



SLOVENSKI STANDARD
SIST EN 13384-1:2003+A2:2008
01-junij-2008

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Chimneys - Thermal and fluid dynamic calculation methods - Part 1: Chimneys serving one appliance

Abgasanlagen - Wärme- und strömungstechnische Berechnungsverfahren - Teil 1: Abgasanlagen mit einer Feuerstätte

Conduits de fumée - Méthodes de calcul thermo-aéraulique - Partie 1: Conduits de fumée ne desservant qu'un seul appareil

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Ta slovenski standard je istoveten z: EN 13384-1:2002+A2:2008

ICS:

91.060.40 Dimniki, jaški, kanali Chimneys, shafts, ducts

SIST EN 13384-1:2003+A2:2008 en,fr

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English Version

Chimneys - Thermal and fluid dynamic calculation methods - Part 1: Chimneys serving one appliance

Conduits de fumée - Méthodes de calcul thermo-aéraulique
- Partie 1: Conduits de fumée ne desservant qu'un seul
appareil

Abgasanlagen - Wärme- und strömungstechnische
Berechnungsverfahren - Teil 1: Abgasanlagen mit einer
Feuerstätte

This European Standard was approved by CEN on 23 October 2002 and includes Corrigendum 1 issued by CEN on 17 December 2003, Amendment 1 approved by CEN on 19 September 2005 and Amendment 2 approved by CEN on 24 February 2008.

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Foreword

This document (EN 13384-1:2002+A2:2008) has been prepared by Technical Committee CEN/TC 166, "Chimneys", the secretariat of which is held by UNI.

This European Standard shall be given the status of a national standard, either by publication of an identical text or by endorsement, at the latest by October 2008 and conflicting national standards shall be withdrawn at the latest by October 2008.

This document includes Corrigendum 1 issued by CEN on 2003-12-17, Amendment 1, approved by CEN on 2005-09-19 and Amendment 2 approved by CEN on 2008-02-24.

This document supersedes EN 13384-1:2002.

The start and finish of text introduced or altered by amendment is indicated in the text by tags $\boxed{A_1}$ $\boxed{A_1}$ and $\boxed{A_2}$ $\boxed{A_2}$.

The modifications of the related CEN Corrigendum have been implemented at the appropriate places in the text and are indicated by the tags \boxed{AC} \boxed{AC} .

This European Standard has been prepared under a mandate given to CEN by the European Commission and the European Free Trade Association and supports essential requirements of EU Directives.

Annexes A, B, C and D are informative.

This European Standard "Chimneys – Thermal and fluid dynamic calculation methods" consists of $\boxed{A_1}$ three $\boxed{A_1}$ Parts:

- *Part 1: Chimneys serving one heating appliance*
- *Part 2: Chimneys with multiple inlets and one inlet with multiple appliances*
- $\boxed{A_1}$ *Part 3: Methods for the development of diagrams and tables for chimneys serving one heating appliance* $\boxed{A_1}$

According to the CEN/CENELEC Internal Regulations, the national standards organizations of the following countries are bound to implement this European Standard: Austria, Belgium, Bulgaria, Cyprus, Czech Republic, Denmark, Estonia, Finland, France, Germany, Greece, Hungary, Iceland, Ireland, Italy, Latvia, Lithuania, Luxembourg, Malta, Netherlands, Norway, Poland, Portugal, Romania, Slovakia, Slovenia, Spain, Sweden, Switzerland and United Kingdom.

1 Scope

This European Standard specifies methods for the calculation of the thermal and fluid dynamic characteristics of chimneys serving one appliance.

The methods in this Part of this European Standard are applicable to negative or positive pressure chimneys with wet or dry operating conditions. It is valid for chimneys with heating appliances for fuels subject to the knowledge of the flue gas characteristics which are needed for the calculation.

