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**Metallic materials — Fatigue testing —  
Fatigue crack growth method**

*Matériaux métalliques — Essais de fatigue — Méthode d'essai de  
propagation de fissure en fatigue*

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Tel. + 41 22 749 01 11  
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## Foreword

ISO (the International Organization for Standardization) is a worldwide federation of national standards bodies (ISO member bodies). The work of preparing International Standards is normally carried out through ISO technical committees. Each member body interested in a subject for which a technical committee has been established has the right to be represented on that committee. International organizations, governmental and non-governmental, in liaison with ISO, also take part in the work. ISO collaborates closely with the International Electrotechnical Commission (IEC) on all matters of electrotechnical standardization.

International Standards are drafted in accordance with the rules given in the ISO/IEC Directives, Part 3.

Draft International Standards adopted by the technical committees are circulated to the member bodies for voting. Publication as an International Standard requires approval by at least 75 % of the member bodies casting a vote.

Attention is drawn to the possibility that some of the elements of this International Standard may be the subject of patent rights. ISO shall not be held responsible for identifying any or all such patent rights.

International Standard ISO 12108 was prepared by Technical Committee ISO/TC 164, *Mechanical testing of metals*, Subcommittee SC 5, *Fatigue testing*.

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## Introduction

This International Standard is intended to provide specifications for generation of fatigue crack growth rate data. Test results are expressed in terms of the fatigue crack growth rate as a function of crack-tip stress intensity factor range,  $\Delta K$ , as defined by the theory of linear elastic fracture mechanics [1]-[6]. Expressed in these terms the results characterize a material's resistance to subcritical crack extension under cyclic force test conditions. This resistance is independent of specimen planar geometry and thickness, within the limitations specified in clause 6. All values are given in SI units [7].

This International Standard describes a method of subjecting a precracked notched specimen to a cyclic force. The crack length,  $a$ , is measured as a function of the number of elapsed force cycles,  $N$ . From the collected crack length and corresponding force cycles relationship the fatigue crack growth rate,  $da/dN$ , is determined and is expressed as a function of stress intensity factor range,  $\Delta K$ .

Materials that can be tested by this method are limited by size, thickness and strength only to the extent that the material must remain predominantly in an elastic condition during testing and that buckling is precluded.

Specimen size may vary over a wide range. Proportional planar dimensions for six standard configurations are presented. The choice of a particular specimen configuration may be dictated by the actual component geometry, compression test conditions or suitability for a particular test environment.

Specimen size is a variable that is subjective to the test material's 0,2 % proof strength and the maximum stress intensity factor applied during test. Specimen thickness may vary independent of the planar size, within defined limits, so long as large-scale yielding is precluded and out-of-plane distortion or buckling is not encountered. Any alternate specimen configuration other than those included in this International Standard may be used, provided there exists an established stress intensity factor calibration expression, i.e. stress intensity factor function,  $g(a/W)$  [9]-[11].

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Residual stresses [12], [13], crack closure [14], [15], specimen thickness, cyclic waveform, frequency and environment, including temperature, may markedly affect the fatigue crack growth data but are in no way reflected in the computation of  $\Delta K$ , and so should be recognized in the interpretation of the test results and be included as part of the test report. All other demarcations from this method should be noted as exceptions to this practice in the final report.

For crack growth rates above  $10^{-5}$  mm/cycle the typical scatter in test results generated in a single laboratory for a given  $\Delta K$  can be in the order of a factor of two [16]. For crack growth rates below  $10^{-5}$  mm/cycle, the scatter in the  $da/dN$  calculation may increase to a factor of 5 or more. To assure the correct description of the material's  $da/dN$  versus  $\Delta K$  behaviour, a replicate test conducted with the same test parameters is highly recommended.

Service conditions may exist where varying  $\Delta K$  under conditions of constant  $K_{\max}$  or  $K_{\text{mean}}$  control [17] may be more representative than data generated under conditions of constant force ratio; however, these alternate test procedures are beyond the scope of this International Standard.

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# Metallic materials — Fatigue testing — Fatigue crack growth method

## 1 Scope

This International Standard describes tests for determining the fatigue crack growth rate from the threshold stress-intensity factor range,  $\Delta K_{th}$ , to the onset of unstable crack extension as the maximum stress intensity factor approaches  $K_{max}$  controlled instability, as determined in accordance with ISO 12737 [8].

This International Standard is primarily intended for use in evaluating isotropic metallic materials under predominantly linear-elastic stress conditions and with force applied only perpendicular to the crack plane (mode I stress condition), and with a constant stress ratio,  $R$ .

