Noise considerations – Control valve aerodynamic noise prediction
method**

**Vannes de régulation des processus industriels –
Partie 8-3: Considérations sur le bruit – Méthode de prédiction du bruit
aérodynamique des vannes de régulation**

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INTERNATIONAL ELECTROTECHNICAL COMMISSION

INDUSTRIAL-PROCESS CONTROL VALVES –**Part 8-3: Noise considerations –
Control valve aerodynamic noise prediction method**

FOREWORD

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International Standard IEC 60534-8-3 has been prepared by subcommittee 65B: Measurements and control devices, of IEC technical committee 65: Industrial-process measurement, control and automation.

This third edition cancels and replaces the second edition published in 2000. This edition constitutes a technical revision.

The significant technical changes with respect to the previous edition are as follows:

- predicting noise as a function of frequency;
- using laboratory data to determine the acoustical efficiency factor.

The text of this standard is based on the following documents:

FDIS	Report on voting
65B/765/FDIS	65B/780/RVD

Full information on the voting for the approval of this standard can be found in the report on voting indicated in the above table.

This publication has been drafted in accordance with the ISO/IEC Directives, Part 2.

A list of all the parts of the IEC 60534 series, under the general title *Industrial-process control valves* can be found on the IEC website..

The committee has decided that the contents of this publication will remain unchanged until the stability date indicated on the IEC web site under "<http://webstore.iec.ch>" in the data related to the specific publication. At this date, the publication will be

- reconfirmed,
- withdrawn,
- replaced by a revised edition, or
- amended.

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INTRODUCTION

The mechanical stream power as well as acoustical efficiency factors are calculated for various flow regimes. These acoustical efficiency factors give the proportion of the mechanical stream power which is converted into internal sound power.

This method also provides for the calculation of the internal sound pressure and the peak frequency for this sound pressure, which is of special importance in the calculation of the pipe transmission loss.

At present, a common requirement by valve users is the knowledge of the sound pressure level outside the pipe, typically 1 m downstream of the valve or expander and 1 m from the pipe wall. This standard offers a method to establish this value.

The equations in this standard make use of the valve sizing factors as used in IEC 60534-1 and IEC 60534-2-1.

In the usual control valve, little noise travels through the wall of the valve. The noise of interest is only that which travels downstream of the valve and inside of the pipe and then escapes through the wall of the pipe to be measured typically at 1 m downstream of the valve body and 1 m away from the outer pipe wall.

Secondary noise sources may be created where the gas exits the valve outlet at higher Mach numbers. This method allows for the estimation of these additional sound levels which can then be added logarithmically to the sound levels created within the valve.

Although this prediction method cannot guarantee actual results in the field, it yields calculated predictions within 5 dB(A) for the majority of noise data from tests under laboratory conditions (see IEC 60534-8-1). The current edition has increased the level of confidence of the calculation. In some cases the results of the previous editions were more conservative.

The bulk of the test data used to validate the method was generated using air at moderate pressures and temperatures. However, it is believed that the method is generally applicable to other gases and vapours and at higher pressures. Uncertainties become greater as the fluid behaves less perfectly for extreme temperatures and for downstream pressures far different from atmospheric, or near the critical point. The equations include terms which account for fluid density and the ratio of specific heat.

NOTE Laboratory air tests conducted with up to 1 830 kPa (18,3 bar) upstream pressure and up to 1 600 kPa (16,0 bar) downstream pressure and steam tests up to 225 °C showed good agreement with the calculated values.

A rigorous analysis of the transmission loss equations is beyond the scope of this standard. The method considers the interaction between the sound waves existing in the pipe fluid and the first coincidence frequency in the pipe wall. In addition, the wide tolerances in pipe wall thickness allowed in commercial pipe severely limit the value of the very complicated mathematical approach required for a rigorous analysis. Therefore, a simplified method is used.

Examples of calculations are given in Annex A.

This method is based on the IEC standards listed in Clause 2 and the references given in the Bibliography.

INDUSTRIAL-PROCESS CONTROL VALVES –

Part 8-3: Noise considerations – Control valve aerodynamic noise prediction method

1 Scope

This part of IEC 60534 establishes a theoretical method to predict the external sound-pressure level generated in a control valve and within adjacent pipe expanders by the flow of compressible fluids.

This method considers only single-phase dry gases and vapours and is based on the perfect gas laws.

This standard addresses only the noise generated by aerodynamic processes in valves and in the connected piping. It does not consider any noise generated by reflections from external surfaces or internally by pipe fittings, mechanical vibrations, unstable flow patterns and other unpredictable behaviour.

