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Designation: D 4778 – 94 (Reapproved 1999)^{e1} An American National Standard

Standard Test Method for Determination of Corrosion and Fouling Tendency of Cooling Water Under Heat Transfer Conditions1

This standard is issued under the fixed designation D 4778; the number immediately following the designation indicates the year of original adoption or, in the case of revision, the year of last revision. A number in parentheses indicates the year of last reapproval. A superscript epsilon (ϵ) indicates an editorial change since the last revision or reapproval.

 ϵ^1 Nore—A footnote was editorially removed in July 1999.

1. Scope

1.1 This test method provides directions for fabricating and operating a test apparatus to simultaneously monitor the corrosion and fouling tendency of real and pilot cooling water systems under heat transfer conditions.

1.2 Interpretation of the results of this test method must be left to the investigator. Many variables are involved which may not be easily controlled or fully understood. Variations in design and operating conditions may produce results that are not comparable from unit to unit.

1.3 The values stated in inch-pound units are to be regarded as the standard. The values given in parentheses are provided for information only.

1.4 *This standard does not purport to address all of the safety concerns, if any, associated with its use. It is the responsibility of the user of this standard to establish appropesponsibility of the user of this standard to establish appro-*
priate safety and health practices and determine the applica-
Commission Previewable Previewable Previewable Previewable Previewable Previewable Previewab *bility of regulatory limitations prior to use.*

2. Referenced Documents

- D 1129 Terminology Relating to Water²
- D 1193 Specification for Reagent Water²
- D 2777 Practice for Determination of Precision and Bias of Applicable Methods of Committee D-19 on Water2

3. Terminology

3.1 *Definitions*—For definitions of terms used in this test method, refer to Terminology D 1129.

3.2 *Definitions of Terms Specific to This Standard:*

3.2.1 *corrosion*— the deterioration of the metal by reaction with its environment.

3.2.2 *fouling*— deposition of organic matter or inorganic matter, or both, on heat transfer surfaces that result in the loss of heat transfer efficiency.

² *Annual Book of ASTM Standards*, Vol 11.01.

3.2.3 *heat flux*— heat transfer per unit area per unit time.

4. Summary of Test Method

4.1 Water from the system to be tested flows across a heated tube of the desired metallurgy at a constant flow rate and heat flux. Corrosion rate is determined by weight loss while fouling tendency is determined by the deposit weight.

5. Significance and Use

5.1 Deposits on heat transfer surfaces reduce efficiency of the heat exchanger affected. A method for easily determining the corrosion and fouling tendency of a particular water under
 it is a corrosion and fouling tendency of a particular water under
 it is a condition will allow the avaluation of abandaries in heat transfer conditions will allow the evaluation of changes in the various system variables such as heat flux, flow velocity, purport to address all of the $\frac{1}{2}$ the various system variables such as heat flux, flow velocity, integral with its use It is the metallurgy, cycles-of-concentration, and treatment schemes on heat exchanger performance.

6. Apparatus (Fig. 1)

h2.1 *ASTM Standards:* ai/catalog/standards/sist/70ceaaec-b08ly accommodate the cartridge heater. The tube should be cu 6.1 *Test Specimen*— A metal tube of 3⁄8 or 1⁄2 in. (9.5 or 12.5 ASTM D4778-mm) outside diameter with sufficient inside diameter to snuggly accommodate the cartridge heater. The tube should be cut to a length sufficient to extend $\frac{1}{2}$ in. (12.5 mm) from each end of the test assembly. If both corrosion and deposition are to be determined, metallurgy of the test specimen should match that of the heat exchanger being modeled.

> 6.2 *Cartridge Heater*— A 1⁄4 or 3⁄8 in. (6.2 or 9.5 mm) diameter. Heated surface should be 4 to 8 in. (10 to 20 cm) long with a minimum power rating sufficient to provide 110 % of the heat load required (see Eq 7, 8.2.2). The heater should have an unheated section of sufficient length to allow the center of the heated section to be placed consistently in the center of the test specimen.

> 6.3 *Power Controller*— A device to set and control the power to the heater, such as a variable transformer, is used to adjust the heat flux in order to maintain the surface temperature of the test specimen consistent with the heat exchanger being modeled. The power controller should be rated to maintain at least 120 %, but not more than 400 % of the power required.

> 6.4 *Flow Control*— A flow meter or a flow control device such as an orifice, or both, is recommended to maintain a consistent flow rate during the test period.

¹ This test method is under the jurisdiction of ASTM Committee D19 on Water and is the direct responsibility of Subcommittee D19.03 on Sampling of Water and Water-Formed Deposits, Surveillance of Water, and Flow Measurement of Water.

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6.5 *Safety Equipment*— A pressure or flow sensor/controller is necessary to cut power to the heater in the event of a flow interruption. A high temperature cutoff is recommended for added protection.

6.6 *Test Assembly*— See Fig. 2.

7. Reagents and Materials

7.1 *Purity of Reagents*—Reagent grade chemicals shall be used in all tests. Unless otherwise indicated, it is intended that all reagents conform to the specifications of the Committee on Analytical Reagents of the American Chemical Society where such specifications are available.³ Other grades may be used, provided it is first ascertained that the reagent is of sufficiently high purity to permit its use without lessening the accuracy of the determinations.

7.2 *Purity of Water*— Unless otherwise indicated, references to water shall be understood to mean reagent water as defined by Type III of Specification D 1193.

³ "Reagent Chemicals, American Chemical Society Specifications," Am. Chemical Soc., Washington, DC. For suggestions on the testing of reagents not listed by the American Chemical Society, see "Analar Standards for Laboratory Chemicals,'' BDH Ltd., Poole, Dorset, U. K., and the "United States Pharmacopeia.''

