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Dimniki - Računske metode termodinamike in dinamike fluidov - 1. del: Dimniki za eno ogrevalno napravo

Chimneys - Thermal and fluid dynamic calculation methods - Part 1: Chimneys serving one heating appliance

Abgasanlagen - Wärme- und strömungstechnische Berechnungsverfahren - Teil 1: Abgasanlagen mit einer Feuerstätte

Conduits de fumée - Méthodes de calcul thermo-aéraulique - Partie 1: Conduits de fumée ne desservant qu'un seul appareil

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ICS:

91.060.40	Dimniki, jaški, kanali	Chimneys, shafts, ducts
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**Chimneys - Thermal and fluid dynamic calculation
methods - Part 1: Chimneys serving one heating appliance**

Conduits de fumée - Méthodes de calcul thermo-
aéraulique - Partie 1: Conduits de fumée ne desservant
qu'un seul appareil

Abgasanlagen - Wärme- und strömungstechnische
Berechnungsverfahren - Teil 1: Abgasanlagen mit einer
Feuerstätte

This European Standard was approved by CEN on 24 January 2015 and includes Amendment 1 approved by CEN on 27 April 2019.

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European foreword

This document (EN 13384-1:2015+A1:2019) has been prepared by Technical Committee CEN/TC 166 “Chimneys”, the secretariat of which is held by ASI.

This European Standard shall be given the status of a national standard, either by publication of an identical text or by endorsement, at the latest by January 2020, and conflicting national standards shall be withdrawn at the latest by January 2020.

Attention is drawn to the possibility that some of the elements of this document may be the subject of patent rights. CEN [and/or CENELEC] shall not be held responsible for identifying any or all such patent rights.

This document supersedes A1 EN 13384-1:2015 A1.

This document includes Amendment 1 approved by CEN on 26 June 2019.

According to EN 13384-1:2002+A2:2008 and EN 13384-1:2015+A1:2019 the following fundamental changes are given:

- editorial mistakes have been corrected;
- mistakes in formulas have been corrected;
- for wood the rise of the dew point to take into account the acid condensation has been deleted;
- table for material characteristics in Table B.5 has been adapted to EN 15287-1 and supplemented by radiation coefficients;
- in Calculation of thermal resistance according to Annex A are linked to the method of EN 15287-1 for taking into account the temperature dependence has been added;
- for non-concentric ducts the calculation of the mean temperature of the air supply has been amended;
- for chimney fans a calculation procedure has been added;
- “heating appliance” replaced by “combustion appliance”;
- New calculation for combustion air mass flow introduced;
- “Supply air” replaced by “combustion air”.

This document has been prepared under a mandate given to CEN by the European Commission and the European Free Trade Association.

The start and finish of text introduced or altered by amendment is indicated in the text by tags A1 A1.

This European Standard “Chimneys — Thermal and fluid dynamic calculation methods” consists of three Parts:

- Part 1: Chimneys serving one combustion appliance

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- Part 2: Chimneys serving more than one combustion appliance
- Part 3: Methods for the development of diagrams and tables for chimneys serving one heating appliance

According to the CEN-CENELEC Internal Regulations, the national standards organisations of the following countries are bound to implement this European Standard: Austria, Belgium, Bulgaria, Croatia, Cyprus, Czech Republic, Denmark, Estonia, Finland, Former Yugoslav Republic of Macedonia, France, Germany, Greece, Hungary, Iceland, Ireland, Italy, Latvia, Lithuania, Luxembourg, Malta, Netherlands, Norway, Poland, Portugal, Romania, Serbia, Slovakia, Slovenia, Spain, Sweden, Switzerland, Turkey and the United Kingdom.

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1 Scope

This European Standard specifies methods for the calculation of the thermal and fluid dynamic characteristics of chimneys serving one A_1 combustion A_1 appliance.

The methods in this part of this European Standard are applicable to negative or positive pressure chimneys with wet or dry operating conditions. It is valid for chimneys with A_1 combustion A_1 appliances for fuels subject to the knowledge of the flue gas characteristics which are needed for the calculation.

The methods in this part of this European Standard are applicable to chimneys with one inlet connected with one appliance. The methods in Part 2 of this European Standard are applicable to chimneys with multiple inlets and one inlet with multiple appliances. Part 3 describes methods for the development of diagrams and tables for chimneys serving one A_1 combustion A_1 appliance.

2 Normative references

A_1 The following documents are referred to in the text in such a way that some or all of their content constitutes requirements of this document. For dated references, only the edition cited applies. For undated references, the latest edition of the referenced document (including any amendments) applies. A_1

EN 1443, *Chimneys - General requirements*

EN 1856-1, *Chimneys - Requirements for metal chimneys - Part 1: System chimney products*

EN 1859, *Chimneys — Metal chimneys — Test methods*

EN 13502, *Chimneys - Requirements and test methods for clay/ceramic flue terminals*

EN 15287-1:2007+A1:2010, *Chimneys - Design, installation and commissioning of chimneys - Part 1: Chimneys for non-roomsealed heating appliances*

prEN 16475-2, *Chimneys - Accessories - Part 2: Chimney fans - Requirements and test methods*

CEN/TR 1749, *European scheme for the classification of gas appliances according to the method of evacuation of the combustion products (types)*

3 Terms and definitions

For the purposes of this document, the terms and definitions given in EN 1443 and the following apply.

