

---

---

**Mechanical vibration, shock and  
condition monitoring — Vocabulary**

*Vibrations et chocs mécaniques, et leur surveillance — Vocabulaire*

**iTeh STANDARD PREVIEW**  
**(standards.iteh.ai)**

[ISO 2041:2009](https://standards.iteh.ai/catalog/standards/sist/2f54bbfe-70bb-46da-8714-a72775b8ec8f/iso-2041-2009)

<https://standards.iteh.ai/catalog/standards/sist/2f54bbfe-70bb-46da-8714-a72775b8ec8f/iso-2041-2009>



**PDF disclaimer**

This PDF file may contain embedded typefaces. In accordance with Adobe's licensing policy, this file may be printed or viewed but shall not be edited unless the typefaces which are embedded are licensed to and installed on the computer performing the editing. In downloading this file, parties accept therein the responsibility of not infringing Adobe's licensing policy. The ISO Central Secretariat accepts no liability in this area.

Adobe is a trademark of Adobe Systems Incorporated.

Details of the software products used to create this PDF file can be found in the General Info relative to the file; the PDF-creation parameters were optimized for printing. Every care has been taken to ensure that the file is suitable for use by ISO member bodies. In the unlikely event that a problem relating to it is found, please inform the Central Secretariat at the address given below.

**iTeh STANDARD PREVIEW**  
**(standards.iteh.ai)**

ISO 2041:2009

<https://standards.iteh.ai/catalog/standards/sist/2f54bbfe-70bb-46da-8714-a72775b8ec8f/iso-2041-2009>



**COPYRIGHT PROTECTED DOCUMENT**

© ISO 2009

The reproduction of the terms and definitions contained in this International Standard is permitted in teaching manuals, instruction booklets, technical publications and journals for strictly educational or implementation purposes. The conditions for such reproduction are: that no modifications are made to the terms and definitions; that such reproduction is not permitted for dictionaries or similar publications offered for sale; and that this International Standard is referenced as the source document.

With the sole exceptions noted above, no other part of this publication may be reproduced or utilized in any form or by any means, electronic or mechanical, including photocopying and microfilm, without permission in writing from either ISO at the address below or ISO's member body in the country of the requester.

ISO copyright office  
Case postale 56 • CH-1211 Geneva 20  
Tel. + 41 22 749 01 11  
Fax + 41 22 749 09 47  
E-mail [copyright@iso.org](mailto:copyright@iso.org)  
Web [www.iso.org](http://www.iso.org)

Published in Switzerland

<b>Contents</b>	<b>Page</b>
<b>Foreword</b> .....	<b>iv</b>
<b>Introduction</b> .....	<b>v</b>
<b>Scope</b> .....	<b>1</b>
<b>1 General</b> .....	<b>1</b>
<b>2 Vibration</b> .....	<b>15</b>
<b>3 Mechanical shock</b> .....	<b>29</b>
<b>4 Transducers for shock and vibration measurement</b> .....	<b>31</b>
<b>5 Signal processing</b> .....	<b>34</b>
<b>6 Condition monitoring and diagnostics</b> .....	<b>40</b>
<b>Bibliography</b> .....	<b>43</b>
<b>Alphabetical index</b> .....	<b>44</b>

**iTeh STANDARD PREVIEW**  
**(standards.iteh.ai)**

[ISO 2041:2009](#)

<https://standards.iteh.ai/catalog/standards/sist/2f54bbfe-70bb-46da-8714-a72775b8ec8f/iso-2041-2009>

## Foreword

ISO (the International Organization for Standardization) is a worldwide federation of national standards bodies (ISO member bodies). The work of preparing International Standards is normally carried out through ISO technical committees. Each member body interested in a subject for which a technical committee has been established has the right to be represented on that committee. International organizations, governmental and non-governmental, in liaison with ISO, also take part in the work. ISO collaborates closely with the International Electrotechnical Commission (IEC) on all matters of electrotechnical standardization.

International Standards are drafted in accordance with the rules given in the ISO/IEC Directives, Part 2.

The main task of technical committees is to prepare International Standards. Draft International Standards adopted by the technical committees are circulated to the member bodies for voting. Publication as an International Standard requires approval by at least 75 % of the member bodies casting a vote.

Attention is drawn to the possibility that some of the elements of this document may be the subject of patent rights. ISO shall not be held responsible for identifying any or all such patent rights.