## 2 Normative reference

The following normative document contains provisions which, through reference in this text, constitute provisions of this International Standard. For dated references, subsequent amendments to, or revisions of, this publication do not apply. However, parties to agreements based on this International Standard are encouraged to investigate the possibility of applying the most recent edition of the normative document indicated below. For undated references, the latest edition of the normative document referred to applies. Members of ISO and IEC maintain registers of currently valid International Standards.

ISO 12108:2002

ISO 4965:1979, Axial load fatigue testing machines — Dynamic force calibration — Strain gauge technique

## 3 Terms and definitions

For the purposes of this International Standard, the following terms and definitions apply.

### 3.1

#### crack length

$a$

linear measure of a principal planar dimension of a crack from a reference plane to the crack tip, also called crack size

### 3.2

#### cycle

$N$

smallest segment of a force-time or stress-time function which is repeated periodically

NOTE The terms fatigue cycle, force cycle and stress cycle are used interchangeably. The letter  $N$  is used to represent the number of elapsed force cycles.

### 3.3

#### fatigue crack growth rate

$da/dN$

extension in crack length per force cycle

### 3.4

#### maximum force

$F_{\max}$

force having the highest algebraic value in the cycle; a tensile force being positive and a compressive force being negative

### 3.5

#### minimum force

$F_{\min}$

force having the lowest algebraic value in the cycle; a tensile force being positive and a compressive force being negative

### 3.6

#### force range

$\Delta F$

the algebraic difference between the maximum and minimum forces in a cycle

$$\Delta F = F_{\max} - F_{\min}$$

### 3.7

#### force ratio

$R$

algebraic ratio of the minimum force or stress to maximum force or stress in a cycle

$$R = F_{\min} / F_{\max}$$

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NOTE It is also called stress ratio.

### 3.8

#### stress intensity factor

$K$

magnitude of the ideal crack tip stress field for the opening mode force application to a crack in a homogeneous, linear-elastically stressed body where opening mode of a crack corresponds to the force being applied to the body perpendicular to the crack faces only (mode I stress condition)

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### 3.9

#### maximum stress intensity factor

$K_{\max}$

highest algebraic value of the stress intensity factor in a cycle, corresponding to  $F_{\max}$

### 3.10

#### minimum stress intensity factor

$K_{\min}$

lowest algebraic value of the stress intensity factor in a cycle, corresponding to  $F_{\min}$

NOTE This definition remains the same, regardless of the minimum force being tensile or compressive. For a negative force ratio ( $R < 0$ ) there is an alternate, commonly used definition for the minimum stress intensity factor,  $K_{\min} = 0$ . See 3.11.

### 3.11

#### stress intensity factor range

$\Delta K$

algebraic difference between the maximum and minimum stress intensity factors in a cycle

$$\Delta K = K_{\max} - K_{\min}$$

NOTE The force variables  $\Delta K$ ,  $R$  and  $K_{\max}$  are related as follows:  $\Delta K = (1 - R) K_{\max}$ . For a negative force ratio ( $R < 0$ ) there is an alternate, commonly used definition for the stress intensity factor range,  $\Delta K = K_{\max}$ . See 3.10 and 10.6.

**3.12****fatigue crack growth threshold** $\Delta K_{th}$ asymptotic value of  $\Delta K$  for which  $da/dN$  approaches zero

NOTE For most materials the threshold is defined as the stress intensity factor range corresponding to  $10^{-8}$  mm/cycle. When reporting  $\Delta K_{th}$ , the corresponding lowest decade of  $da/dN$  data used in its determination should also be included.

**3.13****normalized  $K$ -gradient** $C = (1/K) dK/da$ fractional rate of change of  $K$  with increased crack length,  $a$ 

$$C = 1/K (dK/da) = 1/K_{max} (dK_{max}/da) = 1/K_{min} (dK_{min}/da) = 1/\Delta K (d\Delta K/da)$$

**3.14** **$K$ -decreasing test**test in which the value of the normalized  $K$ -gradient,  $C$ , is negative

NOTE A  $K$ -decreasing test is conducted by reducing the stress intensity factor either by continuously shedding or by a series of steps, as the crack grows.

**3.15** **$K$ -increasing test**test in which the value of  $C$  is positive

NOTE For standard specimens, a constant force amplitude results in a  $K$  increasing test where the value of  $C$  is positive and increasing.

**3.16****stress intensity factor function** $g(a/W)$ 

mathematical expression, based on experimental, numerical or analytical results, that relates the stress intensity factor to force and crack length for a specific specimen configuration

**4 Symbols and abbreviations****4.1 Symbols**

See Table 1.