7.3 *Acetone*.

7.4 *Hydrochloric Acid* (HCl), Inhibited.

- 7.5 *Isopropyl Alcohol* (C_3H_8O).
- 7.6 *Trichloroethylene*.

7.7 *Trisodium Phosphate* —(Na₃PO₄·12H₂O), also available as Na ${}_{3}PO_{4}$ ${}_{3}H_{2}O$. Either grade is satisfactory.

8. Procedure

8.1 *Installation of Test Device*:

8.1.1 Placement of the test device with respect to the cooling water system is an important factor in monitoring fouling and corrosion in interpreting the test results. Fouling and corrosion are both affected by temperature. In the case of corrosion, the higher the water temperature, the greater will be the corrosivity of the water. Fouling, however, is a far more complex phenomenon, involving one or more of several types of foulants, namely, particulate matter, precipitates, biomass, corrosion products, and contamination. There are five phases involved in the fouling phenomenon: initiation, attachment, removal, transport, and aging.

8.1.2 Several of the foulants are temperature sensitive. Precipitates, such as calcium carbonate, tend to precipitate more rapidly as temperatures increase. Most biomasses, on the other hand, would agglomerate more rapidly at temperatures between 90 and 105°F (32.2 and 40°C).

tween 90 and 105°F (32.2 and 40°C).

8.1.3 The test device may be installed to take its inlet water $\frac{d_I}{d_I}$ = outside di from one of three locations: cold water supply to a heat exchanger, a heat exchanger outlet, or warm water return to the
cooling tower. The choice of location is a function of the type $W = 9.8 (T_s - T_b)V^{0.8}L(1$ cooling tower. The choice of location is a function of the type of fouling problem(s) experienced with the particular system. No matter where it is placed, the fouling conditions in the test device should simulate the plant equipment as closely as possible. Specifically, the surface or interface temperature and the shear stress of the water film against the heated surface in $\frac{8}{V}$ $\frac{1}{2}$ water the test device should be the same as in the plant equipment being monitored. the particular system.

conditions in the test $W =$ power supplied to heater, W,
 $W =$ power supplied to heater, W, he test device should be the same as in the plant equipment-b058-4 \pm 6 length of heater section, ft, and-d4778-941999e1

8.2 *Determination of Setup Conditions*:

8.2.1 Calculate plant heat exchange conditions as follows:

$$
V_p = 0.408 (F_p) P / (D^2 N)
$$

where:
 $V =$

= water velocity, ft/s,

 $\frac{p}{F}$ = process,
 $\frac{p}{F}$ = water flo

= water flow rate, gal/min,

P = number of passes,

 $D =$ number of tubes in process heat exchanger, in., and

 $N =$ number of tubes in process heat exchanger.

$$
(q/A)_p = 1910 (T_o - T_i) (F_p)/(DLN)
$$

where:

 q/A = heat flux on inner tube, Btu/h/ft²,

- $\frac{p}{T}$ = process,
 $\frac{p}{T}$ = temperat = temperature,° F,
- = outlet water,

ⁱ = inlet water,

- $F =$ water flow rate, gal/min,
- $D =$ inside diameter of tube in process heat exchanger, in.,
- *L* = length of heater section, ft, and
- *N* = number of tubes in process heat exchanger.

8.2.2 Calculate the test device setup as follows:

$$
V_t = [V_p] [D/(d_2 - d_1)]
$$
\n(3)

where:

 $V =$ water velocity, ft/s,

 $t = \text{test}$ device,
= process

 $p \nvert p$ = process,
 $D = \text{inside di}$

= inside diameter of tube in process heat exchanger, in.,

 d_2 = inside diameter of outer tube in test device, in., and d_1 = outside diameter of inner (heated) tube in test device.

outside diameter of inner (heated) tube in test device, in.

$$
F_t = 2.45 V_t (d_2^2 - d_1^2)
$$
\n(4)

where:

 $F =$ water flow rate, gal/min,

 $t =$ test device,
 $V =$ water veloc

= water velocity, ft/s,

 d_2 = inside diameter of outer tube in test device, in., and d_1 = outside diameter of inner (heated) tube in test device, ¹ = outside diameter of inner (heated) tube in test device, in.

$$
W = 9.8 (T_s - T_b)V^{0.8}L (1 + 0.096 T_b) \text{ (for } d_1 = 0.50)
$$
 (5)

where:

 $T =$ temperature, ${}^{\circ}$ F,

 $=$ surface or interface,

 $=$ bulk water, and

 $=$ water velocity, ft/s,

 d_1 = outside diameter of inner (heated) tube in test device,

$$
W = 7.94 (T_s - T_b) V^{0.8} L(1 + 0.096 T_b) \text{ (for } d_1 = 0.375) \tag{6}
$$

where:

W = power supplied to heater, W,

 $T =$ temperature, ${}^{\circ}$ F,

- *^s* = surface or interface,
- $\begin{array}{ll} b & = \text{bulk water}, \\ V & = \text{water velocity}. \end{array}$

in.

- $=$ water velocity, ft/s,
- $L =$ length of heater section, ft, and $d_1 =$ outside diameter of inner (heater
- outside diameter of inner (heated) tube in test device, in.

$$
E = (WR)^{0.5}
$$

(7)

where:

 E = voltage of heater, V,

- *W* = power supplied to heater, W, and
- *R* = resistance of heater, ohm.
- 8.3 *Preparation of Test Specimen*:

(2)

(1)