ISO 2041 was prepared by Technical Committee ISO/TC 108, *Mechanical vibration, shock and condition monitoring*.

This third edition cancels and replaces the second edition (ISO 2041:1990) which has been technically revised. This revision reflects advances in technology and refinements in terms used in the previous version. As such, it incorporates more precise definitions of some terms reflecting changes in accepted meaning. New terms which were driven by changes in technology (primarily in the areas of signal processing, condition monitoring and vibration and shock diagnostics and prognostics) and, in order to be a stand-alone standard, terms from ISO 2041:1990 still in common usage are incorporated.

ITEH STANDARD PREVIEW

(standards.iteh.ai)

ISO 2041:2009

<https://standards.iteh.ai/catalog/standards/sist/2f54bbfe-70bb-46da-8714-a72775b8ec8f/iso-2041-2009>

## Introduction

Vocabulary is the most basic of subjects for standardization. Without an accepted standard for the definition of terminology, the development of other technical standards in a technical area becomes a laborious and time-consuming task that would ultimately result in the inefficient use of time and a high probability of misinterpretation.

## iTeh STANDARD PREVIEW (standards.iteh.ai)

[ISO 2041:2009](https://standards.iteh.ai/catalog/standards/sist/2f54bbfe-70bb-46da-8714-a72775b8ec8f/iso-2041-2009)

<https://standards.iteh.ai/catalog/standards/sist/2f54bbfe-70bb-46da-8714-a72775b8ec8f/iso-2041-2009>

**iTeh STANDARD PREVIEW**  
**(standards.iteh.ai)**

ISO 2041:2009

<https://standards.iteh.ai/catalog/standards/sist/2f54bbfe-70bb-46da-8714-a72775b8ec8f/iso-2041-2009>

# Mechanical vibration, shock and condition monitoring — Vocabulary

## Scope

This International Standard defines terms and expressions unique to the areas of mechanical vibration, shock and condition monitoring.

## 1 General

### 1.1

#### **displacement** **relative displacement**

(vibration and shock) time varying quantity that specifies the change in position of a point on a body with respect to a reference frame

NOTE 1 The reference frame is usually a set of axes at a mean position or a position of rest. In general, a rotation displacement vector, a translation displacement vector, or both can represent the displacement.

NOTE 2 A displacement is designated as relative displacement if it is measured with respect to a reference frame other than the primary reference frame designated in a given case.

NOTE 3 Displacement can be:

- oscillatory, in which case simple harmonic components can be defined by the displacement amplitude (and frequency), or
- random, in which case the root-mean-square (rms) displacement (and band-width and probability density distribution) can be used to define the probability that the displacement will have values within any given range.

Displacements of short time duration are defined as transient displacements. Non-oscillatory displacements are defined as sustained displacements, if of long duration, or as displacement pulses, if of short duration.

### 1.2

#### **velocity** **relative velocity**

(vibration and shock) rate of change of displacement

NOTE 1 In general, velocity is time-dependent.

NOTE 2 The reference frame is usually a set of axes at a mean position or a position of rest. In general, a rotation velocity vector, a translation velocity vector, or both can represent the velocity.

NOTE 3 A velocity is designated as relative velocity if it is measured with respect to a reference frame other than the primary reference frame designated in a given case. The relative velocity between two points is the vector difference between the velocities of the two points.

NOTE 4 Velocity can be:

- oscillatory, in which case simple harmonic components can be defined by the velocity amplitude (and frequency), or
- random, in which case the root-mean-square (rms) velocity (and band-width and probability density distribution) can be used to define the probability that the velocity will have values within any given range.

Velocities of short time duration are defined as transient velocities. Non-oscillatory velocities are defined as sustained velocities, if of long duration.

**1.3**  
**acceleration**  
**relative acceleration**

(vibration and shock) rate of change of velocity

NOTE 1 In general, acceleration is time-dependent.

NOTE 2 The reference frame is usually a set of axes at a mean position or a position of rest. In general, a rotation acceleration vector, a translation acceleration vector, or both and the Coriolis acceleration can represent the acceleration.

NOTE 3 An acceleration is designated as relative acceleration if it is measured with respect to a reference frame other than the inertial reference frame designated in a given case. The relative acceleration between two points is the vector difference between the accelerations of the two points.

