# INTERNATIONAL STANDARD



First edition 1998-08-15

## Spherical plain bearings —

Part 4: Spherical plain bearing rod ends

Rotules lisses —

iTeh STANDARD PREVIEW (standards.iteh.ai)

<u>ISO 12240-4:1998</u> https://standards.iteh.ai/catalog/standards/sist/470dfaed-053f-4f8b-a5f6bf50a58e4741/iso-12240-4-1998



## Foreword

ISO (the International Organization for Standardization) is a worldwide federation of national standard bodies (ISO member bodies). The work of preparing International Standards is normally carried out through ISO technical committees. Each member body interested in a subject for which a technical committee has been established has the right to be represented on that committee. International organizations, governmental and non-governmental, in liaison with ISO, also take part in the work. ISO collaborates closely with the International Electrotechnical Commission (IEC) on all matters of electrotechnical standardization.

Draft International Standards adopted by the technical committees are circulated to the member bodies for voting. Publication as an International Standard requires approval by at least 75 % of the member bodies casting a vote.

International Standard ISO 12240-4 was prepared by Technical Committee ISO/TC 4, Rolling bearings, Subcommittee SC 7, Spherical plain bearings.

DD This first edition cancels and replaces ISO 6126:1987 of which it constitutes a technical revision.

ISO 12240 consists of the following parts, under the general title Spherical plain bearings:

ISO 12240-4:1998 Part 1: Radial spherical plain bearings

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- Part 2: Angular contact radial spherical plain bearings -12240-4-1998
- Part 3: Thrust spherical plain bearings
- Part 4: Spherical plain bearing rod ends

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## Spherical plain bearings —

Part 4: Spherical plain bearing rod ends

## 1 Scope

This part of ISO 12240 specifies dimensions, tolerances and radial internal clearances for various dimension series of spherical plain bearing rod ends.

The dimensions and tolerances specified in this part of ISO 12240 have been selected to permit the design and use of spherical plain bearing rod ends which incorporate radial spherical plain bearings having various sliding material combinations.

The specified tolerance values apply for finished spherical plain bearing rod ends before any coating, plating, ring splitting or fracturing. (standards.iteh.ai)

Spherical plain bearing rod ends need not conform to the designs illustrated but compliance is required as regards dimensions, tolerances and radial internal clearances specified.

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NOTE — Spherical plain bearing rod ends for airframe applications and specific spherical plain baring rod ends for direct connection to hydraulic cylinders are not covered by this part of ISO 12240.

### 2 Normative references

The following standards contain provisions which, through reference in this text, constitute provisions of this part of ISO 12240. At the time of publication, the editions indicated were valid. All standards are subject to revision, and parties to agreements based on this part of ISO 12240 are encouraged to investigate the possibility of applying the most recent editions of the standards indicated below. Members of IEC and ISO maintain registers of currently valid International Standards.

ISO 582:1995, Rolling bearings – Chamfer dimensions – Maximum values.

ISO 965-1:1992, ISO general purpose metric screw threads – Tolerances – Part 1: Principles and basic data.

ISO 1132-1: —<sup>1)</sup>, Rolling bearings – Tolerances – Part 1: Terms and definitions.

ISO 6811:1998, Spherical plain bearings – Vocabulary.

ISO 12240-1:1998, Spherical plain bearings – Part 1: Radial spherical plain bearings.

<sup>1)</sup> To be published. (Revision of ISO 1132:1980)

## 3 Definitions and symbols

For the purposes of this part of ISO 12240, the definitions given in ISO 1132-1 and ISO 6811 apply. The symbols (except those for tolerances) shown in the figures and the values given in the tables denote nominal dimensions unless specified otherwise.

