
**Glass-reinforced thermosetting plastics
(GRP) pipes — Determination of the long-
term ultimate bending strain and the
long-term ultimate relative ring deflection
under wet conditions**

iTeh STANDARD PREVIEW
 *Tubes en plastiques thermodurcissables renforcés de verre (PRV) —
Détermination de l'effort à la flexion ultime à long terme et réflexion
annulaire relative ultime à long terme dans des conditions mouillées*
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Contents

Page

| | |
|---|----|
| Foreword..... | iv |
| 1 Scope..... | 1 |
| 2 Normative references | 1 |
| 3 Terms and definitions..... | 1 |
| 4 Principle | 3 |
| 5 Apparatus..... | 4 |
| 5.1 Compressive loading machine | 4 |
| 5.2 Force application surfaces..... | 4 |
| 5.3 Water container | 5 |
| 5.4 Measuring devices | 5 |
| 6 Test piece..... | 6 |
| 7 Number of test pieces..... | 6 |
| 8 Determination of the dimensions of the test piece | 7 |
| 8.1 Length | 7 |
| 8.2 Wall thickness..... | 7 |
| 8.3 Mean diameter..... | 7 |
| 9 Conditioning | 7 |
| 10 Procedure..... | 7 |
| 11 Calculation..... | 8 |
| 11.1 Extrapolation of the strain data to obtain the x -year value, $\epsilon_{x, \text{wet}}$ | 8 |
| 11.2 Calculation of the long-term ultimate relative ring deflection under wet conditions, $y_{u, \text{wet}}, x^1 d_m$ | 8 |
| 12 Test report..... | 8 |
| Annex A (informative) Equal increments of $\lg(\text{time in h})$ | 10 |

Foreword

ISO (the International Organization for Standardization) is a worldwide federation of national standards bodies (ISO member bodies). The work of preparing International Standards is normally carried out through ISO technical committees. Each member body interested in a subject for which a technical committee has been established has the right to be represented on that committee. International organizations, governmental and non-governmental, in liaison with ISO, also take part in the work. ISO collaborates closely with the International Electrotechnical Commission (IEC) on all matters of electrotechnical standardization.

International Standards are drafted in accordance with the rules given in the ISO/IEC Directives, Part 2.

The main task of technical committees is to prepare International Standards. Draft International Standards adopted by the technical committees are circulated to the member bodies for voting. Publication as an International Standard requires approval by at least 75 % of the member bodies casting a vote.

Attention is drawn to the possibility that some of the elements of this document may be the subject of patent rights. ISO shall not be held responsible for identifying any or all such patent rights.

ISO 10471 was prepared by Technical Committee ISO/TC 138, *Plastics pipes, fittings and valves for the transport of fluids*, Subcommittee SC 6, *Reinforced plastics pipes and fittings for all applications*.

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Glass-reinforced thermosetting plastics (GRP) pipes — Determination of the long-term ultimate bending strain and the long-term ultimate relative ring deflection under wet conditions

1 Scope

This International Standard specifies a method for determining by extrapolation the long-term ultimate bending strain and the calculation of the long-term ultimate relative ring deflection of glass-reinforced thermosetting plastics (GRP) pipes, under wet conditions.

Two methods of loading are given, one using plates the other beam bars.

NOTE Either method may be used for measurements of relative vertical deflection up to 28 %. When it is expected that this level is going to be exceeded then the procedure is limited to the use of at least one beam bar.

2 Normative references

The following referenced documents are indispensable for the application of this document. For dated references, only the edition cited applies. For undated references, the latest edition of the referenced document (including any amendments) applies.

ISO 7685, *Plastics piping systems — Glass-reinforced thermosetting plastics (GRP) pipes — Determination of initial specific ring stiffness*

ISO 10471:2003

ISO 10928:1997, *Plastics piping systems — Glass-reinforced thermosetting plastics (GRP) pipes and fittings — Methods for regression analysis and their use*

3 Terms and definitions

For the purposes of this document, the following terms and definitions apply.