NOTE 4 In the case of time-dependent accelerations, various self-explanatory modifiers, such as peak, average, and rms (root-mean-square), are often used. The time intervals over which the average or root-mean-square values are taken should be indicated or implied.

NOTE 5 Acceleration can be:

- oscillatory, in which case simple harmonic components can be defined by the acceleration amplitude (and frequency), or
- random, in which case the rms acceleration (and band-width and probability density distribution) can be used to define the probability that the acceleration will have values within any given range.

Accelerations of short time duration are defined as transient accelerations. Non-oscillatory accelerations are defined as sustained accelerations, if of long duration, or as acceleration pulses, if of short duration.

**1.4**  
**standard acceleration due to gravity**

$g_n$   
unit, 9,806 65 metres per second-squared (9,806 65 m/s<sup>2</sup>)

NOTE 1 Value adopted in the International Service of Weights and Measures and confirmed in 1913 by the 5th CGPM as the standard for acceleration due to gravity.

NOTE 2 This "standard value" ( $g_n = 9,806\ 65\ \text{m/s}^2 = 980,665\ \text{cm/s}^2 \approx 386,089\ \text{in/s}^2 \approx 32,174\ 0\ \text{ft/s}^2$ ) should be used for reduction to standard gravity of measurements made in any location on Earth.

NOTE 3 Frequently, the magnitude of acceleration is expressed in units of  $g_n$ .

NOTE 4 The actual acceleration produced by the force of gravity at or below the surface of the Earth varies with the latitude and elevation of the point of observation. This variable is often expressed using the symbol  $g$ . Caution should be exercised if this is done so as not to create an ambiguity with this use and the standard symbol for the unit of the gram.

**1.5**  
**force**

dynamic influence that changes a body from a state of rest to one of motion or changes its rate of motion

NOTE 1 A force could also change a body's size or shape if the body resists motion.

NOTE 2 The newton is the unit of force. One newton is the force required to give a mass of one kilogram an acceleration of one metre per second squared.

**1.6**  
**restoring force**

reaction force caused by the elastic property of a structure when it is being deformed

**1.7**  
**jerk**

rate of change of acceleration



**1.8****inertial reference system****inertial reference frame**

coordinate system or frame which is fixed in space or moves at constant velocity without rotational motion and thus, not accelerating

**1.9****inertial force**

reaction force exerted by a mass when it is being accelerated

**1.10****oscillation**

variation, usually with time, of the magnitude of a quantity with respect to a specified reference when the magnitude is alternately greater and smaller than the specified reference

NOTE 1 See **vibration** (2.1).

NOTE 2 Variations with time such as shock processes or creeping motions are also considered to be oscillations in a more general sense of the word.

**1.11****environment**

aggregate, at a given moment, of all external conditions and influences to which a system is subjected

NOTE See **induced environment** (1.12) and **natural environment** (1.13).

**1.12****induced environment**

conditions external to a system generated as a result of the operation of the system

**1.13****natural environment**

conditions generated by the forces of nature and the effects of which are experienced by a system when it is at rest as well as when it is in operation

**1.14****preconditioning**

climatic and/or mechanical and/or electrical treatment procedure which may be specified for a particular system so that it attains a defined state

**1.15****conditioning**

climatic and/or mechanical and/or electrical conditions to which a system is subjected in order to determine the effect of such conditions upon it

**1.16****excitation****stimulus**

external force (or other input) applied to a system that causes the system to respond in some way

**1.17****response (of a system)**

output quantity of a system

**1.18****transmissibility**

non-dimensional complex ratio of the response of a system in forced vibration to the excitation

NOTE 1 The ratio may be one of forces, displacements, velocities or accelerations.

NOTE 2 This is sometimes known as a transmissibility function.

**1.19**

**overshoot**

when the maximum transient response exceeds the desired response

NOTE 1 If the output of a system is changed from a steady value A to a steady value B by varying the input, such that value B is greater than A, then the response is said to overshoot when the maximum transient response exceeds value B.

NOTE 2 The difference between the maximum transient response and the value B is the value of the overshoot. This is usually expressed as a percentage.

**1.20**

**undershoot**

when the minimum transient response falls below the desired response

NOTE 1 If the output of a system is changed from a steady value A to a steady value B by varying the input, such that value B is less than A, then the response is said to undershoot when the minimum transient response is less than value B.