В	Inner ring width
С	Outer ring width
<i>C</i> <sub>1</sub>	Width of rod end eye
D	Outside diameter of bearing outer ring
d	Bore diameter of inner ring
<i>d</i> <sub>1</sub>	Outside diameter of inner ring face
d2	Outside diameter of rod end eye
<i>d</i> <sub>3</sub>	Rod end shank diameter
$d_4$	Rod end shank shoulder diameter
$d_5$	Rod end shank diameter with welding end
d6	Centre pin diameter
$d_{\mathbf{k}}$	Sphere diameter
G	Diameter of thread iTeh STANDARD PREVIEW
h, h <sub>1</sub> , h <sub>2</sub>	Centre height of rod end (standards.iteh.ai)
l <sub>1</sub> , l <sub>3</sub>	Thread length
l <sub>2</sub> , l <sub>4</sub> , l <sub>6</sub>	Overall length of rod end https://standards.iteh.ai/catalog/standards/sist/470dfaed-053f-4f8b-a5f6-
$l_5$	Length of shoulder on rod end shank0a58e4741/iso-12240-4-1998
l7	Length of the flat surface from the bearing bore centre to the shank
l <sub>8</sub>	Length of the locating pin
<sup>r</sup> s min <sup>2)</sup>	Smallest single chamfer dimension, inner ring
<sup>r</sup> 1s min <sup>2)</sup>	Smallest single chamfer dimension, outer ring
V <sub>dmp</sub>	Variation of mean bore diameter
V <sub>d</sub> p	Variation of bore diameter in a single radial plane
W	Width across flats
α	Angle of tilt
$\Delta_{BS}$	Deviation of a single inner ring width
$\Delta_{dmp}$	Deviation of mean bore diameter in a single plane
amp	

<sup>2)</sup> The corresponding maximum chamfer dimensions are given in table 1 of ISO 582:1995.

#### 4 Dimension series for spherical plain bearing rod ends

A distinction is made between two basic dimension series in the case of spherical plain bearing rod ends.

Dimension series E and G have been designed so as to permit the insertion of dimension series E or G radial spherical plain bearings into the cylindrical bore of a rod end eye.

In dimension series E and G a distinction is also made between different spherical plain bearing rod end types according to the shank design, i.e. external or internal thread, normal or strengthened form, or welding end type.

Dimension series K has been designed so as to permit the insertion of dimension series K radial spherical plain bearings into the cylindrical or sphered bore of a rod end eye.

In dimension series K a distinction is made between different spherical plain bearing rod end types according to the shank design, i.e. external or internal thread. A choice of sliding material combination is provided for a two piece (integral design, see figure 5) spherical plain bearing rod end.

#### 5 Angles of tilt $\alpha$

The specified angles of tilt (approximate values) represent the angles by which the axes of the inner ring and of the outer ring may be inclined in relation to each other without reducing the projected theoretical contact area of the two bearing rings when the two ring axes are parallel to each other.

NOTE — Attention is drawn to the fact that after mounting a spherical plain bearing rod end on a shaft, the angle through which the rod end can tilt may be restricted by the design of the adjacent components.

## 6 Dimensions, tolerances and radial internal clearances.

#### 6.1 Dimensions

See figures 1 to 6 and tables 1 to 5.

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The position of the lubricating nipple may vary according to the size of the spherical plain bearing rod end.

In case of type B, the lubricating nipple may be positioned at the shank.

The position and size of the lubricating nipple shall be the subject of agreement.

Type and design of lubricating nipple at manufacturer's discretion.



Figure 1 — Spherical plain bearing rod end with external thread type M



Values for widths across flats are not specified in this part of ISO 12240.
 See figure 6.

*l*<sub>8</sub> = 6 mm

# Figure 2 — Spherical plain bearing rod and with internal thread type F V rod and with welding shank type S (standards.iteh.ai)



Figure 4 — Spherical plain bearing rod end with mounted spherical plain radial bearing (cartridge design) Figure 5 — Spherical plain bearing rod end with inner ring only (integral design) <sup>3)</sup>



<sup>3)</sup> This design can apply for types M and F of the K-series.