It is assumed that the downstream piping is straight for a length of at least 2 m from the point where the noise measurement is made.

This method is valid only for steel and steel alloy pipes (see Equations (21) and (23) in 5.5).

The method is applicable to the following single-stage valves: globe (straight pattern and angle pattern), butterfly, rotary plug (eccentric, spherical), ball, and valves with cage trims. Specifically excluded are the full bore ball valves where the product $F_p C$ exceeds 50 % of the rated flow coefficient.

For limitations on special low noise trims not covered by this standard, see Clause 8. When the Mach number in the valve outlet exceeds 0,3 for standard trim or 0,2 for low noise trim, the procedure in Clause 7 is used

The Mach number limits in this standard are as follows:

Mach number location	Mach number limit		
	Clause 5 Standard trim	Clause 6 Noise-reducing trim	Clause 7 High Mach number applications
Freely expanded jet M_j	No limit	No limit	No limit
Valve outlet M_o	0,3	0,2	1,0
Downstream reducer inlet M_r	Not applicable	Not applicable	1,0
Downstream pipe M_2	0,3	0,2	0,8

2 Normative references

The following referenced documents are indispensable for the application of this document. For dated references, only the edition cited applies. For undated references, the latest edition of the referenced document (including any amendments) applies.

IEC 60534 (all parts), *Industrial-process control valves*

IEC 60534-1, *Industrial-process control valves – Part 1: Control valve terminology and general considerations*

3 Terms and definitions

For the purposes of this document, all of the terms and definitions given in the IEC 60534 series and the following apply:

3.1

acoustical efficiency

η

ratio of the stream power converted into sound power propagating downstream to the stream power of the mass flow

3.2

external coincidence frequency

f_g

frequency at which the external acoustic wavespeed is equal to the bending wavespeed in a plate of equal thickness to the pipe wall

3.3

internal coincidence frequency

f_o

lowest frequency at which the internal acoustic and structural axial wave numbers are equal for a given circumferential mode, thus resulting in the minimum transmission loss

3.4

fluted vane butterfly valve

butterfly valve which has flutes (grooves) on the face(s) of the disk. These flutes are intended to shape the flow stream without altering the seating line or seating surface

3.5

independent flow passage

flow passage where the exiting flow is not affected by the exiting flow from adjacent flow passages

3.6

peak frequency

f_p

frequency at which the internal sound pressure is maximum

3.7

valve style modifier

F_d

ratio of the hydraulic diameter of a single flow passage to the diameter of a circular orifice, the area of which is equivalent to the sum of areas of all identical flow passages at a given travel

4 Symbols

Symbol	Description	Unit
A	Area of a single flow passage	m^2
A_η	Valve correction factor for acoustical efficiency (see Table 4)	Dimensionless
A_n	Total flow area of last stage of multistage trim with n stages at given travel	m^2
C	Flow coefficient (K_v and C_v)	Various (see IEC 60534-1)
c_a	External speed of sound (dry air at standard conditions = 343 m/s)	m/s
C_n	Flow coefficient for last stage of multistage trim with n stages	Various (see IEC 60534-1)
c_s	Speed of sound of the pipe (for steel = 5 000 m/s)	m/s
c_{vc}	Speed of sound in the <i>vena contracta</i> at subsonic flow conditions	m/s
c_{vcc}	Speed of sound in the <i>vena contracta</i> at critical flow conditions	m/s
c_2	Speed of sound at downstream conditions	m/s
D	Valve outlet diameter	m
d	Diameter of a flow passage (for other than circular, use d_H)	m
d_H	Hydraulic diameter of a single flow passage	m
d_i	Smaller of valve outlet or expander inlet internal diameters	m
D_i	Internal downstream pipe diameter	m
D_j	Jet diameter at the <i>vena contracta</i>	m
d_o	Diameter of a circular orifice, the area of which equals the sum of areas of all flow passages at a given travel	m
F_d	Valve style modifier	Dimensionless
F_L	Liquid pressure recovery factor of a valve without attached fittings (see Note 4)	Dimensionless
F_{Ln}	Liquid pressure recovery factor of last stage of low noise trim	Dimensionless
F_{LP}	Combined liquid pressure recovery factor and piping geometry factor of a control valve with attached fittings (see Note 4)	Dimensionless
F_p	Piping geometry factor	Dimensionless
f_g	External coincidence frequency	Hz
f_o	Internal coincidence pipe frequency	Hz
f_p	Generated peak frequency	Hz
f_{pR}	Generated peak frequency in valve outlet or reduced diameter of expander	Hz
f_r	Ring frequency	Hz
f_s	Structural loss factor reference frequency = 1 Hz	Hz