## INTERNATIONAL STANDARD

ISO 13050

Second edition 2014-12-01 Corrected version 2016-04-01

# Synchronous belt drives — Metric pitch, curvilinear profile systems G, H, R and S, belts and pulleys

Transmissions synchrones — Pas métrique, systèmes à denture curviligne G, H, R et S, courroies et poulies

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Co	ntent	SS .	Page					
For	eword		v					
1	Scop	e	1					
2	Norn	native references	1					
3		hronous belt drive system types						
4		nomenclature						
_								
5		ey nomenclature						
6		Profile system G						
	6.1	Belt dimensions and tolerances						
		6.1.1 Belt tooth dimensions Belt widths and tolerances						
		6.1.3 Pitch length measurement						
	6.2	Pulleys, profile system G						
	0.2	6.2.1 General						
		6.2.2 Generating tool rack						
		6.2.3 Pulley groove profile dimensions and tolerances						
		6.2.4 Pulley outside diameters						
		6.2.5 Pulley width	12					
7	Prof	ile system H	13					
-	7.1	Belt dimensions and tolerances A.R.D. P.R.E.V. E.V.	13					
		7.1.1 Belt tooth dimensions	13					
,		7.1.1 Belt tooth dimensions 7.1.2 Belt widths and tolerances S. Iteh. a. 1	14					
		7.1.3 Pitch length measurement	15					
		7.1.4 Procedure <u>ISO 13050 2014</u>	15					
	7.2	Pulleys, profile system; Httalog/standards/sist/8f72d0fc-1d4d-4488-b707						
		7.2.1 General d494bcc580fe/iso-13050-2014	17					
		7.2.2 Generating tool rack						
		7.2.3 Pulley groove profile dimensions and tolerances	19					
		7.2.5 Pulley width						
_								
8		ile system R						
	8.1	Belt dimensions and tolerances 8.1.1 Belt tooth dimensions						
		8.1.1 Belt tooth dimensions 8.1.2 Belt widths and tolerances						
		8.1.3 Pitch length measurement						
	8.2	Pulleys, profile system R						
	0.2	8.2.1 General						
		8.2.2 Generating tool rack						
		8.2.3 Pulley groove profile dimensions and tolerances	31					
		8.2.4 Pulley outside diameters						
		8.2.5 Pulley width	35					
9	Prof	ile system S	38					
	9.1	Belt dimensions and tolerances						
		9.1.1 Belt tooth dimensions	38					
		9.1.2 Belt widths and tolerances						
		9.1.3 Pitch length measurement						
	9.2	Pulleys, profile system S						
		9.2.1 General						
		9.2.2 Generating tool rack						
		<ul><li>9.2.3 Pulley groove profile dimensions and tolerances</li><li>9.2.4 Pulley outside diameters</li></ul>						
		9.2.5 Pulley width						
		·						

## ISO 13050:2014(E)

Annex A (normative) Belt pitch lengths and tolerances	47
Annex B (normative) Relationship between centre distance and belt pitch length	49
Annex C (normative) Pulley tolerances	50
Annex D (normative) Flange dimensions	53
Bibliography	54

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ISO 13050:2014 https://standards.iteh.ai/catalog/standards/sist/8f72d0fc-1d4d-4488-b707-d494bce580fe/iso-13050-2014

## Foreword

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The procedures used to develop this document and those intended for its further maintenance are described in the ISO/IEC Directives, Part 1. In particular the different approval criteria needed for the different types of ISO documents should be noted. This document was drafted in accordance with the editorial rules of the ISO/IEC Directives, Part 2 (see <a href="www.iso.org/directives">www.iso.org/directives</a>).

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For an explanation on the meaning of ISO specific terms and expressions related to conformity assessment, as well as information about ISO's adherence to the WTO principles in the Technical Barriers to Trade (TBT) see the following URL: Foreword - Supplementary information

The committee responsible for this document is ISO/TC 41, Belts and pulleys (including veebelts), Subcommittee SC 4, Synchronous belt drives.

ISO 13050:2014

This second edition cancels and replaces the first ledition (ISO 13050:1999), which has been technically revised.

