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**Cylindrical gears for general and heavy  
engineering — Standard basic rack tooth  
profile**

*Engrenages cylindriques de mécanique générale et de grosse  
mécanique — Tracé de référence*

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ISO 53:1998

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## Foreword

ISO (the International Organization for Standardization) is a worldwide federation of national standards bodies (ISO member bodies). The work of preparing International Standards is normally carried out through ISO technical committees. Each member body interested in a subject for which a technical committee has been established has the right to be represented on that committee. International organizations, governmental and non-governmental, in liaison with ISO, also take part in the work. ISO collaborates closely with the International Electrotechnical Commission (IEC) on all matters of electrotechnical standardization.

Draft International Standards adopted by the technical committees are circulated to the member bodies for voting. Publication as an International Standard requires approval by at least 75 % of the member bodies casting a vote.

International Standard ISO 53 was prepared by Technical Committee ISO/TC 60, *Gears*.

This second edition cancels and replaces the first edition (ISO 53:1974), which has been technically revised.

Annex A of this International Standard is for information only.

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Printed in Switzerland

# Cylindrical gears for general and heavy engineering — Standard basic rack tooth profile

## Scope

This International Standard specifies the characteristics of the standard basic rack tooth profile for cylindrical involute gears (external or internal) for general and heavy engineering.

This International Standard applies to the standardized modules specified in ISO 54.

The defined profile does not take into account the possible cut-off of the height of internal teeth. This height is to be calculated for each case.

The standard basic rack tooth profile defined in this International Standard constitutes a geometrical reference for a system of involute gears in order to fix the sizes of their teeth. It does not constitute a definition of a cutter, but a cutter may be defined from this standard basic rack tooth profile in order to realize a conforming profile.

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## 2 Normative references

The following standards contain provisions which, through reference in this text, constitute provisions of this International Standard. At the time of publication, the editions indicated were valid. All standards are subject to revision, and parties to agreements based on this International Standard are encouraged to investigate the possibility of applying the most recent editions of the standards indicated below. Members of IEC and ISO maintain registers of currently valid International Standards.

ISO 54:1996, *Cylindrical gears for general and for heavy engineering — Modules*.

ISO 1122-1:1998, *Vocabulary of gear terms — Part 1: Definitions related to geometry*.

## 3 Terms and definitions

For the purposes of this International Standard, the terms and definitions given in ISO 1122-1 and the following apply.

### 3.1

#### **standard basic rack tooth profile**

tooth profile normal section through the teeth of a basic rack which corresponds to an external gear with number of teeth  $z = \infty$  and diameter  $d = \infty$

Figure 1.

NOTE — The tooth of the standard basic rack tooth profile is bounded by the tip line at the top and by the parallel root line at the bottom. The fillet between the straight part of the profile and the root line is a circular arc with a radius equal to  $\rho_{\text{IP}}$ .

### 3.2 mating standard rack tooth profile

rack tooth profile symmetrical to the standard basic rack tooth profile about the datum line P-P and displaced by half a pitch relative to it

Figure 1.

## 4 Symbols and units

See table 1.