NOTE 2 The difference between the minimum transient response and the value B is the value of the undershoot. This is usually expressed as a percentage.

**1.21**

**system**

set of interrelated elements considered in a defined context as a whole and separated from their environment

**1.22**

**linear system**

system in which the magnitude of the response is proportional to the magnitude of the excitation

NOTE This definition implies that the principle of superposition can be applied to the relationship between output response and input excitation.

**1.23**

**mechanical system**

system comprising elements of mass, stiffness and damping

iteh STANDARD PREVIEW  
(standards.iteh.ai)  
ISO 2041:2009  
<https://standards.iteh.ai/catalog/standards/sist/2f54bbfe-70bb-46da-8714-a72775b8ec8f/iso-2041-2009>

**1.24**

**foundation**

structure that supports a mechanical system

NOTE It can be fixed in a specified reference frame or it can undergo a motion.

**1.25**

**seismic system**

system consisting of a mechanical system attached to a reference base by one or more flexible elements, with damping normally included

NOTE 1 Seismic systems are usually idealized as single-degree-of-freedom systems with viscous damping.

NOTE 2 The natural frequencies of the mass as supported by the flexible elements are relatively low for seismic systems associated with displacement or velocity transducers, and are relatively high for acceleration transducers, as compared with the range of frequencies to be measured.

NOTE 3 When the natural frequency of the seismic system is low relative to the frequency range of interest, the mass of the seismic system may be considered to be at rest over this range of frequencies.

**1.26**

**equivalent system**

system that can be substituted for another system for the purpose of analysis

NOTE Many types of equivalence are common in vibration and shock technology:

- a) a torsional system equivalent to a translational system;
- b) an electrical or acoustical system equivalent to a mechanical system, etc.;
- c) equivalent stiffness;
- d) equivalent damping.

### **1.27 degrees of freedom**

minimum number of generalized coordinates required to define completely the configuration of a mechanical system

NOTE 1 This applies to mechanical systems, not to be confused with statistical degrees of freedom.

NOTE 2 It is often referred to by the acronym DOF.

### **1.28 discrete system lumped parameter system**

mechanical system in which the mass, stiffness, and/or damping elements are discretely located

### **1.29 single-degree-of-freedom system SDOF**

system requiring only one coordinate to define completely its configuration at any instant

### **1.30 multi-degree-of-freedom system**

system for which two or more coordinates are required to define completely the configuration of the system at any instant

### **1.31 continuous system**

mechanical system in which the mass, stiffness, and/or damping properties are spatially distributed rather than discretely located

NOTE The configuration of a continuous system is specified by a function of a continuous spatial variable, or variables, in contrast to a discrete or lumped parameter system that requires only a finite number of coordinates to specify its configuration.

### **1.32 centre of gravity**

point through which the resultant of the weights of its component particles passes without resulting in moment for all orientations of the body with respect to a gravitational field

NOTE If the field is uniform, the centre of gravity coincides with the **centre of mass** (1.33).

### **1.33 centre of mass**

point of a body where the first moment of the overall mass with reference to a Cartesian coordinate system is equal to the first moments of mass of all points of the body

NOTE This is the point at which an object is in balance in a uniform gravitational field.

### **1.34 principal axes of inertia**

three mutually perpendicular axes intersecting each other at a given point about which the products of inertia of a solid body are zero

NOTE 1 If the point is the centre of mass of the body, the axes and moments are called central principal axes and central principal moments of inertia.

NOTE 2 In balancing, the term “principal inertia axis” is used to designate the one central principal axis (of the three such axes) most nearly coincident with the shaft axis of the rotor and is sometimes referred to as the balance axis or the mass axis.

**1.35**  
**moment of inertia**

sum (integral) of the product of the masses of the individual particles (elements of mass) of a body and the square of their perpendicular distances from the axis of rotation

**1.36**  
**product of inertia**

sum (integral) of the product of the masses of the individual particles (elements of mass) of a body and their distances from two mutually perpendicular planes

**1.37**  
**stiffness**

ratio of change of force (or torque) to the corresponding change in translational (or rotational) deformation of an elastic element

NOTE See also **dynamic stiffness** (1.58).

**1.38**  
**compliance**

reciprocal of stiffness

NOTE See also **dynamic compliance** (1.57).