This corrected version of ISO 13050:2014 incorporates the following corrections: in Formulas (1) and (2), "Ao" has been added before the first square bracket in each equation.

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## Synchronous belt drives — Metric pitch, curvilinear profile systems G, H, R and S, belts and pulleys

## 1 Scope

This International Standard specifies the principal characteristics of metric pitch curvilinear synchronous endless belts and pulleys in G, H, R, and S profile systems for use in synchronous belt drives (also known in the past as timing belt drives, positive belt drives, gear belt drives) for mechanical power transmission and where positive indexing or synchronization might be required.

The principal belt and pulley characteristics include the following:

- a) nominal belt tooth dimensions;
- b) belt tooth pitch spacing;
- c) belt length and width dimensions and tolerances;
- d) belt length measurement specifications;
- e) pulley groove dimensions and tolerances;
- f) pulley diameter and width dimensions and tolerances; (standards.iteh.ai)
- g) pulley quality specification.

ISO 13050:2014

## 2 Normative references.iteh.ai/catalog/standards/sist/8f72d0fc-1d4d-4488-b707-d494bce580fe/iso-13050-2014

The following documents, in whole or in part, are normatively referenced in this document and are indispensable for its application. For dated references, only the edition cited applies. For undated references, the latest edition of the referenced document (including any amendments) applies.

ISO 254, Belt drives — Pulleys — Quality, finish and balance

## 3 Synchronous belt drive system types

Four profile systems and 14 profiles for curvilinear synchronous drives are standardized.

## Profile system G

Profile system R

Profile G8M

(Tooth/groove pitch 8mm)

Profile G14M

(Tooth/groove pitch 14mm)

Profile R3M

(Tooth/groove pitch 3mm)

Profile R5M

(Tooth/groove pitch 5mm)

Profile R8M

(Tooth/groove pitch 8mm)

Profile R14M

(Tooth/groove pitch 14mm)

Profile R20M

(Tooth/groove pitch 20mm)

### Profile system H

Profile H3M

(Tooth/groove pitch 3mm)

Profile H5M

(Tooth/groove pitch 5mm)

Profile H8M

(Tooth/groove pitch 8mm)

Profile H14M

(Tooth/groove pitch 14mm)

Profile H20M

(Tooth/groove pitch 20mm)

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**Profile system S** 

Profile S8M

(Tooth/groove pitch 8mm)

Profile S14M

(Tooth/groove pitch 14mm)

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### 4 Belt nomenclature

A belt is identified as follows:

- a) the belt length or pitch length in millimetres, e.g. 1 400 mm;
- b) the profile system, e.g. "G", "H", "R", or "S";
- c) the pitch or tooth pitch in millimetres, e.g. 14 mm;
- d) the width or belt width in millimetres, e.g. 40 mm (for profile system S, the width in millimetres × 10);
- e) double-sided belts are designated by adding the letter "D" before the designation of the profile system.

EXAMPLE A synchronous belt in the above profile systems of 1 400 mm length, 14 mm pitch, and 40 mm wide is identified as follows:

Profile system G	Profile system R
1400-G14M-40	1400-R14M-40
1400-DG14M-40	1400-DR14M-40

 Profile system H
 Profile system S

 1400-H14M-40
 1400-S14M-400

 1400-DH14M-40
 1400-DS14M-400

## 5 Pulley nomenclature STANDARD PREVIEW

A pulley is identified as follows: (standards.iteh.ai)

- a) the letter "P" indicates a pulley; ISO 13050:2014
- b) the number of grooves, e.g. 30, ai/catalog/standards/sist/8f72d0fc-1d4d-4488-b707-d494bce580fe/iso-13050-2014
- c) the profile system, e.g. "G", "H", "R", or "S";
- d) the pitch or groove pitch in millimetres, e.g. 14 mm;
- e) the width or belt width in millimetres, e.g. 40 mm (for profile system S, the width in millimetres × 10).