Table 1 — Symbols and units

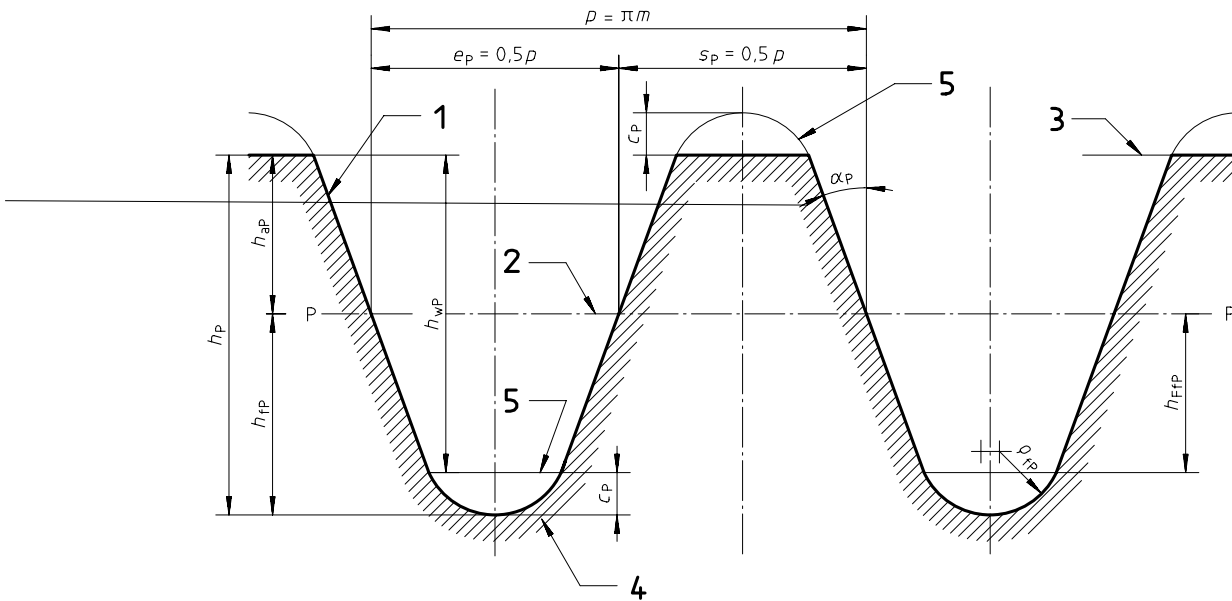
Symbol	Description	Unit
$c_p$	Bottom clearance between standard basic rack tooth and mating standard basic rack tooth	mm
$e_p$	Spacewidth of standard basic rack	mm
$h_{aP}$	Addendum of standard basic rack tooth	mm
$h_{iP}$	Dedendum of standard basic rack tooth	mm
$h_{FP}$	Straight portion of the standard basic rack tooth dedendum	mm
$h_p$	Tooth depth of standard basic rack	mm
$h_{wP}$	Common depth of standard basic rack and mating standard basic rack tooth	mm
$m$	Module <a href="https://standards.iteh.ai/catalog/standards/sist/032d8eaa-c5ba-4f3c-96d6-592df1cd067d/iso-53-1998">https://standards.iteh.ai/catalog/standards/sist/032d8eaa-c5ba-4f3c-96d6-592df1cd067d/iso-53-1998</a>	mm
$p$	Pitch	mm
$s_p$	Tooth thickness of standard basic rack tooth	mm
$U_{FP}$	Size of undercut	mm
$\alpha_{FP}$	Angle of undercut	degrees
$\alpha_p$	Pressure angle	degrees
$\rho_{iP}$	Fillet radius of the basic rack	mm

## 5 Standard basic rack tooth profile

5.1 The characteristics of the standard basic rack tooth profile are given in figure 1 and table 2. Additional basic rack recommendations and applications are given in annex A.

5.2 The standard basic rack tooth profile with module  $m$  has a pitch  $p = \pi m$ .

5.3 The flanks of the standard basic rack tooth profile are straight for the section  $h_{aP}$  plus  $h_{FP}$ .



**Key**

- 1 Standard basic rack profile
- 2 Datum line
- 3 Tip line
- 4 Root line
- 5 Mating standard basic rack tooth profile

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**Figure 1 — Standard basic rack tooth profile and mating standard basic rack tooth profile**

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**Table 2 — Standard basic rack proportions**

Item	Standard basic rack value
$\alpha_P$	20°
$h_{aP}$	1 <i>m</i>
$c_P$	0,25 <i>m</i>
$h_{TP}$	1,25 <i>m</i>
$\rho_{FP}$	0,38 <i>m</i>

**5.4** On the line P-P the tooth thickness is equal to the space-width, i.e. half the pitch.

$$s_P = e_P = \frac{p}{2} = \frac{\pi m}{2} \tag{1}$$

where

- $s_P$  is the tooth thickness of standard basic rack tooth;
- $e_P$  is the spacewidth of standard basic rack;
- $p$  is the pitch;
- $m$  is the module.

**5.5** The flanks of the standard basic rack tooth profile are inclined at the pressure angle,  $\alpha_p$ , to a line normal to the datum line P-P.

**5.6** The tip line and the root line are respectively parallel to the datum line P-P at distances of  $h_{aP}$  and  $h_{fP}$ .

**5.7** The active tooth depth,  $h_{wP}$ , of the standard basic rack tooth profile and the mating standard basic rack tooth profile is equal to  $2h_{aP}$ .

**5.8** The dimensions of the standard basic rack tooth profile use line P-P as the base datum.

**5.9** The fillet radius of the standard basic rack,  $\rho_{fP}$ , is determined by the standard clearance,  $c_p$ .

For a basic rack, where  $\alpha_p = 20^\circ$ ,  $c_p \leq 0,295 m$  and  $h_{fP} = 1 m$ :

$$\rho_{fP \max} = \frac{c_p}{1 - \sin \alpha_p} \quad (2)$$

where

$\rho_{fP \max}$  is the maximum fillet radius of the basic rack;

$c_p$  is the bottom clearance between standard basic rack tooth and mating standard basic rack tooth;

$\alpha_p$  is the pressure angle.

For a basic rack, where  $\alpha_p = 20^\circ$  and  $0,295 m < c_p \leq 0,396 m$ :

$$\rho_{fP \max} = \frac{[(\pi m) / 4 - h_{fP} \tan \alpha_p]}{\tan[(90^\circ - \alpha_p) / 2]} \quad (3)$$

where  $h_{fP}$  is the dedendum of the basic rack tooth. [ISO 53:1998](https://standards.iteh.ai/catalog/standards/sist/032d8eaa-c5ba-4f3c-96d6-592df1cd067d/iso-53-1998)

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The centre of  $\rho_{fP \max}$  is on the centre of the rack space.