**Table 1 — Symbols and their designations**

Symbol	Designation	Unit
<b>Loading</b>		
$C$	Normalized $K$ -gradient	$\text{mm}^{-1}$
$E$	Tensile modulus of elasticity	MPa
$F$	Force	kN
$F_{max}$	Maximum force	kN
$F_{min}$	Minimum force	kN
$\Delta F$	Force range	kN
$K$	Stress intensity factor	$\text{MPa}\cdot\text{m}^{1/2}$
$K_{max}$	Maximum stress intensity factor	$\text{MPa}\cdot\text{m}^{1/2}$
$K_{min}$	Minimum stress intensity factor	$\text{MPa}\cdot\text{m}^{1/2}$
$\Delta K$	Stress intensity factor range	$\text{MPa}\cdot\text{m}^{1/2}$
$\Delta K_i$	Initial stress intensity factor range	$\text{MPa}\cdot\text{m}^{1/2}$

Table 1 — Symbols and their designations (*continued*)

Symbol	Designation	Unit
$\Delta K_{th}$	Threshold stress intensity factor range	MPa·m <sup>1/2</sup>
$N$	Number of cycles	1
$R$	Force ratio or stress ratio	1
$R_m$	Ultimate tensile strength at the test temperature	MPa
$R_{p0,2}$	0,2 % proof strength at the test temperature	MPa
<b>Geometry</b>		
$a$	Crack length or size measured from the reference plane to the crack tip	mm
$a_{cor}$	Crack front curvature correction length	mm
$a_{fat}$	Fatigue crack length measured from the notch root	mm
$a_n$	Machined notch length	mm
$a_p$	Pre-crack length	mm
$B$	Specimen thickness	mm
$D$	Hole diameter for CT, SENT or CCT specimen, loading tup diameter for bend specimens	mm
$g(a/W)$	Stress intensity factor function	1
$h$	Notch height	mm
$W$	Specimen width, distance from reference plane to edge of specimen	mm
$(W - a)$	minimum uncracked ligament	mm
<b>Crack growth</b>		
$da/dN$	Fatigue crack growth rate	mm/cycle
$\Delta a$	Change in crack length, crack extension	mm

## 4.2 Abbreviations for specimen identification

CT Compact tension

CCT Centre cracked tension

SENT Single edge notch tension

SENB3 Three-point single edge notch bend

SENB4 Four-point single edge notch bend

SENB8 Eight-point single edge notch bend

## 5 Apparatus

### 5.1 Testing machine

#### 5.1.1 General

The testing machine shall have smooth start-up and a backlash-free force train if passing through zero force. See ISO 4965. Cycle to cycle variation of the peak force during precracking shall be less than  $\pm 5\%$  and shall be held to within  $\pm 2\%$  of the desired peak force during the test.  $\Delta F$  shall also be maintained to within  $\pm 2\%$  of the desired range during test. A practical overview of test machines and instrumentation is available [33], [34].

### 5.1.2 Testing machine alignment

It is important that adequate attention be given to alignment of the testing machine and during machining and installation of the grips in the testing machine.

For tension-compression testing, the length of the force train should be as short and stiff as practical. Non-rotating joints should be used to minimize off-axis motion.

Asymmetry of the crack front is an indication of misalignment; a strain gauged specimen similar to the test article under investigation can be used in aligning the force train and to minimize nonsymmetrical stress distribution and/or bending strain to less than 5 %.

### 5.1.3 Force measuring system

Accuracy of the force measuring system shall be verified periodically in the testing machine. The calibration for the force transducer shall be traceable to a national organization of metrology. The force measuring system shall be designed for tension and compression fatigue testing and possess great axial and lateral rigidity. The indicated force, as recorded as the output from the computer in an automated system or from the final output recording device in a noncomputer system, shall be within the permissible variation from the actual force. The force transducer's capacity shall be sufficient to cover the range of force measured during a test. Errors greater than 1 % of the difference between minimum and maximum measured test force are not acceptable.

The force measuring system shall be temperature compensated, not have zero drift greater than 0,002 % of full scale, nor have a sensitivity variation greater than 0,002 % of full scale over a 1 °C change. During elevated and cryogenic temperature testing, suitable thermal shielding/compensation shall be provided to the force measuring system so it is maintained within its compensation range.