The methods in this Part of this European Standard are applicable to chimneys with one inlet connected with one appliance. The methods in Part 2 of this European Standard are applicable to chimneys with multiple inlets and one inlet with multiple appliances. ^[A1] Part 3 describes methods for the development of diagrams and tables for chimneys serving one heating appliance. ^[A1]

2 Normative references

This European Standard incorporates by dated or undated reference, provisions from other publications. These normative references are cited at the appropriate places in the text and the publications are listed hereafter. For dated references, subsequent amendments to, or revisions of, any of these publications apply to this European Standard only when incorporated in it by amendment or revision. For undated references the latest edition of the publication referred to applies (including amendments).

^[A2] EN 1443:2003 ^[A2], *Chimneys - General requirements*

^[A1] EN 1856-1 ^[A1], *Chimneys – Requirements for metal chimneys – Part 1: System chimney products*
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EN 1859, *Chimneys - Metal chimneys - Test methods*

EN 13502, *Chimneys - Requirements and test methods for clay/ceramic flue terminals*

^[A2] CEN/TR 1749, *European scheme for the classification of gas appliances according to the method of evacuation of the combustion products (types)* ^[A2]

3 Terms and definitions

For the purposes of this European Standard, the terms and definitions given in ^[A2] EN 1443:2003 ^[A2] and the following apply.

3.1

heat output (Q)

amount of heat produced by a heating appliance per unit of time

3.1.1

nominal heat output (Q_N)

continuous heat output specified by the manufacturer of the heating appliance related to specified fuels

3.1.2

heat output range

range of output below the nominal heat output specified by the manufacturer over which the appliance can be used

3.2

heat input (Q_F)

amount of heat in unit time which is supplied to the heating appliance by the fuel based on its net calorific value H_u

3.3

efficiency of the heating appliance (η_w)

ratio of the heat output (Q) from the appliance to the heat input (Q_F)

3.4

flue gas mass flow (\dot{m})

mass of flue gas leaving the heating appliance through the connecting flue pipe per time unit

3.5

effective height of the chimney (H)

difference in height between the axis of the flue gas inlet into the chimney and the outlet of the chimney

3.6

effective height of the connecting flue pipe (H_v)

difference in height between the axis of the flue gas chimney outlet of the heating appliance and the axis of the flue gas inlet into the chimney

In the case of open fire chimneys, H_v is the difference in height between the height of the upper frame of the furnace and the axis of the flue gas inlet into the chimney

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3.7

draught

positive value of the negative pressure in the flue

3.8

theoretical draught available due to chimney effect (P_H)

pressure difference caused by the difference in weight between the column of air equal to the effective height outside a chimney and the column of flue gas equal to the effective height inside the chimney

3.9

pressure resistance of the chimney (P_R)

pressure which is necessary to overcome the resistance of the flue gas mass flow which exists when carrying the flue gases through the chimney

3.10

wind velocity pressure (P_L)

pressure generated on the chimney due to wind

3.11

ΔP_2 minimum ΔP_2 draught at the flue gas inlet into the chimney (P_Z)

ΔP_2 difference between the minimum theoretical draught and the sum of the maximum pressure resistance of the chimney and the wind velocity pressure ΔP_L

A₂ 3.12**maximum draught at the flue gas inlet into the chimney ($P_{Z_{max}}$)**

difference between the maximum theoretical draught and the minimum pressure resistance in the chimney **A₂**

3.13

minimum draught for the heating appliance (P_w)

difference between the static air pressure of the room of installation of the heating appliance and the static pressure of the flue gas at the chimney outlet of the appliance which is necessary to maintain the correct operation of the heating appliance

A₂ 3.14**maximum draught for the heating appliance ($P_{W_{max}}$)**

difference between the static air pressure of the room of installation of the heating appliance and the static pressure of the flue gas at the outlet of the appliance which is the maximum allowed to maintain the correct operation of the heating appliance **A₂**

3.15

effective pressure resistance of the connecting flue pipe (P_{FV})

static pressure difference between the axis of the inlet of the connecting flue pipe and the axis of the chimney outlet due to the theoretical draught and pressure resistance

3.16

effective pressure resistance of the air supply (P_E)

difference between the static pressure in the open air and the static air pressure in the room of installation of the heating appliance at the same height