NOTE — The actual root fillet, which is outside the active profile, can vary depending on influences such as the method of manufacturing, profile shift and number of teeth.

**5.10** The dimensions of the standard basic rack tooth profile  $c_p$ ,  $h_{aP}$ ,  $h_{fP}$  and  $h_{wP}$  may also be quoted as a multiple of the module  $m$ , i.e. relating to  $m = 1$ . In this case, they are identified by an asterisk, for example

$$h_{fP} = h_{fP}^* \cdot m$$

## Annex A (informative)

### Basic rack recommendations for various applications

#### A.1 Application of basic rack forms

Alternative basic rack tooth profiles may be used based on application requirements

- Standard basic rack tooth profile type A is recommended for gears transmitting high torques.
- Basic rack tooth profile types B and C are recommended for normal service. Type C may be applied for manufacturing with some standard hobs.
- Basic rack tooth profile type D is equivalent to a full radius form for the fillet. The enlarged dedendum,  $h_{fP} = 1,4 m$ , with the associated fillet radii,  $\rho_{fP} = 0,39 m$ , permits the finishing tool to work without interference, while maintaining the maximum fillet radius. This profile is recommended for high-precision gears transmitting high torques and consequently with tooth flanks finished by grinding or shaving. Care should be taken to avoid creating notches in the fillet during finishing, which would create stress concentrations.

Characteristics of the basic rack tooth profile types are given in table A.1.

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Table A.1 — Basic rack tooth profiles

Symbol	Types of basic rack tooth profile			
	A	B	C	D
$\alpha_P$	20°	20°	20°	20°
$h_{aP}$	1 <i>m</i>	1 <i>m</i>	1 <i>m</i>	1 <i>m</i>
$c_P$	0,25 <i>m</i>	0,25 <i>m</i>	0,25 <i>m</i>	0,4 <i>m</i>
$h_{fP}$	1,25 <i>m</i>	1,25 <i>m</i>	1,25 <i>m</i>	1,4 <i>m</i>
$\rho_{fP}$	0,38 <i>m</i>	0,3 <i>m</i>	0,25 <i>m</i>	0,39 <i>m</i>

#### A.2 Basic rack tooth profiles with undercut

A basic rack tooth profile with a chosen undercut,  $U_{FP}$ , and angle of undercut,  $\alpha_{FP}$ , is used for gears cut by a protuberance tool and finished by grinding or shaving, see figure A.1. The specific values of  $U_{FP}$  and  $\alpha_{FP}$  depend on influences such as the method of manufacturing, which are not specified in this International Standard.

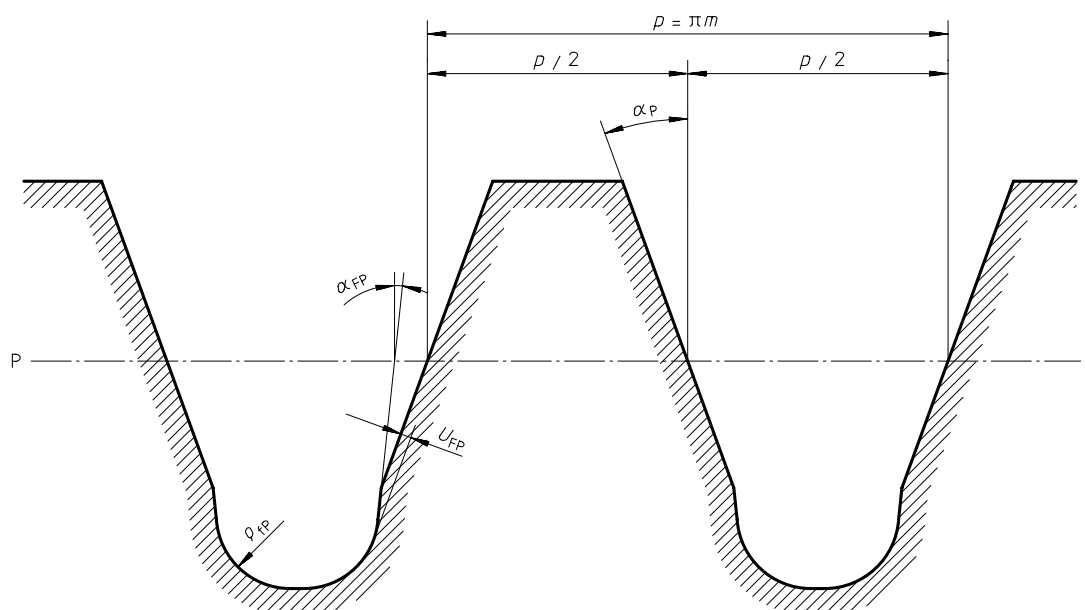


Figure A.1 — Basic rack tooth profile with a chosen undercut

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