A_1 ISO and IEC maintain terminological databases for use in standardization at the following addresses:

- IEC Electropedia: available at <http://www.electropedia.org/>
- ISO Online browsing platform: available at <http://www.iso.org/obp> A_1

3.1

heat output

Q

amount of heat produced by a A_1 combustion A_1 appliance per unit of time

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3.1.1

nominal heat output Q_N

continuous heat output specified by the manufacturer of the \square_{A1} combustion \square_{A1} appliance related to specified fuels

3.1.2

heat output range

range of output below the nominal heat output specified by the manufacturer over which the appliance can be used

3.2

heat input Q_F

amount of heat in unit time which is supplied to the \square_{A1} combustion \square_{A1} appliance by the fuel based on its net calorific value H_u

3.3

efficiency of the \square_{A1} combustion \square_{A1} appliance η_w

ratio of the heat output (Q) from the appliance to the heat input (Q_F)

3.4

flue gas mass flow \dot{m}

mass of flue gas leaving the \square_{A1} combustion \square_{A1} appliance through the connecting flue pipe per time unit

3.5

effective height of the chimney H

difference in height between the axis of the flue gas inlet into the chimney and the outlet of the chimney

3.6

effective height of the connecting flue pipe H_V

difference in height between the axis of the flue gas chimney outlet of the \square_{A1} combustion \square_{A1} appliance and the axis of the flue gas inlet into the chimney

Note 1 to entry

In the case of open fire chimneys, H_V is the difference in height between the height of the upper frame of the furnace and the axis of the flue gas inlet into the chimney.

3.7

draught

positive value of the negative pressure in the flue

3.8

theoretical draught available due to chimney effect P_H

pressure difference caused by the difference in weight between the column of air equal to the effective height outside a chimney and the column of flue gas equal to the effective height inside the chimney

3.9**pressure resistance of the chimney** P_R

pressure which is necessary to overcome the resistance of the flue gas mass flow which exists when carrying the flue gases through the chimney

3.10**wind velocity pressure** P_L

pressure generated on the chimney due to wind

3.11**minimum draught at the flue gas inlet into the chimney** P_Z

difference between the minimum theoretical draught and the sum of the maximum pressure resistance of the chimney and the wind velocity pressure

3.12**maximum draught at the flue gas inlet into the chimney** P_{Zmax}

difference between the maximum theoretical draught and the minimum pressure resistance in the chimney

3.13**minimum draught for the \square_{A1} combustion \square_{A1} appliance** P_W

difference between the static air pressure of the room of installation of the \square_{A1} combustion \square_{A1} appliance and the static pressure of the flue gas at the chimney outlet of the appliance which is necessary to maintain the correct operation of the \square_{A1} combustion \square_{A1} appliance

3.14**maximum draught for the \square_{A1} combustion \square_{A1} appliance** P_{Wmax}

difference between the static air pressure of the room of installation of the \square_{A1} combustion \square_{A1} appliance and the static pressure of the flue gas at the outlet of the appliance which is the maximum allowed to maintain the correct operation of the \square_{A1} combustion \square_{A1} appliance

3.15**effective pressure resistance of the connecting flue pipe** P_{FV}

static pressure difference between the axis of the inlet of the connecting flue pipe and the axis of the chimney outlet due to the theoretical draught and pressure resistance

3.16**effective pressure resistance of the air supply** P_B

difference between the static pressure in the open air and the static air pressure in the room of installation of the \square_{A1} combustion \square_{A1} appliance at the same height

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3.17

minimum draught required at the flue gas inlet into the chimney

P_{Ze}
sum of the minimum draught required for the $\boxed{A_1}$ combustion $\langle A_1 \rangle$ appliance and the draught required to overcome the effective pressure resistance of the connecting flue pipe and the effective pressure resistance of the air supply

3.18

maximum allowed draught at the flue gas inlet into the chimney

P_{Zemax}
sum of the maximum draught allowed for the $\boxed{A_1}$ combustion $\langle A_1 \rangle$ appliance and the draught required to overcome the effective pressure resistance of the connecting flue pipe and the effective pressure resistance of the air supply

3.19

maximum positive pressure at the flue gas inlet into the chimney

P_{Zo}
difference of the maximum pressure resistance and the minimum theoretical draught of the chimney added by the wind velocity pressure

3.20

minimum positive pressure at the flue gas inlet into the chimney

P_{Z0min}
difference of the minimum pressure resistance and the maximum theoretical draught of the chimney

