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Prevailing torque type steel nuts — Mechanical and performance properties

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Cont	r <mark>ents</mark>	Page
Forewo	ord	iv
1	Scope	1
2	Normative references	1
3	Terms and definitions	2
4	Symbols	2
5	Thread	3
6	Lubrication	3
7	Mechanical properties of prevailing torque type nuts	3
8	Performance requirements for prevailing torque properties	3
9 9.1 9.2 9.3	Test method General Proof load test Prevailing torque test	12 12 12
Annex	A (normative) Temperature resistance of prevailing torque non-metallic insert type nuts	17
Annex	B (informative) Basis for the evaluation of the total coefficient of friction, μ_{tot}	18
Bibliog	ISO 2320:2008 https://standards.iteh.ai/catalog/standards/sist/213967cb-43c7-466f-8445-	19

a96759d07c48/iso-2320-2008

Foreword

ISO (the International Organization for Standardization) is a worldwide federation of national standards bodies (ISO member bodies). The work of preparing International Standards is normally carried out through ISO technical committees. Each member body interested in a subject for which a technical committee has been established has the right to be represented on that committee. International organizations, governmental and non-governmental, in liaison with ISO, also take part in the work. ISO collaborates closely with the International Electrotechnical Commission (IEC) on all matters of electrotechnical standardization.

International Standards are drafted in accordance with the rules given in the ISO/IEC Directives, Part 2.

The main task of technical committees is to prepare International Standards. Draft International Standards adopted by the technical committees are circulated to the member bodies for voting. Publication as an International Standard requires approval by at least 75 % of the member bodies casting a vote.

Attention is drawn to the possibility that some of the elements of this document may be the subject of patent rights. ISO shall not be held responsible for identifying any or all such patent rights.

ISO 2320 was prepared by Technical Committee ISO/TC 2, Fasteners, Subcommittee SC 1, Mechanical properties of fasteners.

This fourth edition cancels and replaces the third edition (ISO 2320:1997), which has been technically revised. It also incorporates the Technical Corrigendum ISO 2320:1997/Cor.1;2006.

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Prevailing torque type steel nuts — Mechanical and performance properties

1 Scope

This International Standard specifies the mechanical and performance properties for prevailing torque type steel nuts when tested at an ambient temperature range of +10 °C to +35 °C. It includes a single test to determine the prevailing torque properties (performance properties) and/or the torque/clamp force properties.

This International Standard applies to prevailing torque all metal type nuts and prevailing torque non-metallic insert type nuts:

- a) with triangular ISO thread according to ISO 68-1;
- b) with diameter/pitch combination according to ISO 261 and ISO 262;
- c) with coarse pitch thread M3 to M39 and mechanical properties according to ISO 898-2;
- d) with fine pitch thread M8×1 to M39×3 and mechanical properties according to ISO 898-6;
- e) within the temperature range of -50 °C to +150 °C for prevailing torque all metal type nuts;

NOTE 1 See Clause 7, paragraph 3. See Clause 7, paragraph 3. a96759d07c48/iso-2320-2008

f) within the temperature range of –50 °C to +120 °C for prevailing torque non-metallic insert type nuts.

NOTE 2 See Clause 7, paragraph 4.

2 Normative references

The following referenced documents are indispensable for the application of this document. For dated references, only the edition cited applies. For undated references, the latest edition of the referenced document (including any amendments) applies.

ISO 273:1979, Fasteners — Clearance holes for bolts and screws

ISO 898-1, Mechanical properties of fasteners made of carbon steel and alloy steel — Part 1: Bolts, screws and studs

ISO 898-2, Mechanical properties of fasteners — Part 2: Nuts with specified proof load values — Coarse thread

ISO 898-6, Mechanical properties of fasteners — Part 6: Nuts with specified proof load values — Fine pitch thread

ISO 965-2, ISO general purpose metric screw threads — Tolerances — Part 2: Limits of sizes for general purpose external and internal screw threads — Medium quality

ISO 16047, Fasteners — Torque/clamp force testing

Terms and definitions

For the purposes of this document, the terms and definitions given in ISO 16047 and the following apply.