EXAMPLE A pulley in the above profile systems of 30 grooves, 14 mm pitch, and 40 mm wide is identified as follows:

Profile system GProfile system RP30-G14M-40P30-R14M-40

Profile system HProfile system SP30-H14M-40P30-S14M-400

## 6 Profile system G

## 6.1 Belt dimensions and tolerances

## 6.1.1 Belt tooth dimensions

The nominal belt tooth dimensions are given in <a>Table 1</a> and shown in <a>Figure 1</a>.

Table 1 — Nominal tooth dimensions

Dimensions in millimetres

Profile	Belt pitch	bg	hg	R1	Ao	<b>a</b> a	xo	L1
G8M	8	5,200	3,43	0,74	2,29	0,80	0,379	1,378
G14M	14	9,100	6,00	1,30	4,00	1,40	0,661	2,422
a a is the belt design pitch differential.								

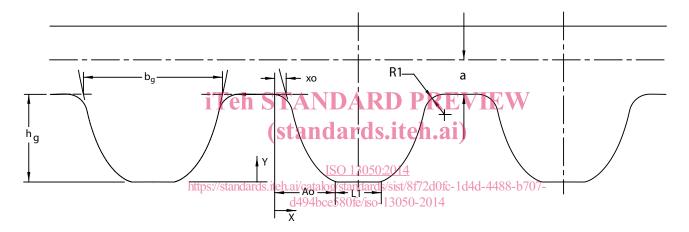


Figure 1 — Tooth dimensions — Profile system G

#### Detail of tooth face

For  $Ao \ge x \ge xo$ :

$$y = Ao \left[ ln \left( Ao / x + \sqrt{\left( Ao / x \right)^2 - 1} \right) - \sqrt{1 - \left( x / Ao \right)^2} \right]$$
 (1)

where

*xo* corresponds to y = hg;

L1 is equal to bg - 2(Ao - xo).

NOTE 1 Indicated base tooth profile remains constant in all parts.

NOTE 2 "0" reference points remain in contact with the part outer surface generated by part radius of curvature.

NOTE 3 Groove profile bottom surface in circular part form is an arc whose chordal distance is L1 and whose radius originates at the part centre.

#### 6.1.2 Belt widths and tolerances

Belt widths and tolerances are given in <u>Table 2</u>.

## iTeh Table 2 Widths and width tolerances

(standards.iteh.ai)

Dimensions in millimetres

	Tolerance on width for belt pitch lengths							
Profile ht	Nominal belt width ps://standards.iich.a/catalo d494bce	<u>SO 13050:2014</u> y <b>Up,to, and including</b> 580fe/is8 <b>-13</b> 030 <b>-2</b> 014	Over 840 mm and up to and including 1 680 mm	0ver 1 680 mm				
	12	+0,8 -0,8	+0,8 -1,2	+0,8 -1,2				
G8M	21	+0,8 -0,8	+0,8 -1,2	+0,8 -1,2				
GOM	36	+0,8 -0,8	+0,8 -1,2	+0,8 -1,2				
	62	+1,2 -1,2	+1,2 -1,6	+1,6 -1,6				
	20	+0,8 -0,8	+0,8 -1,2	+0,8 -1,2				
	37	+0,8 -0,8	+0,8 -1,2	+0,8 -1,2				
G14M	68	+1,2 -1,6	+1,6 -1,6	+1,6 -2,0				
	90	+1,6 -1,6	+1,6 -2,0	+2,0 -2,0				
	125	+2,4 -2,4	+2,4 -2,8	+2,4 -3,2				

## 6.1.3 Pitch length measurement

See Annex A for tolerances and Annex B for the relationship between the centre distance and the belt pitch length.

### 6.1.3.1 Measuring fixture (see Figure 3)

The pitch length of a synchronous belt shall be determined by placing the belt on a measuring fixture composed of the following elements.

**6.1.3.1.1 Two pulleys of equal diameter**, as specified in <u>Table 3</u>, of the proper belt profile and having standard tooth space dimensions. These pulleys should be made to the tolerances shown in <u>Table 3</u>. One pulley shall be free to rotate on a fixed-position shaft, while the other shall be free to rotate on a moveable shaft to permit the centre distance to change.