3.1

vertical compressive force

F

vertical force, expressed in newtons, applied to a horizontal pipe to cause a vertical deflection

3.2

mean diameter

d_m

diameter, expressed in metres, of the circle corresponding with the middle of the pipe wall cross-section and given by either of the following equations:

$$d_m = d_i + e \quad (1)$$

$$d_m = d_e - e \quad (2)$$

where

d_i is the internal diameter, in metres;

d_e is the external diameter, in metres;

e is the wall thickness of the pipe, in metres.

3.3
vertical deflection

y
vertical change in diameter of a pipe in a horizontal position, expressed in metres, in response to a vertical compressive force (see 3.1)

3.4
relative vertical deflection

y/d_m
ratio of the vertical deflection, y (see 3.3), to the mean diameter of the pipe, d_m (see 3.2)

3.5
ultimate vertical deflection under wet conditions

$y_{u, wet}$
vertical deflection of the pipe, y (see 3.3), expressed in metres, when failure occurs under wet conditions (see Clause 4)

3.6
ultimate relative vertical deflection under wet conditions

$y_{u, wet}/d_m$
ratio of the ultimate vertical deflection under wet conditions, $y_{u, wet}$ (see 3.5), to the mean diameter of the pipe, d_m (see 3.2)

3.7
long-term ultimate ring deflection under wet conditions

$y_{u, wet, x}$
extrapolated value of the ultimate vertical deflection, expressed in metres, of the pipe under wet conditions, $y_{u, wet}$ (see 3.5), when failure is expected to occur at a time, x , specified in the referring standard

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3.8
long-term ultimate relative ring deflection under wet conditions

$y_{u, wet, x}/d_m$
ratio of the long-term ultimate ring deflection under wet conditions, $y_{u, wet, x}$ (see 3.7), to the mean diameter of the pipe, d_m (see 3.2)

3.9
failure

loss of the structural integrity of a test piece as evidenced by the inability of the test piece to carry the load

3.10
time to failure

t_u
time elapsed, expressed in hours, until failure (see 3.9) occurs

3.11
specific ring stiffness

S
physical characteristic of a pipe, expressed in newtons per square metre, that is a measure of the resistance to ring deflection per metre length under external load and is defined by Equation (3):

$$S = \frac{E \times I}{d_m^3} \quad (3)$$

where

E is the apparent modulus of elasticity, in newtons per square metre, determined by testing in accordance with ISO 7685;

I is the second moment of area in the longitudinal direction per metre length, in metres to the fourth power per metre (m^4/m), i.e.

$$I = \frac{e^3}{12} \quad (4)$$

e being the wall thickness of the pipe, in metres;

d_m is the mean diameter of the pipe, in metres (see 3.2)

3.12

initial specific ring stiffness

S_0

value of S , expressed in newtons per square metre, determined by testing in accordance with ISO 7685

3.13

strain factor

D_g

dimensionless factor used to transform a deflection value into a strain value

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4 Principle

Each of several cut lengths of pipe is supported horizontally and loaded throughout its length to compress it diametrically to achieve a desired level of strain. The force application surfaces are either bearing plates or beam bars.

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The pipe is immersed in water at a given temperature for a period of time during which the force remains constant and the increasing vertical deflection is measured at intervals until failure (see 3.9) occurs. The relative vertical deflection at failure [ultimate relative vertical deflection, $y_{u, \text{wet}}/d_m$ (see 3.6)] is converted into a bending strain at failure (ultimate bending strain, $\varepsilon_{u, \text{wet}}$, in percent), either calculated using Equation (5) or determined from a strain-deflection calibration curve (see 10.3).

NOTE The strain may also be measured directly by the use of waterproofed strain gauges.

$$\varepsilon_{u, \text{wet}} = D_g \times \frac{e}{d_m} \times \frac{y_{u, \text{wet}}}{d_m} \times 100 \quad (5)$$

where

D_g is calculated using Equation (6):

$$D_g = \frac{4,28}{\left(1 + \frac{y_{u, \text{wet}}}{2 \times d_m}\right)^2} \quad (6)$$

$y_{u, \text{wet}}$ being the ultimate vertical deflection under wet conditions, in metres;

d_m is the mean diameter of the pipe (see 3.2), in metres;

e is the mean wall thickness of the pipe, obtained from several measurements around the pipe circumference, in metres.

