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**Cranes and lifting appliances — Selection  
of wire ropes —**

**Part 1:  
General**

*Grues et appareils de levage — Choix des câbles —  
Partie 1: Généralités*  
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ISO 4308-1:2003

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Published in Switzerland

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## Foreword

ISO (the International Organization for Standardization) is a worldwide federation of national standards bodies (ISO member bodies). The work of preparing International Standards is normally carried out through ISO technical committees. Each member body interested in a subject for which a technical committee has been established has the right to be represented on that committee. International organizations, governmental and non-governmental, in liaison with ISO, also take part in the work. ISO collaborates closely with the International Electrotechnical Commission (IEC) on all matters of electrotechnical standardization.

International Standards are drafted in accordance with the rules given in the ISO/IEC Directives, Part 2.

The main task of technical committees is to prepare International Standards. Draft International Standards adopted by the technical committees are circulated to the member bodies for voting. Publication as an International Standard requires approval by at least 75 % of the member bodies casting a vote.

Attention is drawn to the possibility that some of the elements of this document may be the subject of patent rights. ISO shall not be held responsible for identifying any or all such patent rights.

ISO 4308-1 was prepared by Technical Committee ISO/TC 96, *Cranes*, Subcommittee SC 3, *Selection of wire ropes*.

This third edition cancels and replaces the second edition (ISO 4308-1:1986), which has been technically revised.

ISO 4308 consists of the following parts, under the general title *Cranes and lifting appliances — Selection of wire ropes*:

— *Part 1: General*

— *Part 2: Mobile cranes — Coefficient of utilization*

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# Cranes and lifting appliances — Selection of wire ropes —

## Part 1: General

### 1 Scope

This part of ISO 4308 specifies two methods for the selection of wire rope to be used on lifting appliances as designated in ISO 4306-1, one based on the value of the rope selection factor  $C$  and the other based on the value of the coefficient of utilization  $Z_p$ .

This part of ISO 4308 establishes the minimum requirements for acceptable strength and performance levels of wire ropes with respect to the design, application and maintenance of the lifting appliance.

This part of ISO 4308-1 establishes the minimum requirements for the diameters of drums and sheaves that are to be associated with the selected wire rope.

A non-exhaustive list of types of lifting appliance to which this part of ISO 4308 is applicable is given in Annex A.

Annex B provides some examples of rope selection.

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Annex C gives factors, additional to those mentioned above, which may need consideration when selecting the wire rope.

Annex D specifies the selection method for the diameter of the compensating sheave when used in relation to hoists.

### 2 Normative references

The following referenced documents are indispensable for the application of this document. For dated references, only the edition cited applies. For undated references, the latest edition of the referenced document (including any amendments) applies.

ISO 2408:1985, *Steel wire ropes for general purposes — Characteristics*

ISO 4301-1:1986, *Cranes and lifting appliances — Classification — Part 1: General*

ISO 4306-1:1990, *Cranes — Vocabulary — Part 1: General*

ISO 4309, *Cranes — Wire ropes — Care, maintenance (including installation), inspection*

### 3 Terms and definitions

For the purposes of this part of ISO 4308, the following terms and definitions apply.

**3.1 parallel-closed rope**  
stranded rope consisting of at least two layers of strand laid helically in one closing operation around a strand or fibre centre

**3.2 rotation-resistant rope**  
**multi-strand rope**  
**non-rotating rope**  
stranded rope designed to generate reduced levels of torque and rotation when loaded

NOTE 1 Rotation-resistant ropes generally comprise an assembly of two or more layers of strands laid helically around a centre, the direction of lay of the outer strands being opposite to that of the underlying layer.

NOTE 2 Ropes having three or four strands can also be designed to exhibit rotation-resistant properties.

**3.3 single-layer rope**  
stranded rope consisting of one layer of strands laid helically around a core

**3.4 stranded rope**  
assembly of several strands, laid helically in one or more layers around a core (single-layer rope) or centre (rotation-resistant or parallel-closed rope) (standards.iteh.ai)

NOTE Single-layer ropes consisting of three and four strands may or may not have a core.