### **6.1.3.1.2 Means of applying a total measuring force** to the moveable pulley.

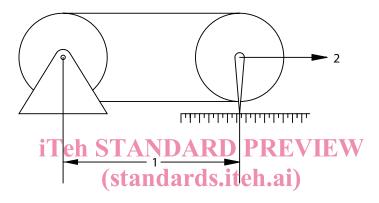
**6.1.3.1.3 Means of measuring the centre distance** between the two pulleys with the necessary degree of accuracy for centre distance measurement.

### **6.1.3.2** Total measuring force

The total measuring force to be applied for measuring belts is given in Table 4.

#### 6.1.3.3 Procedure

In measuring the pitch length of a synchronous belt, the belt should be rotated at least two revolutions to seat it properly and to divide the total force equally between the two spans of the belt. The pitch length shall be calculated by adding the pitch circumference of one of the pulleys to twice the measured centre distance.



### Key

- 1 centre distance
- 2 total measuring force

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Figure 2 — Diagram of fixture for measuring pitch length

Table 3 — Belt length measuring pulleys

Dimensions in millimetres

Profile	Number of grooves	Pitch circumfer- ence	Outside diame- ter <sup>a</sup>	Radial runout F.I.M. <sup>b</sup>	Axial runout F.I.M. <sup>b</sup>	
G8M	34	272	84,980	0,013	0,025	
dol			±0,013	0,015		
C14M	40	560	175,454	0.012	0,051	
G14M			±0,025	0,013		

<sup>&</sup>lt;sup>a</sup> Pulleys outside of the diameter tolerance range specified can be used if the resulting belt length measurements are corrected for the actual pulley diameters.

NOTE The number of pulley teeth specified in <u>Table 3</u> determines the recommended sizes for measuring the belt pitch length. Practically, other sizes of pulleys could be used provided they have the same number of teeth, and meet the dimensional requirements of <u>Table 3</u>.

b Full indicator movement.

Table 4 — Total measuring force

Forces in newtons

Profile	Belt width e mm								
	12	20	21	36	37	62	68	90	125
G8M	267	_	467	756	_	1223	_	_	_
G14M	_	1179	_	_	2046	_	3447	4315	5627

## 6.2 Pulleys, profile system G

#### 6.2.1 General

See Annex C for tolerances.

The pulley is characterized by a curvilinear groove profile. This groove profile is defined as the profile formed by the generating tool rack form required to machine-finish the curvilinear profile. The profile is different for each pulley diameter, but can be closely approximated by a nominal groove profile over specified ranges of number of grooves.

## 6.2.2 Generating tool rack

Dimensions and tolerances for the generating tool rack for pulleys with profile system G are given in Table 5 and shown in Figure 3. STANDARD PREVIEW