**1.39**  
**neutral surface**  
**neutral surface of a beam in simple flexure**

surface in which there is no strain

NOTE It should be stated whether or not the neutral surface is a result of the flexure alone, or whether it is a result of the flexure and other superimposed loads.

**1.40**  
**neutral axis**  
**neutral axis of a beam in simple flexure**

line or plane in a beam where the longitudinal stress, tensile or compressive is zero

**1.41**  
**transfer function**

mathematical representation of the relationship between the input and output of a linear time-invariant system

NOTE 1 A transfer function is usually a complex function defined as the ratio of the Laplace transforms of the output to the input of a linear time-invariant system.

NOTE 2 It is usually given as a function of frequency, and is usually a complex function. See **response** (1.17), **transmissibility** (1.18) and **transfer impedance** (1.50).

**1.42**  
**complex excitation**

excitation expressed as a complex quantity with amplitude and phase angle

NOTE 1 The concepts of complex excitations and responses were evolved historically in order to simplify calculations. The actual excitation and response are the real parts of the complex excitation and response. If the system is linear, the concept is valid because superposition holds in such a situation.

NOTE 2 This term should not be confused with excitation by a complex vibration, or vibration of complex waveform. The use of the term “complex vibration” in this sense is deprecated.

iTeh STANDARD PREVIEW  
(standards.iteh.ai)

ISO 2041:2009

<https://standards.iteh.ai/catalog/standards/sist/2f4bb6-70bb-46da-8714-a72775b8ec8f/iso-2041-2009>

**1.43****complex response**

response of a system expressed as a complex quantity with amplitude and phase angle from a specified excitation

NOTE See the notes under **complex excitation** (1.42).

**1.44****modal analysis**

vibration analysis method that characterizes a complex structural system by its modes of vibration, i.e. its natural frequencies, modal damping and mode shapes, and based on the principle of superposition

**1.45****modal matrix**

linear transformation matrix which consists of the eigen vectors or modal vectors of a system

NOTE It renders the system both inertially and elastically uncoupled, i.e. the modal mass and modal stiffness matrices are transformed into diagonal matrices.

**1.46****modal stiffness**

stiffness element associated with a specified mode of vibration

**1.47****modal density**

number of modes per unit bandwidth

NOTE Modal density is a measure widely used in structural dynamics as a diagnostic tool in assessing vibration power flow in complex, structural systems. It can play a crucial role in determining changes in vibration power flow that may be a precursor to fatigue failure in some part of the structure, or a metric used in structural condition monitoring evaluations. In addition to these applications, it is a parameter required by the Statistical Energy Analysis method for evaluating the high-frequency response of complex structures and in selecting appropriate vibration-control methods and devices.

<https://standards.iteh.ai/catalog/standards/sist/2f54bbfe-70bb-46da-8714-a72775b8ec8f/iso-2041-2009>

**1.48****mechanical impedance**

complex ratio of force to velocity at a specified point and degree-of-freedom in a mechanical system

NOTE 1 The force and velocity may be taken at the same or different points and degrees-of-freedom in the system undergoing simple harmonic motion.

NOTE 2 In the case of torsional mechanical impedance, the terms “force” and “velocity” should be replaced by “torque” and “angular velocity”, respectively.

NOTE 3 In general, the term “impedance” applies to linear systems only.

NOTE 4 The concept is extended to non-linear systems where the term “incremental impedance” is used to describe a similar quantity.

**1.49****direct mechanical impedance  
driving point mechanical impedance**

complex ratio of the force to velocity taken at the same point or degree-of-freedom in a mechanical system during simple harmonic motion

NOTE See the notes under **mechanical impedance** (1.48).

**1.50****transfer (mechanical) impedance**

complex ratio of the force applied at point  $i$ , in a specified degree-of-freedom in a mechanical system, to the velocity at another point  $j$  in a specified direction or degree-of-freedom in the same system, during simple harmonic motion

NOTE See the notes under **mechanical impedance** (1.48).

### 1.51

#### **free impedance**

ratio of the applied excitation complex force to the resulting complex velocity with all other connection points of the system free, i.e. having zero restraining forces

NOTE 1 Historically, often no distinction has been made between blocked impedance and free impedance. Caution should, therefore, be exercised in interpreting published data.

NOTE 2 Free impedance is the arithmetic reciprocal of a single element of the mobility matrix. While experimentally determined free impedances could be assembled into a matrix, this matrix would be quite different from the blocked impedance matrix resulting from mathematical modelling of the structure and, therefore, would not conform to the requirements for using mechanical impedance in an overall theoretical analysis of the system.