3.1

prevailing torque type nut

nut which is not free-running on a mating thread by virtue of a self-contained prevailing torque feature, and which provides a degree of resistance to rotation independent of clamping or compression forces

3.2

prevailing torque developed by the nut

torque necessary to rotate the nut on its mating externally threaded component and with no axial force in the mating component

3.3

prevailing-on torque

torque to rotate the nut on its mating externally threaded component with the torque measured while the nut is in motion and with no clamp force

3.4

prevailing-off torque

torque to rotate after backing off the nut until the removal of the clamp force in the externally threaded component in the following 360° rotation of the nut

3.5

prevailing torque all metal type nut 1 STANDARD PREVIEW

nut which has a one piece or a multiple piece metal construction and derives its prevailing torque characteristics from a controlled distortion of the nut thread and/or body or from metallic insert(s)

3.6

ISO 2320:2008

prevailing torque non-metallic insert type nut tatalog/standards/sist/213967cb-43c7-466f-8445-nut which has a multiple piece construction and derives its prevailing torque characteristics from insert(s) of non-metallic material located and retained in the nut

3.7

seating point

point in the tightening process where clamp force first appears

Symbols

For the purpose of this International Standard, the following symbols apply together with those defined in ISO 16047.

Symbol Designation

d	nominal diameter
d_4	diameter of the hole of the fixture
F_{P}	proof load
F ₆₅	lower load limit for the evaluation of the coefficient of total friction at 65 % of $F_{\mbox{\footnotesize{P}}}$
F ₇₅	upper load limit for the evaluation of the coefficient of total friction at 75 % of $F_{\mbox{\scriptsize P}}$
F ₈₀	test clamp force (shut-down force for the tightening process) at 80 % of $F_{\mbox{\scriptsize P}}$
P	pitch of the thread
T_{FV}	prevailing-on torque, in newton metres

T_{Fd}	prevailing-off torque, in newton metres
T ₆₅	lower torque limit for the evaluation of the coefficient of total friction at F_{65}
T ₇₅	upper torque limit for the evaluation of the coefficient of total friction at ${\cal F}_{75}$
T ₈₀	test torque corresponding to 80 $\%$ of the proof load, in newton metres (see Tables 1 to 8)
μ_{tot}	coefficient of total friction

5 Thread

Threads for prevailing torque type nuts shall be in accordance with ISO 965-2 except for the prevailing torque feature:

- a) for prevailing torque non-metallic insert type nuts, the GO gauge shall be suitable for free installation (by hand) until it is seated against the prevailing torque feature;
- b) for prevailing torque all metal type nuts, the GO gauge shall be suitable for free installation (by hand) to one pitch at least.

6 Lubrication

At the option of the manufacturer a lubricant may be applied to the manufacturing lot to fulfil the performance requirements.

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7 Mechanical properties of prevailing torque type nuts

The mechanical properties of prevailing torque type nuts shall conform to ISO 898-2 or ISO 898-6.

With regard to proof load, the test method specified in 9.2 shall apply.

For prevailing torque all metal type nuts, users should consult an experienced fastener materials expert for temperatures outside the range of -50 °C to +150 °C to determine appropriate choices for a given application.

For prevailing torque non-metallic insert type nuts, use at or near the temperature limits of -50 °C and +120 °C may reduce the prevailing torque capability and may require the use of an adequate non-metallic material. Users should consult an experienced fastener materials expert for temperatures outside the range of -50 °C to +120 °C to determine appropriate choices for a given application.

8 Performance requirements for prevailing torque properties

The prevailing-on torque shall not exceed the value specified for the applicable nut in Tables 1 to 8.

The prevailing-off torque shall exceed the value specified for the applicable nut in Tables 1 to 8.

For delivery inspection, the 1st installation/removal test applies, unless otherwise agreed.

For initial type testing and in case of dispute, a 5th removal test shall also be applied unless otherwise agreed.

Prevailing torque performance decreases as a function of the number of reuses; the consumer should take into consideration the consequences of the decreased performance before any reuse of the nut.

By request of the customer, a temperature resistance test for prevailing torque non-metallic insert type nuts as given in Annex A may be carried out.

Paragraphs 3 and 4 of Clause 7 also apply to performance requirements.

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Table 1 — Test clamp force and prevailing torques for prevailing torque type nuts of property class 04