				1	<u> </u>	1	r –	1	<del></del>	<u>1</u>	1	1	_	-	1	<u> </u>	<u> </u>	r	<u> </u>	1	1
With welding shank	$d_6$		шш	1	1	I	m	m	4	4	4	4	4	4	4	9	9	9	9	9	
ding s	$d_5$	max.	шш	1	<u>ا</u>	1	16	19	22	25	29	35	42	49	54	99	64	72	82	97	
n weld	1 <sub>6</sub>	max.	E	I	ı	I	40	45	52	59	99	78	89	104	118	132	150	173	199	237	
With internal thread With	<sup>z</sup> y		ш Ш	1	1	1	24	27	31	35	38	45	51	61	69	17	88	100	115	141	lipple
	$d_4$	max.	шш	14	14	17	20	23	27	31	36	44	52	60	67	72	77	90	100	112	ating r
	$d_3$	n	шш	11	11	13	16	19	22	25	28	35	42	47	52	58	62	70	80	95	ubric
	l5	N	E E	2	ъ	പ	6,5	6,5	∞	10	10	12	15	15	18	20	20	20	20	25	l a l
	l4	max.	С Ш	43	43	50	60	69	83	92	106	128	149	169	191	199	219	246	284	324	throu
With	l <sub>3</sub>	min.	шш	11	1	15	15	18	21	24	8	36	45	09	65	65	68	70	80	85	e provision for relubrication. sion for relubrication through a lubrication hole and not through a lubricating nipple.
	h1		шш	30	30	36	43	50	61	67	77	94	110	125	142	145	160	175	200	230	
rnal	<i>l</i> <sub>2</sub>	max.	С Ш	49	49	56	65	73	85	94	107	128	149	184	199	217	244	281	319	364	ation
With external thread	۲1	min.	un M	<u>0</u>	2	5	26	2	34	36	5	53	65	8	86	32	104	115	125	140	ubric
Vit	Ч		E	36	36	42	<b>8</b>	<mark>æ</mark> r	63	69	<b>6</b>	6	110	140	150	163	185	210	235	270	n. ugh a
	$l_{7}$	min.	E ht	e bs://	2 stanc	- lards	۲ iteh	[ ai∕ca	<u>160</u> talog	stan	4 <del>0</del> -4 Nard	: <del>29</del> 9 s/sist	8 <b>4</b> 470	0 <del>7</del> dfaeo	9 <b>4</b> 0	0 6 8 1-41	8 Bb-a	-913	85	98	ricatic 1 throu
	$d_2$	max.	E E	22	22	25	gbi	5 <mark>99</mark> 15	8 <b>6</b> 4	7 <b>4</b> ]/i	ि <mark>त्</mark> द्र 1	2 <mark>68</mark> 1(	4	9 <b>8</b> 88	94	104	114	137	162	182	ve provision for relubrication. ision for relubrication throug
welding shank	C1	max.	шш	4,5	4,5	6,5	7,5	8,5	10,5	11,5	13,5	18	20	22	24	28	31	39	43	48	
	G			M5	M6	M8	M10	M12	M14	M16	M20×1,5	M24×2	M30×2	M36×3	M39×3	M42×3	M45×3	M52×3	M56×4	M64×4	
ead or	б	ŭ	0	13	13	15	12	10	œ	10	თ	7	9	9	7	~	ø	9	9	9	not hë 'e pro
With external or internal thread	r <sub>1s</sub> 1)	min.	E	0,3	0,3	0,3	0,3	0,3	0,3	0,3	0,3	0,6	0,6	-	-	-	-	-	-	-	n. ds do ds hav
ntern	rs	min.	ш	0,3	0,3	0,3	0,3	0,3	0,3	0,3	0,3	0,6	0,6	0,6	0,6	0,6	0,6	-	-	-	desig od en od en
al or i	$d_k^{(2)}$		ш	10	10	13	16	18	22	25	29	35	40	47	53	60	66	80	92	105	tegral ring r ring r
xtern	<i>C</i> 1)		E	4	4	5	9	7	თ	10	12	16	18	20	22	25	28	36	40	45	Reference only; not for integral design. Reference only. These spherical plain bearing rod ends do not hav These spherical plain bearing rod ends have provi
Vith e	В		ш	Q	9	∞	ი	10	12	14	16	20	22	25	28	32	35	44	49	55	
	٩ı	u	mm	œ	ω	10	13	15	18	20	24	29	34	39	45	50	55	99	77	88	ice on ice on pheric pheric
	D1)		mm	14	14	16	19	22	26	30	35	42	47	55	62	68	75	06	105	120	Reference only; Reference only. These spherical These spherical
	q		mm	53)	6 3)	8 3)	10 3)	12 3)	15 4)	17 4)	20 4)	25	30	35	40	45	50	60	70	80	4) 3) 1 4) 3) 1 4) 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1

