# TECHNICAL REPORT

## ISO/TR 16806

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# Pneumatic fluid power — Cylinders — Load capacity of pneumatic slides and their presentation method

Transmissions pneumatiques — Vérins — Capacité de charge des unités de guidage pneumatique et leur méthode de présentation

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ISO/TR 16806 was prepared by Technical Committee ISO/TC 131, *Fluid power systems*, Subcommittee SC 3, Cylinders.

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### Introduction

In pneumatic fluid power systems, power is transmitted and controlled through a gas under pressure within a circuit. A pneumatic slide consists of a mounting surface for attaching a load, which is moved by an air cylinder and guided by stiff shafts to maintain alignment. There are limits to the amount of load that can be attached to a pneumatic slide, and these limits should be described as shown in this Technical Report.

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# Pneumatic fluid power — Cylinders — Load capacity of pneumatic slides and their presentation method

#### 1 Scope

- 1.1 This Technical Report describes how to calculate the loading limits for a pneumatic slide based upon:
- external forces applied in the three principle planes of a tool plate, and applied at any point;
- external torque applied in the three principle planes of a tool plate;
- bearing limits determined by the slide manufacturer in conjunction with the bearing supplier.
- **1.2** This Technical Report also describes how to calculate tool plate deflections due to the loads.
- **1.3** This Technical Report describes how to present the rating information in technical documentation for application by a user. **Teh STANDARD PREVIEW**
- **1.4** This Technical Report assumes that all of the applied loads and torque will be absorbed by the guide rods and not by the piston rod. Only the axial thrust load (but not the resulting moments) will be absorbed by the piston rod.

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#### 2 Normative references

The following referenced documents are indispensable for the application of this document. For dated references, only the edition cited applies. For undated references, the latest edition of the referenced document (including any amendments) applies.

ISO 5598:1985, Fluid power systems and components — Vocabulary

#### 3 Terms and definitions

For the purposes of this document, the terms and definitions given in ISO 5598 and the following apply.

#### 3.1

#### pneumatic slide

#### slide

mechanism containing a movable loading plate with guide rods, operated by an air cylinder

#### 3.2

#### quide rod

shaft, passing through a set of bearings, which controls the deflection and twist of the loading plate

#### 3.3

#### loading plate

plate onto which is placed a load to be moved

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#### 3.4

#### tool plate

loading plate attached at the end of the piston rod and guide rods

#### 3.5

#### carriage plate

loading plate attached in the middle of the slide, containing the guide rod bearings

NOTE In this design, mounting plates are attached at both ends of the guide rods for mounting the slide, allowing the carriage to move.

#### 3.6

#### housing

portion of the slide containing the bearings, when there is no carriage plate, and used for mounting the slide

#### 4 Rating factors

#### 4.1 Pressure containing capability

The manufacturer shall determine the maximum pressure that the pressure containing envelope is capable of sustaining if there is no load attached.

#### 4.2 Maximum axial load

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The manufacturer shall determine the maximum load for both push and pull directions, when the load reactions pass through the centre of the piston rod. Describe the limitations for any column buckling.

### 4.3 Maximum combined loading for a tool plate, and its deflections

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The manufacturer shall determine the following coefficients: 16806-2003

$$A = 2l_1 / f$$

$$B = 2l_1 / f (l_1 + l_2)$$

$$C = l_2 (l_1 + l_2) / (2l_1 + 3l_2)$$

$$D = (3l_2^2 + l_1 l_2 - l_1^2) / (2l_1 + 3l_2)$$

$$H = 12EI / l_2 (2l_1 + 3l_2)$$

$$W = w (l_1 + l_2)$$

#### where

- $l_1$  is the distance between the two bearing centrelines on one guide rod (this may vary with stroke); if there is only one bearing on a guide rod, then  $l_1$  is the length of the bearing;
- is the distance from the outer edge of the tool plate to the centreline of the closest bearing (this may vary with stroke);
- f is the scaling factor chosen by the manufacturer to bring calculated numbers into convenient size for tabulation;
- *E* is the modulus of elasticity for the guide rods;

