INTERNATIONAL STANDARD

ISO 15086-2

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Hydraulic fluid power — Determination of fluid-borne noise characteristics of components and systems —

Part 2:

Measurement of speed of sound in a fluid in iTeh Salpipedard PREVIEW

Transmissions hydrauliques a Evaluation des caractéristiques du bruit liquidien des composants et systèmes —

Partie 2: Mesurage de la vitesse du son émis dans un fluide dans une https://standards.tell.av.ca.alog/standards/sist/048clb3b-0593-4157-9101-411ac65e5cdd/iso-15086-2-2000



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Foreword

ISO (the International Organization for Standardization) is a worldwide federation of national standards bodies (ISO member bodies). The work of preparing International Standards is normally carried out through ISO technical committees. Each member body interested in a subject for which a technical committee has been established has the right to be represented on that committee. International organizations, governmental and non-governmental, in liaison with ISO, also take part in the work. ISO collaborates closely with the International Electrotechnical Commission (IEC) on all matters of electrotechnical standardization.

International Standards are drafted in accordance with the rules given in the ISO/IEC Directives, Part 3.

Draft International Standards adopted by the technical committees are circulated to the member bodies for voting. Publication as an International Standard requires approval by at least 75 % of the member bodies casting a vote.

Attention is drawn to the possibility that some of the elements of this part of ISO 15086 may be the subject of patent rights. ISO shall not be held responsible for identifying any or all such patent rights.

International Standard ISO 15086-2 was prepared by Technical Committee ISO/TC 131, *Fluid power systems*, Subcommittee SC 8, *Product testing*.

ISO 15086 consists of the following parts, under the general title *Hydraulic fluid power* — *Determination of fluid borne noise characteristics of components and systems*:

Part 1: Introduction (standards.iteh.ai)

— Tart T. Introduction

— Part 2: Measurement of the speed of sound in a fluid in a pipe https://standards.itch.ai/catalog/standards/sist/048cfb3b-0593-4f57-910f-

Annexes A, B and C form a normative part of this part of ISO 15086. Annexes D and E are for information only

Introduction

In hydraulic fluid power systems, power is transmitted and controlled through a liquid under pressure within an enclosed circuit. During the process of converting mechanical power into hydraulic fluid power, flow and pressure fluctuations and structure-borne vibrations are generated.

Hydro-acoustical characteristics of hydraulic components can be measured with acceptable accuracy if the speed of sound in the fluid is precisely known.

The measurement technique for determining the speed of sound in a pipe, as described in this part of ISO 15086, is based upon the application of plane wave transmission line theory to the analysis of pressure fluctuations in rigid pipes [1].

Two different measurement approaches are presented, namely the use of:

- three pressure transducers in a pipe,
- acoustic antiresonance in a closed-end pipe system.

The three-pressure-transducer method should be used at any time when the speed of sound is to be measured under the effective working conditions in a system. DARD PREVIEW

The antiresonance method should be used to produce a table of speed-of-sound data as a function of mean pressure and temperature for a particular fluid.

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Hydraulic fluid power — Determination of fluid borne noise characteristics of components and systems —

Part 2:

Measurement of the speed of sound in a fluid in a pipe

1 Scope

This part of ISO 15086 describes the procedure for the determination of the speed of sound in a fluid enclosed in a pipe, by measurements from pressure transducers mounted in the pipe.

This part of ISO 15086 is applicable to all types of hydraulic circuit operating under steady state conditions, irrespective of size, for pressure pulsations over a frequency range from 25 Hz to 2 500 Hz.

2 Normative references eh STANDARD PREVIEW

The following normative documents contain provisions which, through reference in this text, constitute provisions of this part of ISO 15086. For dated references, subsequent amendments to, or revisions of, any of these publications do not apply. However, parties to agreements based on this part of ISO 15086 are encouraged to investigate the possibility of applying the most recent editions of the normative document references, the latest edition of the normative document referred to applies. Members of ISO and IEC maintain registers of currently valid International Standards.

ISO 1000:1992, SI units and recommendations for the use of their multiples and of certain other units.

ISO 1219-1:1991, Fluid power systems and components — Graphic symbols and circuit diagrams — Part 1: Graphic symbols.

ISO 5598:1985, Fluid power systems and components — Vocabulary.

3 Terms and definitions

For the purposes of this part of ISO 15086, the terms and definitions given in ISO 5598 and the following apply.

3.1

flow ripple

fluctuating component of flowrate in a hydraulic fluid, caused by interaction with a flow ripple source within the system

3.2

pressure ripple

fluctuating component of pressure in a hydraulic fluid, caused by interaction with a flow ripple source within the system

3.3

fundamental frequency

lowest frequency of pressure ripple measured by the frequency-analysis instrument

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3.4

harmonic

sinusoidal component of the pressure ripple or flow ripple occurring at an integer multiple of the fundamental frequency

NOTE A harmonic may be represented by its amplitude and phase, or alternatively by its real and imaginary parts.