Thread	Test clamp force	Clamp force for evaluation of total friction coefficient $\mu_{\mathrm{tot}}^{}\mathrm{b}}$		Prevailing torque N⋅m			
d P	F ₈₀ ^a N	Upper limit F_{75}^{c}	Lower limit F_{65}^{d}	1st installation	1st removal	5th removal	
		N	N	$T_{Fv,max}^{}e}$	$T_{Fd,min}^{\qquad \qquad f}$	$T_{Fd,min}^{\qquad \qquad f}$	
M3	1 528	1 433	1 242	0,43	0,12	0,08	
M4	2 672	2 505	2 171	0,9	0,18	0,12	
M5	4 320	4 050	3 510	1,6	0,29	0,2	
M6	6 112	5 730	4 966	3	0,45	0,3	
M7	8 800	8 250	7 150	4,5	0,65	0,45	
M8	11 120	10 425	9 035		0.05	0.0	
M8×1	11 920	11 175	9 685	6	0,85	0,6	
M10	17 600	16 500	14 300				
M10×1,25	18 640	17 475	15 145	10,5	1,5	1	
M10×1	19 600	18 375	15 925				
M12	25 600	24 000	20 800		2,3		
M12×1,5	26 800	25 125	21 775	15,5		1,6	
M12×1,25	28 000	26 250	22 750				
M14	34 960	32 775	28 405	0.4	3,3 TEVV	0.0	
M14×1,5	38 000	35 625	1 ∆ 30 875 ∆	RD PREV		2,3	
M16	47 760	44 775	38 805	20	4,5	_	
M16×1,5	50 800	47 625	Sta41275arc	s.iten.ai)		3	
M18	58 400	54 750	47 450	40		4.0	
M18×1,5	65 360	61 275	53 1 05	0:2008	6	4,2	
M20	74 480	https69s825lards.i		rds/sist/213967cb-43c	:7-466f- <u>844</u> 5-	5.0	
M20×1,5	82 720	77 550	a9 675210)7c48/	so-2320-2008		5,3	
M22	92 080	86 325	74 815		0.5	0.5	
M22×1,5	101 200	94 875	82 225	68	9,5	6,5	
M24	107 280	100 575	87 165	00	11,5		
M24×2	116 720	109 425	94 835	80		8	
M27	139 520	130 800	113 360	0.1	13,5	10	
M27×2	150 800	141 375	122 525	94		10	
M30	170 560	159 900	138 580	108	16	10	
M30×2	188 800	177 000	153 400			12	
M33	210 960	197 775	171 405	465	18		
M33×2	231 360	216 900	187 980	122		14	
M36	248 400	232 875	201 825	400	21		
M36×3	262 960	246 525	213 655	136		16	
M39	296 720	278 175	241 085		23		
M39×3	313 120	293 550	254 410	150		18	

a The clamp force for property class 04 nuts is equal to 80 % of the proof load of property class 04 nuts for 3 mm $\leq d \leq$ 39 mm. Proof loads for nuts are given in ISO 898-2 and ISO 898-6.

b See Annex B.

^c The value of the upper limit of the clamp force is equal to 75 % of the proof load, see Annex B.

The value of the lower limit of the clamp force is equal to 65 % of the proof load, see Annex B.

e The prevailing torques for first assembly apply for all metal type nuts only. For prevailing torque non-metallic insert type nuts, the maximum torques shall be 50 % of the values.

f Values in this table are required for testing performed under laboratory acceptance test conditions. Utilization of this type of fastener is application dependent and performance for parts may vary in normal use. It is recommended that additional testing of complete joints, using production components, be performed when there are questions of product performance.

Table 2 — Test clamp force and prevailing torques for prevailing torque type nuts of property class 05

Thread	Test clamp force	Clamp force for evaluation of total friction coefficient $\mu_{tot}^{\ \ b}$		Prevailing torque		
d P	F_{80}^{a}	Upper limit	Lower limit	1st installation	1st removal	5th removal
	N	F_{75}^{c}	$F_{65}{}^{d}$	$T_{Fv,max}^{}e}$	$T_{Ed.min}^{f}$	$T_{Fd,min}^{\qquad \qquad f}$
		N	N	¹ Fv,max	¹ Fd,min	¹ Fd,min
M3	2 000	1 875	1 625	0,6	0,15	0,1
M4	3 520	3 300	2 860	1,2	0,22	0,15
M5	5 680	5 325	4 615	2,1	0,35	0,24
M6	8 000	7 500	6 500	4	0,55	0,4
M7	11 600	10 875	9 425	6	0,85	0,6
M8	14 640	13 725	11 895		4.45	0.0
M8×1	15 680	14 700	12 740	8	1,15	0,8
M10	23 200	21 750	18 850			
M10×1,25	24 480	22 950	19 890	14	2	1,4
M10×1	25 760	24 150	20 930			,
M12	33 760	31 650	27 430			
M12×1,5	35 200	33 000	28 600	21	3,1	2,1
M12×1,25	36 800	34 500	29 900			
M14	46 000	43 125	37 375	0.4		_
M14×1,5	50 000	46 875	40 625	31	4,4	3
M16	62 800 T	58 8 75	△ 51 025 P R	EV 42 W	6	4.0
M16×1,5	66 800	62 625	54 275	420 * *		4,2
M18	76 800	72000 n d	ard62 400 ch	ai)	_	
M18×1,5	86 000	80 625	69 875	56	8	5,5
M20	98 000	91 875	79 625		10,5	7
M20×1,5	108,800	102 000	88 400	72		
M22	121 200 S.//Sta	andards iteh a/catalog	standard 8/475 ² 13967	cb-43c/-4661-844.	13	_
M22×1,5	133 200	124 875 739d	0/c48/108-225	90		9
M24	141 200	132 375	114 725	100	15	40.5
M24×2	153 600	144 000	124 800	106		10,5
M27	183 600	172 125	149 175	100	17	40
M27×2	198 400	186 000	161 200	123		12
M30	224 400	210 375	182 325	140	19	4.4
M30×2	248 400	232 875	201 825			14
M33	277 600	260 250	225 550	165	21,5	45.5
M33×2	304 400	285 375	247 325	160		15,5
M36	326 800	306 375	265 525		24	47.5
M36×3	346 000	324 375	281 125	180		17,5
M39	390 400	366 000	317 200	222	26,5	19,5
M39×3	412 000	386 250	334 750	200		