### 1.52

#### **blocked impedance**

impedance at the input when all output degrees of freedom are connected to a load of infinite mechanical impedance

NOTE 1 Blocked impedance is the frequency-response function formed by the ratio of the phasor of the blocking or driving-point force response at point  $i$ , to the phasor of the applied excitation velocity at point  $j$ , with all other measurement points on the structure "blocked", i.e. constrained to have zero velocity. All forces and moments required to fully constrain all points of interest on the structure need to be measured in order to obtain a valid blocked impedance matrix.

NOTE 2 Any changes in the number of measurement points or their location will change the blocked impedances at all measurement points.

NOTE 3 The primary usefulness of blocked impedance is in the mathematical modelling of a structure using lumped mass, stiffness and damping elements or finite element techniques. When combining or comparing such mathematical models with experimental mobility data, it is necessary to convert the analytical blocked impedance matrix into a mobility matrix or vice versa.

### 1.53

#### **frequency-response function**

frequency-dependent ratio of the motion-response Fourier transform to the Fourier transform of the excitation force of a linear system

NOTE 1 Excitation can be harmonic, random or transient functions of time. The test results obtained with one type of excitation can thus be used for predicting the response of the system to any other type of excitation.

NOTE 2 Motion may be expressed in terms of velocity, acceleration or displacement; the corresponding frequency-response function designations are mobility, acceleration and dynamic compliance or impedance, effective (i.e. apparent) mass and dynamic stiffness, respectively (see Table 1).

### 1.54

#### **mobility**

##### **mechanical mobility**

complex ratio of the velocity, taken at a point in a mechanical system, to the force, taken at the same or another point in the system

NOTE 1 Mobility is the ratio of the complex velocity-response at point  $i$  to the complex excitation force at point  $j$  with all other measurement points on the structure allowed to respond freely without any constraints other than those constraints which represent the normal support of the structure in its intended application.

NOTE 2 The term "point" designates both a location and a direction.

NOTE 3 The velocity response can be either translational or rotational, and the excitation force can be either a rectilinear force or a moment.

NOTE 4 If the velocity response measured is a translational one and if the excitation force applied is a rectilinear one, the units of the mobility term will be  $m/(N \cdot s)$  in the SI system.

NOTE 5 Mechanical mobility is the matrix inverse of mechanical impedance.

**1.55****direct (mechanical) mobility**  
**driving-point (mechanical) mobility**

complex ratio of velocity and force taken at the same point in a mechanical system

NOTE Driving-point mobility is the frequency-response function formed by the ratio, in metres per Newton second, of the velocity-response complex amplitude at point  $j$  to the excitation force complex amplitude applied at the same point with all other measurement points on the structure allowed to respond freely without any constraints other than those constraints which represent the normal support of the structure in its intended application.

**1.56****transfer (mechanical) mobility**

mechanical mobility where the velocity and the force are considered at different points of the system

**1.57****dynamic compliance**

frequency-dependent ratio of the spectrum, or spectral density, of the displacement to the spectrum, or spectral density, of the force

**1.58****dynamic stiffness**

complex ratio of the force, taken at a point in a mechanical system, to the displacement, taken at the same or another point in the system

NOTE 1 The terms “dynamic elastic constant” and “dynamic spring constant” are sometimes used.

NOTE 2 The dynamic stiffness may be dependent upon strain (amplitude and frequency), strain-rate, temperature or other conditions.

NOTE 3 The dynamic stiffness,  $k^*$ , of a linear translational single-degree-of-freedom system characterized by the equation

$$m \frac{d^2x}{dt^2} + c \frac{dx}{dt} + kx = F \quad \text{where } F = F_0 e^{i\omega t}$$

is equal to

$$k = (F_0 + m\omega^2 x_0 - i\omega c x_0) / x_0$$

where

- $c$  is the linear (viscous) damping coefficient;
- $e$  is the base of natural logarithms;
- $F_0$  is the force amplitude;
- $i = \sqrt{-1}$ ;
- $k$  is the elastic (spring) constant;
- $m$  is the mass;
- $t$  is the time;
- $x$  is the displacement;
- $x_0$  is the displacement amplitude;
- $\omega$  is the angular frequency.