*I* is the plane moment of inertia for two guide rods;

$$I = \pi \, (d_{\rm G}^{4}) \, / \, 32$$

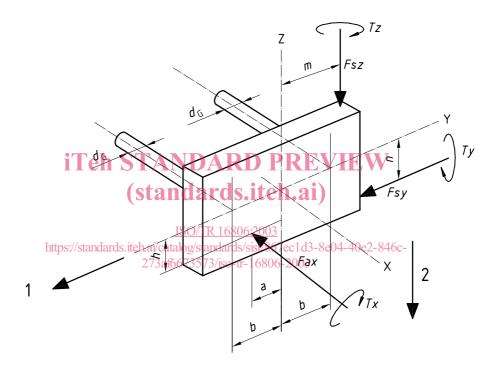
 $d_{\mathsf{G}}$  is the diameter of guide rod;

w is the weight of guide rod per unit of length.

### 5 Presentation of ratings

### 5.1 Sketch of loading on tool plate

See Figure 1.



#### Key

- 1 inline deflections
- 2 parallel deflections

Figure 1 — Tool plate identifications

### 5.2 Tabulations

Coefficient A	
---------------	--

	BORE SIZE			
STROKE				

Coefficient B	Co	effi	cie	nt	В
---------------	----	------	-----	----	---

STROKE	BORE	SIZE	
OTRORL			
			-

Coefficient C

	Сი	effi	cie	nt	D
,	-	~	CIC		$\boldsymbol{\nu}$

STROKE	BORE SIZE		STROKE	BORE SIZE			
OTRORE	iTeh ST	AND	RD P	REV			
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Coefficient  $\it W$ 

Co	effic	۱ <u>e</u> n	$\mathbf{t} H$
$\sim$	CIIIC	, 101	1 4 4 4

STROKE	BORE SIZE				
STROKE					

STROKE		BORE	SIZE	
STROKE				

#### 5.3 Graph

See Figure 2.

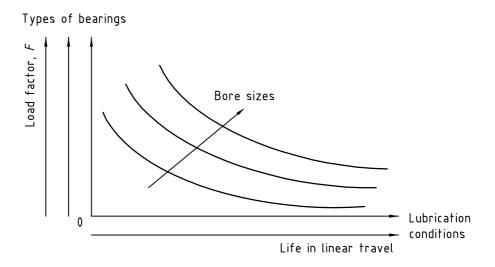


Figure 2 — Graph

#### 5.4 Calculation formulas

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5.4.1 Maximum tool plate load capacity

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$$F^{2} = \{(Fsy + W)/B + [(a)Fax + Tz]/A\}^{2} + \{[Fsz (1 + m/b) + W]/B + [(h)Fax + Ty]/A + [Tx + (n)Fsy]/bB\}^{2}$$

where

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A, B, W are coefficients determined in 4.3 and tabulated in 5.2;

a, b, h, m, n are dimensions on the tool plate as shown in Figure 1;

Fax, Fsy, Fsz, Tx, Ty, Tz are the applied forces and moments as shown in Figure 1.

The above formula describes the maximum combined loads that can be carried by the tool plate. If some of the loads do not exist in an application then it is possible to increase the other loads.

 $F = fR_A$ , the load factor presented in Figure 2

where

f is the arbitrary scaling factor described in 4.3;

 $R_{\rm A}$  is the bearing capacity which the slide manufacturer establishes, in conjunction with a bearing supplier, taking into account the bearing design, materials, its life rating, and lubrication conditions. These are then reflected in Figure 2.

#### 5.4.2 Linear deflections of the tool plate

— For inline deflections:

$$\delta = [4(C)Fsy + 2(a)Fax + 2Tz + W(D)]/H$$