The clamp force for property class 05 nuts is equal to 80 % of the proof load of property class 05 nuts for 3 mm $\leq d \leq$ 39 mm. Proof loads for nuts are given in ISO 898-2 and ISO 898-6.

b See Annex B.

^C The value of the upper limit of the clamp force is equal to 75 % of the proof load, see Annex B.

The value of the lower limit of the clamp force is equal to 65 % of the proof load, see Annex B.

e The prevailing torques for first assembly apply for all metal type nuts only. For prevailing torque non-metallic insert type nuts, the maximum torques shall be 50 % of the values.

f Values in this table are required for testing performed under laboratory acceptance test conditions. Utilization of this type of fastener is application dependent and performance for parts may vary in normal use. It is recommended that additional testing of complete joints, using production components, be performed when there are questions of product performance.

Table 3 — Test clamp force and prevailing torques for prevailing torque type nuts of property class 5

Thread	Test clamp force	Clamp force for evaluation of total friction coefficient $\mu_{\mathrm{tot}}^{\ \ \mathrm{b}}$		Prevailing torque N·m		
d P	F_{80}^{a}	Upper limit	Lower limit	1st installation	1st removal	5th removal
	N	F ₇₅ ^c	F_{65}^{d}	т е	T f	T f
		N	N	$T_{Fv,max}^{}}$	$T_{Fd,min}^{\qquad \qquad f}$	$T_{Fd,min}^{}}$
M3	1 528	1 433	1 242	0,43	0,12	0,08
M4	2 672	2 505	2 171	0,9	0,18	0,12
M5	4 320	4 050	3 510	1,6	0,29	0,2
M6	6 112	5 730	4 966	3	0,45	0,3
M7	8 800	8 250	7 150	4,5	0,65	0,45
M8	11 120	10 425	9 035	_	0.05	0.0
M8×1	11 920	11 175	9 685	6	0,85	0,6
M10	17 600	16 500	14 300			
M10×1,25	18 640	17 475	15 145	10,5	1,5	1
M10×1	19 600	18 375	15 925			
M12	25 600	24 000	20 800			
M12×1,5	26 800	25 125	21 775	15,5	2,3	1,6
M12×1,25	28 000	26 250	22 750			
M14	34 960	32 775	28 405			
M14×1,5	38 000	35 625	30 875	24	3,3	2,3
M16	47 760	44775	A 38 805 R	PREVI	FW.	
M16×1,5	50 800	47 625	41 275	7 F F32 V L	4 ,5	3
M18	58 400	54 750	an 47450 Cj	teh.ai)		4.0
M18×1,5	65 680	61 575	53 365	C111421)	6	4,2
M20	74 480	69 825	60 515		7.5	F 0
M20×1,5	82 400	77 250	66 950	8 54	7,5	5,3
M22	92 000	11ttps://starr.lards.iteli.	i/catalo <u>9/stari lards/sis</u> 74 750	t/21396/cb-43c/-4	4661-8445- 9,5	0.5
M22×1,5	100 800	94 500	a96/5960/c48/IS0-23	20-200 68		6,5
M24	107 200	100 500	87 100	00	11,5	0
M24×2	116 800	109 500	94 900	80		8
M27	113 600	106 500	92 300	94	13,5	40
M27×2	123 200	115 500	100 100			10
M30	139 200	130 500	113 100	400	40	40
M30×2	153 600	144 000	124 800	108	16	12
M33	172 000	161 250	139 750	122	18	4.4
M33×2	188 800	177 000	153 400			14
M36	202 400	189 750	164 450	136	21	10
M36×3	214 400	201 000	174 200			16
M39	242 400	227 250	196 950	150		40
M39×3	255 200	239 250	207 350	150	23	18

The clamp force for property class 5 nuts is equal to 80 % of the proof load of property class 5.8 bolts for 3 mm $\leq d \leq$ 24 mm, and 80 % of the proof load of property class 4.8 bolts for d > 24 mm. Proof loads for bolts are given in ISO 898-1.

b See Annex B.