3.5

hydraulic noise generator

hydraulic component generating flow ripple and consequently pressure ripple in the circuit

3.6

measurement pipe

pipe in which the pressure transducers are mounted

3.7

impedance

complex ratio of the pressure ripple to the flow ripple occurring at a given point in a hydraulic system and at a given frequency

3.8

entry impedance.

impedance at the entry of a pipe or piping system

3.9

first acoustic antiresonance frequency STANDARD PREVIEW

lowest frequency at which the magnitude of the entry impedance of the measurement pipe is at a minimum (Standards.1teh.al)

4 Symbols and subscripts

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4.1 Symbols

A, A', a, B, B', b Frequency-dependent wave propagation coefficients (complex numbers)

- c Acoustic velocity in the fluid
- d Internal diameter of pipe
- f Frequency of the wave pulsation harmonic
- f_i Vector of frequencies at which measurements are conducted
- f_o First acoustic antiresonance frequency (in hertz)
- H Transfer function (complex number) between two pressure transducer signals after calibration

correction

- H' Transfer function (complex number) between two pressure transducer signals under calibration
- H* Transfer function (complex number) between two pressure transducer signals
- $j \sqrt{-1}$
- L Distance between transducers 1 and 2 (Method 1)
- L' Distance between transducers 2 and 3 (Method 1)

l	Distance between pressure transducers (Method 2)
P_1	Pressure ripple of transducer PT1 (complex number)
P_2	Pressure ripple of transducer PT2 (complex number)
P_3	Pressure ripple of transducer PT3 (complex number)
$Q_1 \rightarrow 2$	Flow ripple at location 1, from 1 to 2 (complex number)
$Q_2 \rightarrow 1$	Flow ripple at location 2, from 2 to 1 (complex number)
$Q_{2\rightarrow 3}$	Flow ripple at location 3, from 2 to 3 (complex number)
S_i	Coherence function corresponding to measurement frequencies, f_i
\mathcal{E}	Error (complex number)
$\overline{arepsilon}$	Conjugate of complex number ε (complex number)
\mathcal{E}_{X}	Real part of ε
$arepsilon_{y}$	Imaginary part of $arepsilon$
ρ	Density of fluid (standards itch ai)
ν	(standards.iteh.ai) Kinematic viscosity of fluid
ω	2πf ISO 15086-2:2000 https://standards.iteh.ai/catalog/standards/sist/048cfb3b-0593-4f57-910f-
NOTE	$H, H', H^*, P_1, P_2, P_3, Q_{1 \to 2}, Q_{2 \to 1}, Q_{2 \to 3}$ are all frequency-dependent terms and hence are designated by upper-

case letters.

Units used in this part of ISO 15086 are in accordance with ISO 1000.

Graphical symbols are in accordance with ISO 1219-1 unless otherwise stated.

4.2 Subscripts

O Index for old value

N Index for new value

5 Instrumentation

5.1 Static measurements

The instruments used to measure

- a) mean flow (Method 1 only);
- b) mean fluid pressure;
- c) fluid temperature;

shall meet the requirements for "industrial class" accuracy of measurement, i.e. class C as given in annex B.

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5.2 Dynamic measurements

The instruments used to measure pressure ripple shall have the following characteristics:

- a) resonant frequency ≥ 30 kHz;
- b) linearity $\geq \pm 1 \%$;
- c) preferably include acceleration compensation.

The instruments need not respond to steady-state pressure. It may be advantageous to filter out any steady-state signal component using a high-pass filter. This filter shall not introduce an additional amplitude or phase error exceeding 0,5 % or 0,5° respectively of the current measurement.

5.3 Frequency analysis of pressure ripple

A suitable instrument shall be used to measure the amplitude and phase of the pressure ripple.

The instrument shall be capable of measuring the pressure ripple from the pressure transducers such that, for a particular harmonic, the measurements from each transducer are performed simultaneously and synchronised in time with respect to each other.

The instrument shall have an accuracy and resolution for harmonic measurements of

a) amplitude within ± 0,5 %; Teh STANDARD PREVIEW

b) phase within $\pm 0.5^{\circ}$; (standards.iteh.ai)

c) frequency within $\pm 0.5\%$; ISO 15086-2:2000

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over the frequency range from 25 Hz to 2 500 Hzc65e5cdd/iso-15086-2-2000

5.4 Uncertainty

Compliance with the above specification will result in an uncertainty in measurement of speed of sound of less than \pm 3 %.

6 Hydraulic noise generator

6.1 General

Any type of hydraulic noise generator may be used, provided that sufficient pressure ripple is created at the pressure transducers to allow accurate measurements to be taken.