3.17

A₂ **minimum **A₂** draught required at the flue gas inlet into the chimney (P_{Z_e})**

sum of the minimum draught required for the heating appliance and the draught required to overcome the effective pressure resistance of the connecting flue pipe and the effective pressure resistance of the air supply

A₂ 3.18**maximum allowed draught at the flue gas inlet into the chimney ($P_{Z_{e_{max}}}$)**

sum of the maximum draught allowed for the heating appliance and the draught required to overcome the effective pressure resistance of the connecting flue pipe and the effective pressure resistance of the air supply **A₂**

3.19

A₂ **maximum **A₂** positive pressure at the flue gas inlet into the chimney (P_{Z_0})**

A₂ sum of the difference of the maximum pressure resistance and the minimum theoretical draught of the chimney and the wind velocity pressure **A₂**

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A2 3.20

minimum positive pressure at the flue gas inlet into the chimney (P_{ZOmin})

difference of the minimum pressure resistance and the maximum theoretical draught of the chimney **A2**

3.21

maximum differential pressure of the heating appliance (P_{WO})

maximum difference between the static pressure of the flue gas at the chimney outlet of the appliance and the static pressure of the air at the inlet to the heating appliance specified for its correct operation

A2 3.22

minimum differential pressure of the heating appliance (P_{WOmin})

minimum difference between the static pressure of the flue gas at the outlet of the appliance and the static pressure of the air at the inlet to the heating appliance specified for its correct operation. This can be a negative value. **A2**

3.23

maximum differential pressure at the flue gas inlet into the chimney (P_{ZOe})

difference between the maximum differential pressure of the heating appliance and the sum of the effective pressure resistance of the connecting flue pipe and the effective pressure resistance of the air supply

3.24

secondary air

ambient air added to the flue gas in addition to the nominal flue gas mass flow

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A2 3.25

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minimum differential pressure at the flue gas inlet into the chimney (P_{ZOemin})

difference between the minimum differential pressure of the heating appliance and the sum of the effective pressure resistance of the connecting flue pipe and the effective pressure resistance of the air supply **A2**

3.26

secondary air device

draught regulator or a draught diverter

3.27

draught regulator

component which automatically supplies ambient air to the chimney, the connecting flue pipe or the heating appliance

3.28

draught diverter

device, placed in the combustion products passage of the heating appliance, that is intended to maintain the quality of combustion within certain limits and to keep the combustion stable under certain conditions of updraught and downdraught

3.29

temperature limit of the inner wall (T_g)

allowed minimum temperature of the inner wall of the chimney outlet

A1 3.30**air-flue gas system**

system of concentric or non-concentric ducts for transport of combustion air from the open air to the heating appliance and products of combustion from the heating appliance to the open air

3.31**air-supply duct**

component or components parallel to the chimney (separate or concentric) that conveys combustion air from the outside atmosphere to the inlet of the connecting air supply pipe

3.32**balanced flue chimney**

chimney where the point of air entry to the air supply duct is adjacent to the point of discharge of combustion products from the flue, the inlet and outlet being so positioned that wind effects are substantially balanced

3.33**chimney segment**

calculation part of a chimney

3.34**condensate mass flow ($\Delta\dot{m}_D$)**

mass of water vapour of the flue gas condensed in the heating appliance, connecting flue pipe or the chimney per time unit

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3.35**connecting air supply pipe (standards.iteh.ai)**

component or components connecting the air supply duct outlet with the room-sealed heating appliance combustion air inlet

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3.36**condensation factor (f_K)**

proportion of the theoretical maximum condensation mass flow usable in the calculation **A1**

4 Symbols, terminology and units

The symbols given in this clause can be completed by one or more indices to indicate location or materials if necessary.