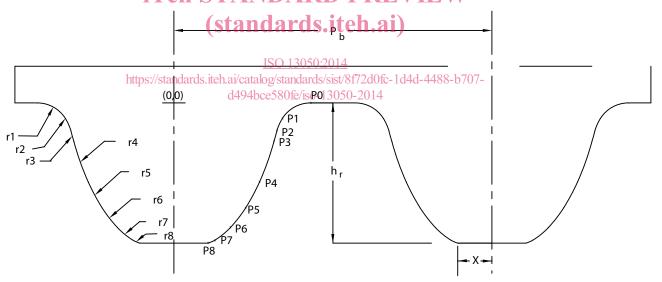


Figure 3 — Pulley generating tool rack dimensions

Dimensions in millimetres

Table 5 — Pulley generating tool rack dimensions

1,485 1,458 0,856 0,843 0,820 1,505 1,476 0,831 0,951 1,467 -3,513-3,4520,843 0,831 0,820 1,496 1,485 1,467 1,458 1,453 0,856 0,951 1,476 0,6400,805 1,511 1,588 1,672 1,812 1,872 1,908 1,745 -5,8991,940 1,009 -3,3841,808 1,850 1,913 1,969 0,979 1,024 1,039 1,181 1,883 1,983 1,562 3,586 1,505 1,614 1,725 3,489 3,541  $\mathbf{r}_7$ -3,072-5,4891,367 1,410 1,443 1,559 2,486 2,522 2,582 2,649 -5,267 2,702 2,746 2,701 3,205 3,042 6,236 6,245 6,249 3,123 3,152 3,181 6,129 6,163 6,222 **L**9 1,940 3,213 3,505 1,894 1,978 1,978 3,161 3,297 3,417 3,463 3,533 1,841  $P_5$ 10,051 10,092 10,067 10,115 5,882 9,625 10,051 5,111 9,584 5,901 5,921 5,932  $\mathbf{r}_{\mathbf{2}}$ 4,014 4,030 -1,2752,462 3,800 3,988 2,447 2,345 3,768 2,427 3,951 3,971  $P_4$ 13,0<mark>9</mark> 13,150 11,867 8 13,051 **S**1 8,833 9,487 9,964 **1**4 -F626 454 185 185 4,405 2,617  $P_3$ https: /standa rds/sis 8f72d0fc-1d4d-4488-b707-1,984 9,448  $\mathbf{r}_3$ 1 I -1,312-1,313-0,668-0,512-1,244-1,2852,578 2,583 2,586 2,751 4,536 4,519 4,490 4,484 4,484 4,476 2,567  $\mathbf{P}_2$ 0,846 0,918 2,188 2,206 1,086 0,957 2,241 2,180 2,586 -0,815-0,883-0,348-0,2323,040 -0,164-0,684-0,7174,685 4,608 4,568 2,710 2,746 2,829 2,943 4,738 4,652 4,777 4,537  $P_1$ 0,828 0,819 0,818 0,846 1,437 5,9190 3,448 3,426 3,412 600'9 5,940 5,900 5,887 3,480 3,558 5,986 5,961  $\mathbf{P}_0$ 6,065 6,040 6,024 3,513 3,487 3,452 3,467 3,541 hr 13,956 13,780 13,674 13,831 7,784 7,843 7,930 8,040 7,891  $\mathbf{P}_{\mathbf{b}}$ Number of grooves 150 to 250 91 to 149 61 to 90 28 to 37 38 to 58 90 to 250 27 to 29 36 to 45 46 to 60 22 to 27 59 to 89 30 to 35

**G8M** 

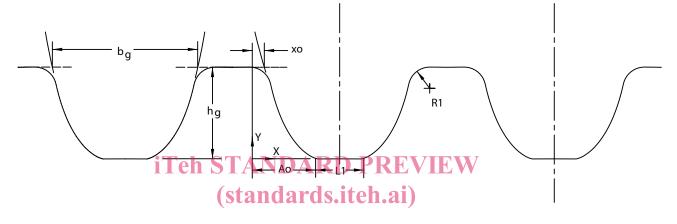
## 6.2.3 Pulley groove profile dimensions and tolerances

Dimensions and tolerances for the pulley groove profile for G8M and G14M pulleys are given in <u>Table 6</u> and shown in <u>Figures 4</u> and <u>5</u>.

Table 6 — Pulley groove profile dimensions

Dimensions in millimetres

Profile	Number of grooves	$\mathbf{b}_{\mathrm{g}}$	$\mathbf{h}_{\mathrm{g}}$	$\mathbf{R}_1$	Ao	xo	L1	a
G8M	22 to 89	5,400	3,43	0,80	2,29	0,379	1,60	0,80
	90 to 250	5,660	3,43	0,80	2,29	0,379	1,60	
G14M	28 to over	9,450	6,00	1,40	4,00	0,661	2,80	1,40



## Figure 4 Pulley groove profile

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For  $Ao \ge x \ge xo$ :

$$y = Ao \left[ In \left( Ao / x + \sqrt{\left( Ao / x \right)^2 - 1} \right) - \sqrt{1 - \left( x / Ao \right)^2} \right]$$
 (2)

where

*xo* corresponds to y = hg;

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NOTE 2 "0" Reference points remain in contact with the part outer surface generated by part radius of curvature.

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