EXAMPLE Pumps and motors create a pressure ripple consisting essentially of many harmonics of the fundamental frequency. In these cases, the fundamental frequency is equal to the product of the shaft rotational frequency and the number of gear teeth, vanes or pistons, etc. (as appropriate to the machine employed).

Suitable alternatives include:

- an auxiliary valve with a rotating spool allowing flow to pass to the return line over part of its rotation;
- an electrohydraulic servo-valve driven by a frequency generator.
- The servo-valve may be operated with a white noise signal in order to obtain significant pressure ripple measurements at each frequency of interest.

6.2 Generator vibration

If necessary, the measurement pipe should be structurally isolated from the generator to minimize vibration.

7 Test conditions

7.1 General

The required operating conditions shall be maintained throughout each test within the limits specified in Table 1.

7.2 Fluid temperature

The temperature of the fluid shall be that measured at the entry to the measurement pipe.

7.3 Fluid density and viscosity

The density and viscosity of the fluid shall be known to an accuracy within the limits specified in Table 2.

7.4 Mean fluid pressure

The mean fluid pressure of the fluid shall be that measured at the entry to the measurement pipe.

7.5 Mean flow measurementh STANDARD PREVIEW

The mean flow shall be measured down-stream of the measurement pipe (Method 1 only).

Table 1 — Permissible variations in tests conditions. https://standards.iteh.ai/catalog/standards/ciet/1/2 softs is 1/2 softs in tests.

Test parameter d/iso	Permissible variation
Mean flow	± 2 %
Mean pressure	± 2 %
Temperature	± 2 °C

Table 2 — Required accuracy of fluid property data

Property	Required accuracy
Density	± 2 %
Viscosity	± 5 %

8 Test rig

8.1 General

If, at any test condition, the pressure ripple amplitudes are too small for satisfactory frequency-spectrum analysis to be performed, an alternative noise generator shall be selected.

The pressure transducers shall be mounted such that their diaphragms are flush, within \pm 0,5 mm, with the inner wall of the pipe.

Two alternative specifications for the measurement pipe and transducer position are given, in accordance with the method employed.

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8.2 Thermal insulation

Temperature shall be measured at both ends of the measurement pipe. The difference in temperature between the two ends of the measurement pipe shall not exceed 2 °C at any test condition. If necessary, sufficient thermal lagging shall be applied to the measurement pipe to enable this requirement to be met.

8.3 Method 1: Three-transducer method

8.3.1 This method is suitable when the velocity of sound is to be measured at the same time as other hydro-acoustical characteristics of hydraulic components, such as impedance, source flow ripple or transfer matrix coefficients. The measurement pipe shall be installed at the place in the test system where measurement of the speed of sound is needed. Several measurement pipes may be used simultaneously, if required.

The measurement pipe shall be uniform and straight. Its internal diameter shall be between 80 % and 120 % of the diameter of the pipes, or component ports, to which it is connected. The pipe should be supported in such a manner that vibration is minimized.

For cases where other hydro-acoustic properties are not being measured simultaneously, a pump (and if necessary, a hydraulic noise generator) shall be mounted at one end of the measurement pipe. The other end shall be terminated by a loading valve without free-moving internal parts, such as a needle valve.

Mean pressure shall be measured at the upstream end of the measurement pipe.

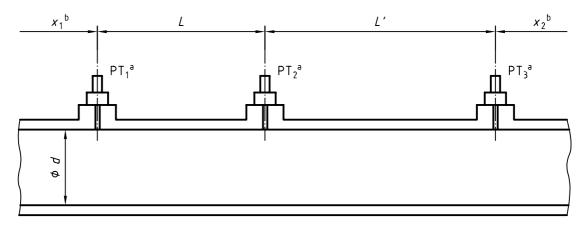
8.3.2 Three pressure transducers are required for Method 1, configured as shown in Figure 1. The transducer spacing shall be selected according to the standard specifications of hydro-acoustical measurements to be carried out simultaneously. Otherwise, the distances *L* and *L* between the pressure transducers shall be as specified in Table 3.

(standards.iteh.ai)
Table 3 — Spacing of transducers: Method 1

	$L = \frac{180 \cdot 150}{1}$	80-330 mm ± 2 mm	
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1	4 <u>4</u> 1ac65e5cdd/	_{SO-1} 470 mm2±120mm	

The distance between each end of the measurement pipe and the nearest pressure transducer shall be at least 10 d, where d is the internal diameter of the pipe. The distances L and L' between the transducers, as shown in Figure 1, shall be measured to an accuracy of \pm 0,5 mm.

No other components shall be connected between the inlet port and outlet port of the measurement pipe.



- a Pressure transducers.
- b Distances to end of measurement pipe, $x_1 \ge 10d$ and $x_2 \ge 10d$.

Figure 1 — Arrangement of three pressure transducers in measurement pipe