These values of ultimate bending strain and the applicable times, t_u (see 3.10), are used in the procedures described in method A of ISO 10928:1997 to determine the long-term ultimate bending strain under wet conditions, $\varepsilon_{u, \text{wet}, x}$. If it is required to determine the long-term ultimate relative ring deflection under wet conditions, $y_{u, \text{wet}, x}/d_m$ (see 3.8), expressed in percent, then the long-term ultimate bending strain is converted to deflection using Equation (7):

$$\frac{y_{u, \text{wet}, x}}{d_m} = \frac{\varepsilon_{u, \text{wet}, x}}{D_g \times \frac{e}{d_m}} \quad (7)$$

NOTE It is assumed that values for the following test parameters will be set by the standard making reference to this International Standard:

- a) the time, x , to which the values are to be extrapolated (see 3.7 and 3.8);
- b) the test temperature (see 5.3 and 10.1);
- c) the length and number of test pieces (see Clauses 6 and 7);
- d) the distribution of the times to failure (see 10.8);
- e) the pH of the test water.

The standard test procedure is to determine and analyse the bending strains induced by the vertical deflections. This procedure accommodates sample to sample variation and results in a strain versus time expression applicable to a range of pipe classifications.

5 Apparatus

5.1 Compressive loading machine

The machine shall comprise a system capable of applying a force, without shock, through two parallel force application surfaces conforming to 5.2 so that a horizontally orientated test piece of pipe conforming to Clause 6 and immersed in water can be compressed vertically and maintained under a constant force for the duration of the test in accordance with 10.7.

Equipment shall be available for determining the force applied to within ± 1 % of the total force.

Ensure that the applied force is not affected by buoyancy effects or friction.

NOTE For test pieces subjected to high predetermined forces, for which failure is expected to occur within 100 h, an automatic recording device will help in recording failure times and deflections accurately.

5.2 Force application surfaces

NOTE The method allows the use of either bearing plates or beam bars for loading the test piece, subject to reporting the choice used. Either method may be used for measurements of relative vertical deflection up to 28 %. When it is expected that 28 % deflection is going to be exceeded then the procedure is limited to the use of at least one beam bar.

5.2.1 General arrangement

The surfaces shall be provided by a pair of bearing plates, conforming to 5.2.2, or a pair of beam bars, conforming to 5.2.3, or a combination of one such plate and one such bar. However, if the applied force is expected to cause a relative deflection in excess of 28 %, at least one of the surfaces shall be a beam bar. The surface's major axes shall be perpendicular to and centred on the direction of application of force F by the compressive loading machine, as shown in Figure 1. The surfaces to be in contact with the test piece shall be flat, smooth, clean and parallel.

5.2.2 Plates

The plate(s) shall have a width of at least 100 mm and a length at least equal to the length of the test piece (see Clause 6). They shall be sufficiently stiff so that they do not visibly bend or otherwise deform during the test.

5.2.3 Beam bars

Each beam bar shall be sufficiently stiff that it does not visibly bend or otherwise deform during the test. Each beam bar shall have a length at least equal to the length of the test piece (see Clause 6) and a flat face (see Figure 1) without sharp edges. The width of the flat face shall be 15 mm to 55 mm.

The beam bars shall be so constructed and supported that no other surface of the beam bar structure comes into contact with the test piece during the test.

5.3 Water container

Required is a container large enough to accommodate submerged test pieces conforming to Clause 6 whilst they are subject to the compressive force in accordance with 10.6 and containing test water maintained at the specified temperature (see Clause 4).

The water level shall be maintained sufficiently constant to avoid any significant effect on the value of the vertical force applied to the test piece.

5.4 Measuring devices

Required are devices capable of determining:

- a) the necessary dimensions (length, diameters, wall thickness) to an accuracy within $\pm 1,0$ mm;
- b) the deflection of the test piece in the vertical direction during the test to an accuracy within $\pm 1,0$ % of the maximum value.

NOTE When selecting the device to measure the change in diameter of the test piece, consideration should be given to the potentially corrosive environment in which the device is to be used.