 $^{^{\}rm C}$ The value of the upper limit of the clamp force is equal to 75 % of the proof load, see Annex B.

d The value of the lower limit of the clamp force is equal to 65 % of the proof load, see Annex B.

e The prevailing torques for first assembly apply for all metal type nuts only. For prevailing torque non-metallic insert type nuts, the maximum torques shall be 50 % of the values.

f Values in this table are required for testing performed under laboratory acceptance test conditions. Utilization of this type of fastener is application dependent and performance for parts may vary in normal use. It is recommended that additional testing of complete joints, using production components, be performed when there are questions of product performance.

Table 4 — Test clamp force and prevailing torques for prevailing torque type nuts of property class 6

		Clamp force for evaluation of total friction coefficient $\mu_{\mathrm{tot}}^{\ \ \mathrm{b}}$		Prevailing torque		
Thread	Test clamp force			N·m		
d P	F_{80}^{a}	Upper limit	Lower limit	1st installation	1at ramaval	5th removal
	N	F ₇₅ ^c	$F_{65}{}^{d}$		1st removal	
		N	N	T _{Fv,max} e	$T_{Fd,min}^{\qquad \qquad f}$	$T_{Fd,min}^{\qquad \qquad f}$
M3	1 768	1 658	1 437	0,43	0,12	0,08
M4	3 088	2 895	2 509	0,9	0,18	0,12
M5	5 000	4 688	4 063	1,6	0,29	0,2
M6	7 072	6 630	5 746	3	0,45	0,3
M7	10 160	9 525	8 255	4,5	0,65	0,45
M8	12 880	12 075	10 465		0.05	0.0
M8×1	13 760	12 900	11 180	6	0,85	0,6
M10	20 400	19 125	16 575			
M10×1,25	21 520	20 175	17 485	10,5	1,5	1
M10×1	22 720	21 300	18 460			
M12	29 680	27 825	24 115			
M12×1,5	31 040	29 100	25 220	15,5	2,3	1,6
M12×1,25	32 400	30 375	26 325			
M14	40 480	37 950	32 890	0.4	3,3	0.0
M14×1,5	44 000	41 250	A 35 750	EV ²⁴ EW		2,3
M16	55 280	51 825	44 915	•) 22	4,5	2
M16×1,5	58 800	55 125	aru _{47,775} en.	ai) ³²		3
M18	67 600	63 375	54 925	40	0	4.0
M18×1,5	76 000	71 250 <u>IS</u>	O 232612750	42	6	4,2
M20	86 400ps://star	dards.i 8 1h000atalog/	standar 7 0/ 2 00/21396	7cb-43c <u>7</u> -466f-84	45- 7,5	F 2
M20×1,5	96 000	90 0006759d()7c48/i 7 8-0000-200	3 54		5,3
M22	106 400	99 750	86 450	00	0.5	0.5
M22×1,5	116 800	109 500	94 900	68	9,5	6,5
M24	124 000	116 250	100 750	80	11,5	0
M24×2	135 200	126 750	109 850	80		8
M27	161 600	151 500	131 300	0.4	13,5	40
M27×2	174 400	163 500	141 700	94		10
M30	197 600	185 250	160 550	400	16	40
M30×2	218 400	204 750	177 450	108		12
M33	244 000	228 750	198 250	400		4.4
M33×2	268 000	251 250	217 750	122	18	14
M36	287 200	269 250	233 350	400	21	10
M36×3	304 800	285 750	247 650	136		16
M39	343 200	321 750	278 850	450	23	40
M39×3	362 400	339 750	294 450	150		18

The clamp force for property class 6 nuts is equal to 80 % of the proof load of property class 6.8 bolts. Proof loads for bolts are given in ISO 898-1.

b See Annex B.

The value of the upper limit of the clamp force is equal to 75 % of the proof load, see Annex B.

d The value of the lower limit of the clamp force is equal to 65 % of the proof load, see Annex B.

e The prevailing torques for first assembly apply for all metal type nuts only. For prevailing torque non-metallic insert type nuts, the maximum torques shall be 50 % of the values.

f Values in this table are required for testing performed under laboratory acceptance test conditions. Utilization of this type of fastener is application dependent and performance for parts may vary in normal use. It is recommended that additional testing of complete joints, using production components, be performed when there are questions of product performance.