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### 4 Type of rope

Where possible, the wire rope selected shall be in accordance with ISO 2408.

Selection of wire rope not specified by ISO 2408 is permitted, but in such cases the supplier of the wire rope shall show, to the user, documentation which is supported by the rope-maker's technical file for the product, that clearly demonstrates that the product has acceptable strength and performance levels with respect to the design of mechanisms, application and maintenance of the appliance.

### 5 Duty conditions

The mechanisms of lifting appliances shall be classified according to the duty conditions laid down in ISO 4301-1.

## 6 Selection procedure

### 6.1 Calculation of $C$ values

The value of the rope selection factor,  $C$ , is a function of the coefficient of utilization,  $Z_p$ , and is given by Equation (1):

$$C = \sqrt{\frac{Z_p}{K' \cdot R_0}} \quad (1)$$

where

$C$  is the rope selection factor (minimum);

$K'$  is the empirical factor for minimum breaking load of a given rope construction (see Table 3 of ISO 2408:1985 or as otherwise provided by the rope supplier);

$R_0$  is the minimum tensile strength of the wire used in the rope, in newtons per square millimetre<sup>1)</sup>;

$Z_p$  is the minimum practical coefficient of utilization.

### 6.2 $Z_p$ values

Table 1 gives the values of  $Z_p$  which shall be used for each classification group of mechanism in order to meet the minimum requirements of this part of ISO 4308. It also gives the calculated values of  $C$  corresponding to the rope type (6 × 36 WS - IWRC) with  $R_0 = 1\,770 \text{ N/mm}^2$  and with an empirical factor  $K' = 0,356$ .

**Table 1 —  $Z_p$  values and  $C$  values (for  $R_0 = 1\,770 \text{ N/mm}^2$  and  $K' = 0,356$ )**

Classification of mechanism	$Z_p$ value	$C$ value
M1	3,15	0,071
M2	3,35	0,073
M3	3,55	0,075
M4	4,0	0,080
M5	4,5	0,085
M6	5,6	0,094
M7	7,1	0,106
M8	9,0	0,120

NOTE Whilst Equation (1) shows the exact relationship between  $C$  and  $Z_p$ , the values shown in Table 1 have been corrected by rounding to three decimal places.

For ropes having a tensile strength  $R_0$  and an empirical factor  $K'$  different from those shown above, different values of  $C$  may be calculated using Equation (1) and substituted in Equation (2) indicated in 6.3.

1)  $1 \text{ N/mm}^2 = 10^6 \text{ N/m}^2 = 1 \text{ MPa}$

### 6.3 Calculation of minimum rope diameter

The minimum diameter of the rope,  $d_{\min}$ , in millimetres, is given by Equation (2):

$$d_{\min} = C\sqrt{S} \quad (2)$$

where

$d_{\min}$  is the calculated minimum diameter of the rope, and is the value used in the selection process for calculating the drum and sheave diameters;

$C$  is the rope selection factor;

$S$  is the maximum rope tension, in newtons, obtained by taking into account the following factors:

- rated working load of the appliance;
- mass of the pulley block and/or other lifting attachments;
- mechanical advantage of reeving;
- efficiency of the rope reeving;
- the increase in force in the rope caused by the rope inclination at the upper extreme position of the hook, if the rope inclination with respect to the drum axis exceeds 22,5°.

The nominal diameter of the rope selected ( $d$ ) shall be within the range:  $d_{\min}$  to  $d_{\min} \times 1,25$ .

### 6.4 Calculation of minimum breaking force

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The minimum breaking force,  $F_{\min}$ , in newtons, of the particular rope intended for use is given by Equation (3):

$$F_{\min} = S \cdot Z_p \quad (3)$$

where

$S$  is the maximum rope tension, in newtons, as established in 6.3;

$Z_p$  is the minimum practical coefficient of utilization.

Examples of rope selection are given in Annex B.