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## 5.2 Cycle-counter

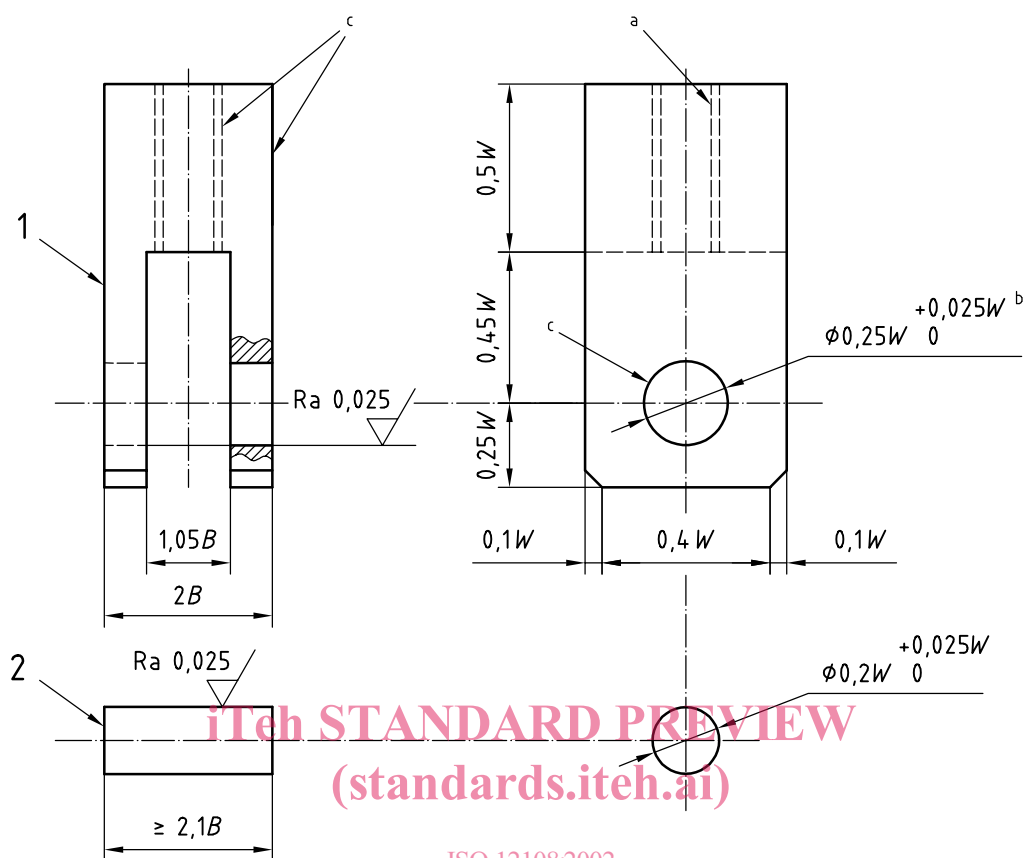
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An accurate digital device is required to count elapsed force cycles. A timer is to be used only as a verification check on the accuracy of the counter. It is preferred that individual force cycles be counted. However, when the crack velocity is below  $10^{-5}$  mm/cycle counting in increments of ten cycles is acceptable.

## 5.3 Grips and fixtures for CT specimens

Force is applied to a CT specimen through pinned joints. Choice of this specimen and gripping arrangement necessitates tension-tension test conditions only. Figure 1 shows the clevis and mating pin assembly used at both the top and bottom of a CT specimen to apply the force perpendicular to the machined starter notch and crack plane. Suggested dimensions are expressed as a proportion of specimen width,  $W$ , or thickness,  $B$ , since these dimensions can vary independently within the limits specified in clause 6. The pin holes have a generous clearance over the pin diameter,  $0,2W$  minimum, to minimize resistance to specimen and pin in-plane rotation which has been shown to cause nonlinearity in the force versus displacement response <sup>[35]</sup>. A surface finish range of 0,8 µm to 1,6 µm is suggested for grip surfaces. With this grip and pin arrangement, materials with low proof strength may sustain plastic deformation at the specimen pin hole; similarly, when testing high strength materials and/or when the clevis displacement exceeds  $1,05B$ , a stiffer force pin, i.e. a diameter greater than  $0,225W$ , may be required. As an alternative approach to circumvent plastic deformation, a flat bottom clevis hole may be used along with a pin diameter equalling  $0,24W$ . Any heat treatable steel thermally processed to a 0,2 % proof strength of 1 000 MPa used in fabricating the clevises will usually provide adequate strength and resistance to fretting, galling and fatigue.

In addition to the generous pin hole clearance, the mating surfaces shall be prepared to minimize friction which could invalidate the provided  $K$ -calibration expression. The use of high viscosity lubricants and greases has been shown to cause hysteresis in the force versus displacement response and is not recommended if compliance measurements are required.

**Key**

- 1 Loading rod  
2 Pin

NOTE For high strength materials or large pin displacements the pin may be stiffened by increasing the diameter to  $0,24W$  along with using D-shaped flat bottom holes.