Table 1 — Spherical plain bearing rod ends, dimension series E

						ensio							
		$d_3$	u	E	49	58	65	70	82	92	105		
	thread	l5	X	ш ш	25	25	30	30	35	40	45		
	With internal thread	<i>l</i> 4	max.	mm	174	194	219	254	296	349	389		
	With i	<i>l</i> <sub>3</sub>	min.	mm	60	65	65	68	70	80	85		
		۱ų		ш ш	130	145	165	195	225	265	295		
	thread	l <sub>2</sub>	max.	Ē	174	194	219	254	296	349	389		
	With external thread	l,	min.	E E	82	06	95	110	120	132	147		
		ч		E E	130	145	165	195	225	265	295		
	Т	ч 1	nin T	E	40	46	50	58	73	85	98	VIEV	V
		$d_2$	(max.	E	<b>8</b>	94	104	4. 5.i	Lee	162	182		•
http		ت tandar	max.	E ai/	CLIS Notal		2 <b>%</b> 10	<b>-5</b> 10 rds/s	2 <b>68</b> ict///	43 1016	48	)53f-4f8b-a	1516-
		G	bi	f50a	M36×3	M42×35	M45×3	M52×37	40×4 ₩60×4	M72×4	M80×4 <sup>~</sup>	<i>1331-</i> 1100- <i>C</i>	1510-
	2												
	nal th	α	2	•	9	7	7	9	9	9	9		
	or internal thread	$r_{1s}^{1} \alpha$		° mm	1 6	1 7	1 7	1 6	1 6	1 6	1 6		
			min.	E E	0,6 1 6	0,6 1 7	0,6 1 7	0,6 1 6	1 1 6	1 1 6	1 1 6		
	With external or internal th	r <sub>1s</sub> <sup>1</sup> )	min. min.	E E	1	1	1	1	80 1 1 6	92 1 1 6	105 1 1 6	design.	
		r <sub>s</sub> r <sub>1s</sub> <sup>1)</sup>	min. min.	mm mm mm	0,6 1	0,6 1	0,6 1	0,6 1	1 1	1	1 1	ıtegral design.	
		$d_{\rm k}^{(2)} = r_{\rm s} = r_{\rm 1s}^{(1)}$	min.		47 0,6 1	53 0,6 1	60 0,6 1	66 0,6 1	80 1 1	92 1 1	105 1 1	ot for integral design.	
		C <sup>1</sup> ) $d_k^{21}$ $r_s$ $r_{1s}^{11}$	min.	mm mm mm	20 47 0,6 1	22 53 0,6 1	25 60 0,6 1	28 66 0,6 1	36 80 1 1	40 92 1 1	45 105 1 1	only; not for integral design. only.	
		B $C^{1}$ $d_{k}^{2}$ $r_{s}$ $r_{1s}^{1}$	s min.		25 20 47 0,6 1	28 22 53 0,6 1	32 25 60 0,6 1	35 28 66 0,6 1	44 36 80 1 1	49 40 92 1 1	55 45 105 1 1	ference only; not for integral design. ference only.	
		$d_1 \qquad B \qquad C^{1} \qquad d_k^{2} \qquad r_s \qquad r_{1s}^{1}$	s min.		39 25 20 47 0,6 1	45         28         22         53         0,6         1	50 32 25 60 0,6 1	55 35 28 66 0,6 1	66         44         36         80         1         1	77 49 40 92 1 1	88 55 45 105 1 1	<ol> <li>Reference only; not for integral design.</li> <li>Reference only.</li> </ol>	

Table 2 — Spherical plain bearing rod ends, according to dimension series E, with strengthened shank, dimension series EH

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