Table 1 - Symbols, terminology and units

Symbol	Terminology	Unit
A	cross section area	m^2
c	specific heat capacity	$J/(kg \cdot K)$
c_p	specific heat capacity of flue gas	$J/(kg \cdot K)$
d	thickness of the section	m
D	diameter	m
D_h	hydraulic diameter	m
H	effective height of the chimney	m
k	coefficient for heat transmission	$W/(m^2 \cdot K)$
K	coefficient of cooling	--
L	length	m
\dot{m}	flue gas mass flow	kg/s
Nu	Nusselt number	--
p	static pressure	Pa
p_L	external air pressure	Pa
P_B	pressure resistance of the air supply for a flue gas mass flow	Pa
P_E	pressure resistance due to friction and form resistance of the chimney	Pa
P_{FV}	effective pressure resistance of the connecting flue pipe	Pa
P_G	difference in pressure caused by change of velocity of flue gas in the chimney	Pa
P_H	theoretical draught available due to chimney effect	Pa
P_{HV}	theoretical draught available due to chimney effect of the connecting flue pipe	Pa
P_L	wind velocity pressure	Pa
P_{NL}	draught required for secondary air devices	Pa
P_R	pressure resistance of the chimney	Pa
P_{RV}	pressure resistance of the connecting flue pipe	Pa
P_W	minimum draught for the heating appliance	Pa
$\sqrt{A_2} P_{Wmax}$	maximum draught for the heating appliance	$Pa \sqrt{A_2}$
P_{WO}	maximum differential pressure of the heating appliance	Pa
$\sqrt{A_2} P_{WOmin}$	minimum differential pressure of the heating appliance	$Pa \sqrt{A_2}$
P_Z	$\sqrt{A_2}$ minimum $\sqrt{A_2}$ draught at the flue gas inlet into the chimney	Pa
$\sqrt{A_2} P_{Zmax}$	maximum draught at the flue gas inlet into the chimney	$Pa \sqrt{A_2}$
P_{Ze}	$\sqrt{A_2}$ minimum $\sqrt{A_2}$ draught required at the flue gas inlet into the chimney	Pa
$\sqrt{A_2} P_{Zemax}$	maximum allowed draught at the flue gas inlet into the chimney	$Pa \sqrt{A_2}$
P_{ZO}	$\sqrt{A_2}$ maximum $\sqrt{A_2}$ positive pressure at the flue gas inlet into the chimney	Pa
$\sqrt{A_2} P_{ZOmin}$	minimum positive pressure at the flue gas inlet into the chimney	$Pa \sqrt{A_2}$

Table 1 (continued)

Symbol	Terminology	Unit
P_{ZOe}	maximum differential pressure at the flue gas inlet into the chimney	Pa
P_{ZOemin}	minimum differential pressure at the flue gas inlet into the chimney	Pa
Pr	Prandtl number	--
Q	heat output	kW
q_c	heat transfer from the flue to the outer surface	K
Q_F	heat input	kW
Q_N	nominal heat output	kW
r	mean value of roughness of the inner wall	m
R	gas constant of the flue gas	J/(kg·K)
R_L	gas constant of the air	J/(kg·K)
Re	Reynolds number	--
s	cross section	m
S_E	flow safety coefficient	--
S_H	correction factor for temperature instability	--
t	temperature	°C
T	temperature, absolute	K
T_g	temperature limit	K
T_{io}	inner wall temperature at chimney outlet	K
T_{iob}	inner wall temperature at the chimney outlet at temperature equilibrium	K
T_L	external air temperature	K
T_m	mean temperature of the flue gas	K
T_p	water dew point	K
T_{sp}	condensing temperature	K
T_u	ambient air temperature	K
T_{ub}	ambient air temperature of the boiler room	K
T_{uh}	ambient air temperature for heated areas	K
T_{uo}	ambient air temperature at the chimney outlet	K
T_{ul}	ambient air temperature for areas external to the building	K
T_{uu}	ambient air temperature for unheated areas inside the house	K
T_W	flue gas temperature of the appliance	K
T_{WN}	flue gas temperature of the appliance at nominal heat output	K
T_{Wmin}	flue gas temperature of the appliance at the lowest possible heat output	K
U	internal chimney segment parameter	m
w	mean velocity within a cross section	m/s
w_m	mean velocity over a defined length	m/s
y	form value	--
z	height above sea level	m
α	coefficient of heat transfer	W/(m ² ·K)
β	ratio of the combustion air mass flow to the flue gas mass flow	--
γ	angle between flow directions	°
δ	wall thickness	m