## 7 Diameter of rope drums and sheaves

The minimum pitch circle diameter of the rope drums and rope sheaves shall be calculated using the minimum rope diameter established in 6.3 and by applying the respective values of  $h_1$ ,  $h_2$  as shown in Table 2 and the rope type factor  $t$ , as shown in Table 3, as applicable, and which relates to the classification of the mechanism, in Equations (4) and (5):

$$D_1 \geq h_1 \cdot t \cdot d_{\min} \quad (4)$$

or

$$D_2 \geq h_2 \cdot t \cdot d_{\min} \quad (5)$$

where

$D_1$  is the minimum pitch circle diameter of the drum;

$D_2$  is the minimum pitch circle diameter of the sheave;

$d_{\min}$  is the minimum diameter of the rope, calculated in accordance with 6.3;

$h_1$  is the selection factor for the drum (ratio of the pitch circle diameter of the drum to the calculated diameter of the rope);

$h_2$  is the selection factor for the sheave (ratio of the pitch circle diameter of the sheave to the calculated diameter of the rope);

$t$  is the rope type factor in accordance with Table 3. The rope type factor takes into account the different bending fatigue performance of different types of rope.

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**Table 2 — Selection factors  $h_1$  and  $h_2$**

Classification of mechanism	Drums	Sheaves
	$h_1$	$h_2$
M1	11,2	12,5
M2	12,5	14,0
M3	14,0	16,0
M4	16,0	18,0
M5	18,0	20,0
M6	20,0	22,4
M7	22,4	25,0
M8	25,0	28,0

For hoists, the minimum pitch circle diameter of any compensating sheave shall be calculated in accordance with Annex D.

**Table 3 — Rope type factor  $t$  for various rope types**

Number of outer strands in the rope	Rope type factor $t$
3 to 5	1,25
6 to 10	1,00
8 to 10 plastic impregnation	0,95
10 and greater RR <sup>a</sup>	1,00
<sup>a</sup> Rotation-resistant rope	

## 8 Stationary ropes

Stationary ropes are fixed at both ends and are not subject to coiling on a drum or over a sheave. Their selection is made in accordance with 6.4, with  $Z_p$  values modified as in Table 4, where the maximum rope tension,  $S$ , shall be established by the manufacturer of the mechanism who shall take account of both the static forces.

**Table 4 —  $Z_p$  values for stationary ropes**

Classification of mechanism	$Z_p$ value
M1	2,5
M2	2,5
M3	3,0
M4	3,5
M5	4,0
M6	4,5
M7	5,0
M8	5,0

## 9 Dangerous conditions

For dangerous conditions, for example the handling of molten metal,

- a) no classification group lower than M5 shall be used,
- b) the  $Z_p$  value shall be increased by 25 % up to a maximum of 9,0 or, alternatively, the  $C$  value for the next higher classification group shall be adopted when selecting the rope.

## 10 Care, maintenance, examination and discard

The selection of the ropes, drums and sheaves according to this part of ISO 4308 does not alone ensure safe operation of the rope for indefinite periods.

The instructions on care, maintenance (including installation), examination and discard for wire ropes given in ISO 4309 shall be adopted.

## Annex A (normative)

### Lifting appliances to which this part of ISO 4308 is applicable

This part of ISO 4308 is applicable to the following non-exhaustive list (extracted from ISO 4306-1) of cranes and lifting appliances:

- a) overhead travelling cranes;
- b) hoists-wire rope;
- c) portal or semi-portal cranes;
- d) portal or semi-portal bridge cranes;
- e) cable and portal cable cranes (hoist and trolley mechanisms only);
- f) mobile cranes;
- g) tower cranes;
- h) railway cranes;
- i) floating cranes;
- j) deck cranes;
- k) derrick and guy derrick cranes;
- l) derrick cranes with rigid bracing;
- m) cantilever cranes (pillar, jib, wall or walking).

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The cranes can be used for hook, grabbing, magnet, ladle, bucket or stacking duties, and can be operated manually, electrically or hydraulically.