<sup>a</sup> Loading rod thread.

<sup>b</sup> Through diameter.

<sup>c</sup> These surfaces are perpendicular and parallel as applicable to within  $0,05W$ .

**Figure 1 — Clevis and pin assembly for gripping a CT specimen**

## 5.4 Grips and fixtures for CCT/SENT specimens

### 5.4.1 General

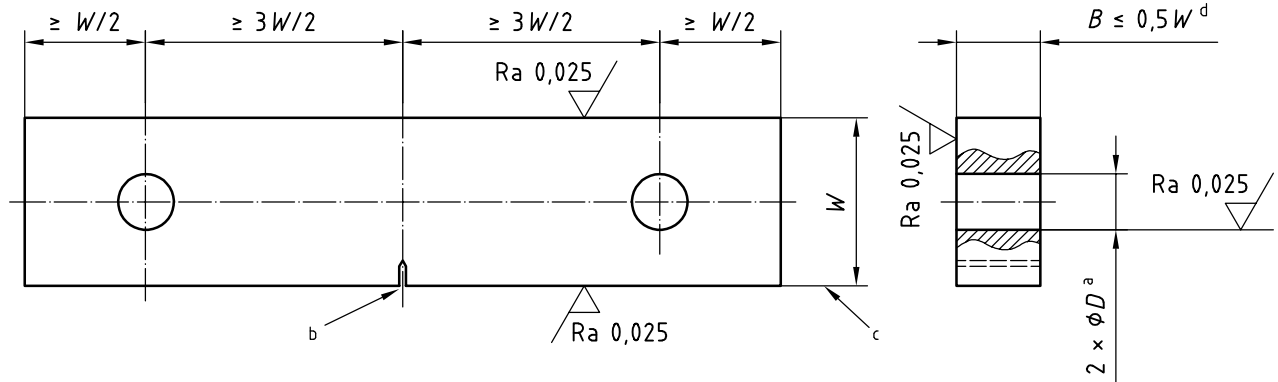
Force can be applied to CCT and SENT specimens through pinned joints and/or through frictional clamping grips. Gripping for the CCT and SENT specimens depends on specimen width and whether the test condition is to be tension-tension or tension-compression. The minimum CCT specimen gauge length varies with gripping arrangement and shall provide a uniform stress distribution in the gauge length during the test.

Equation (6) is applicable only for a single pinned end SENT specimen, as shown in Figure 2. The SENT pinned end specimen (Figure 2) is appropriate for tension-tension test conditions only.

Equation (7) is applicable for a SENT specimen with clamped ends and is appropriate for both tension and compression force conditions. For the clamped end SENT specimen, the grips must be sufficiently stiff to circumvent

any rotation of the specimen ends or any lateral movement of the crack plane; the presence of either condition introduces errors into the stress intensity factor calculation <sup>[29]</sup>.

Surface roughness values in micrometres



NOTE 1 The machined notch is centred to within  $\pm 0,005W$ .

NOTE 2 The surfaces are parallel and perpendicular to within  $\pm 0,002W$ .

NOTE 3 The crack length is measured from the reference loading plane containing the starter V-notch.

NOTE 4 Specimen recommended for notch root tension at a force ratio  $R > 0$  only.

<sup>a</sup>  $D = W/3$ .

<sup>b</sup> See Figure 12 for notch detail.

<sup>c</sup> Reference plane.

<sup>d</sup> The recommended thickness.

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**Figure 2 — Standard single edge notch tension, SENT, specimen**

#### 5.4.2 Tension-tension testing of a CCT specimen

For tension-tension testing of a specimen with a width  $2W$ , less than 75 mm, as shown in Figure 3, a clevis with single force pin is acceptable for gripping provided the specimen gauge length, defined here as the distance between the pin hole centrelines, be at least  $6W$ . Shims may be helpful in circumventing fretting fatigue at the specimen's pin hole. Another step that can be taken to prevent crack initiation at the pin holes is the welding or adhesive bonding of reinforcement plates or tabs to the gripping area, especially when testing very thin materials. Cutting the test section down in width to form a "dog bone" shaped specimen design is another measure that can be adopted to circumvent failure at the pin holes; here the gauge length is defined as the uniform width section and it shall be at least  $3,4W$  in length.

For tension-tension testing of a specimen with a width greater than 75 mm, distributing the force across the specimen width with multiple pin holes is recommended. A serrated grip surface at the specimen-grip interface increases the force that can be transferred. With this force application arrangement the gauge length between the innermost rows of pin holes must be at least  $3W$ .