Table 1 (concluded)

Symbol	Terminology	Unit
ζ	coefficient of flow resistance due to a directional and/or cross sectional and/or mass flow change in the flue	-
η	dynamic viscosity	N·s/m ²
η_W	efficiency of the heating appliance	--
η_{WN}	efficiency of the heating appliance at nominal heat output	--
λ	coefficient of thermal conductivity	W/(m·K)
ρ	density	kg/m ³
ρ_L	density of the external air	
ρ_m	mean density of flue gas averaged over a defined length and over the cross section	kg/m ³
$\sigma(\text{CO}_2)$	volume-concentration of CO ₂	%
$\sigma(\text{H}_2\text{O})$	volume-concentration of H ₂ O (vapour)	%
σ_{Rad}	black body radiation number	W/(m ² ·K ⁴) $\langle A_1 \rangle$
ψ	coefficient of flow resistance due to friction of the flue	--
$\left(\frac{1}{\Lambda}\right)$	thermal resistance	m ² ·K/W

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Table 2 - Additional subscripts

Subscript	Terminology	Unit
a	outside	--
A	flue gas	--
b	equilibrium temperature condition	--
B	combustion air	--
e	entrance	--
G	change in velocity	--
i	inside	--
L	open air (outside)	--
m	mean value	--
M	mixture	--
n	counting index	--
N	nominal value	--
NL	secondary air	--
o	chimney outlet	--
O	positive pressure	--
tot	totalized over all sections (segments)	--
u	ambient air	--
V	connecting flue pipe	--
W	heating appliance	--

5 **A1** Calculation method for non-balanced flue chimneys **A1**

5.1 General principles

A2 The calculation of inside dimensions (cross section) of negative pressure chimneys is based on the following four criteria:

- the minimum draught at the flue gas inlet into the chimney shall be equal to or greater than the minimum draught required at the flue gas inlet into the chimney;
- the minimum draught at the flue gas inlet to the chimney shall be equal to or greater than the effective pressure resistance of the air supply;
- the maximum draught at the flue gas inlet into the chimney shall be equal to or less than the maximum allowed draught at the flue gas inlet into the chimney;
- the temperature of the inner wall at the outlet of the chimney shall be equal to or greater than the temperature limit.

The calculation of inside dimensions (cross section) of positive pressure is based on the following four criteria:

- the maximum positive pressure at the flue gas inlet into the chimney shall be equal or less than the maximum differential pressure at the flue gas inlet into the chimney;
- the maximum positive pressure in the connecting flue pipe and in the chimney shall not be higher than the excess pressure for which both are designated;
- the minimum positive pressure at the flue gas inlet into the chimney shall be equal or greater than the minimum differential pressure at the flue gas inlet into the chimney;
- the temperature of the inner wall at the chimney outlet of the chimney shall be equal to greater than the temperature limit.

NOTE The pressure requirements for maximum draught or minimum positive pressure are only required if there is a limit for the maximum draught for the negative pressure heating appliance or a minimum differential pressure of the positive pressure heating appliance.

In order to verify the criteria two sets of external conditions are used:

- the calculation of the minimum draught and maximum positive pressure is made with conditions for which the capacity of the chimney is minimal (i.e. high outside temperature); and also
- the calculation of the maximum draught and minimum positive pressure and of the inner wall temperature with conditions for which the inside temperature of the chimney is minimal (i.e. low outside temperature). **A2**

5.2 Pressure requirements

5.2.1 Negative pressure chimneys

The following relationships shall be verified:

$$P_Z = P_H - P_R - P_L \geq P_W + P_{FV} + P_B = P_{Ze} \quad \text{in Pa} \quad (1)$$