3.21

maximum differential pressure of the $\boxed{A_1}$ combustion $\langle A_1 \rangle$ appliance

P_{wo}
maximum difference between the static pressure of the flue gas at the chimney outlet of the appliance and the static pressure of the air at the inlet to the $\boxed{A_1}$ combustion $\langle A_1 \rangle$ appliance specified for its correct operation

3.22

minimum differential pressure of the $\boxed{A_1}$ combustion $\langle A_1 \rangle$ appliance

P_{w0min}
minimum difference between the static pressure of the flue gas at the outlet of the appliance and the static pressure of the air at the inlet to the $\boxed{A_1}$ combustion $\langle A_1 \rangle$ appliance specified for its correct operation. This can be a negative value.

3.23

maximum differential pressure at the flue gas inlet into the chimney

P_{Zoe}
difference between the maximum differential pressure of the $\boxed{A_1}$ combustion $\langle A_1 \rangle$ appliance and the sum of the effective pressure resistance of the connecting flue pipe and the effective pressure resistance of the air supply

3.24

secondary air

ambient air added to the flue gas in addition to the nominal flue gas mass flow

3.25**minimum differential pressure at the flue gas inlet into the chimney** P_{zoemin}

difference between the minimum differential pressure of the \square_{A1} combustion \square_{A1} appliance and the sum of the effective pressure resistance of the connecting flue pipe and the effective pressure resistance of the air supply

3.26**secondary air device**

draught regulator or a draught diverter

3.27**draught regulator**

component which automatically supplies ambient air to the chimney, the connecting flue pipe or the \square_{A1} combustion \square_{A1} appliance

3.28**draught diverter**

device, placed in the combustion products passage of the \square_{A1} combustion \square_{A1} appliance, that is intended to maintain the quality of combustion within certain limits and to keep the combustion stable under certain conditions of up draught and down draught

3.29**temperature limit of the inner wall** T_g

allowed minimum temperature of the inner wall of the chimney outlet

3.30**air-supply duct**

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component or components parallel to the chimney (separate or concentric) that conveys combustion air from the outside atmosphere to the inlet of the connecting air supply pipe

3.31**balanced flue chimney**

chimney where the point of air entry to the air supply duct is adjacent to the point of discharge of combustion products from the flue, the inlet and outlet being so positioned that wind effects are substantially balanced

3.32**chimney segment**

calculation part of a chimney

3.33**condensate mass flow** $\Delta \dot{m}_w$

mass of water vapour of the flue gas condensed in the \square_{A1} combustion \square_{A1} appliance, connecting flue pipe or the chimney per time unit

3.34**connecting air supply pipe**

component or components connecting the air supply duct outlet with the room-sealed \square_{A1} combustion \square_{A1} appliance combustion air inlet

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proportion of the theoretical maximum condensation mass flow usable in the calculation

3.36**chimney fan**

exhaust fan or inline fan

3.36.1**exhaust fan**

fan positioned on the outlet of the chimney

3.36.2**inline fan**

fan positioned as a section of the connecting flue

4 Symbols and abbreviations

The symbols given in this clause can be completed by one or more indices to indicate location or materials if necessary.

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Table 1 — Symbols, terminology and units

Symbol	Terminology	Unit
A	cross section area	m^2
c	specific heat capacity	$\text{J}/(\text{kg} \cdot \text{K})$
c_p	specific heat capacity of flue gas	$\text{J}/(\text{kg} \cdot \text{K})$
d	thickness of the section	m
D	diameter	m
D_h	hydraulic diameter	m
E	heat flux ratio	–
g	acceleration due to gravity	m/s^2
H	effective height of the chimney	m
k	coefficient for heat transmission	$\text{W}/(\text{m}^2 \cdot \text{K})$
K	coefficient of cooling	–
L	length	m
l_c	proportion of condensation surface	–
\dot{m}	flue gas mass flow	kg/s
$\dot{m}_{w, \text{red}}$	flue gas mass flow of A_1 combustion appliance reduced by condensed water	kg/s
\dot{m}_{cond}	condensate mass flow	kg/s
N_{u}	Nusselt number	–
N_{seg}	number of segments	–
p	static pressure	Pa
p_L	external air pressure	Pa
P_B	effective pressure resistance of the air supply	Pa
P_E	pressure resistance due to friction and form resistance of the chimney	Pa
P_{FV}	effective pressure resistance of the connecting flue pipe	Pa
P_G	difference in pressure caused by change of velocity of flue gas in the chimney	Pa
P_H	theoretical draught available due to chimney effect	Pa
P_{HV}	theoretical draught available due to chimney effect of the connecting flue pipe	Pa
P_L	wind velocity pressure	Pa
P_{NL}	draught required for secondary air devices	Pa
P_R	pressure resistance of the chimney	Pa
P_{RV}	pressure resistance of the connecting flue pipe	Pa