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## Restrained joint systems for ductile iron pipelines — Calculation rules for lengths to be restrained

*Systèmes d'assemblages verrouillés pour canalisations en fonte  
ductile — Règles de calcul pour les longueurs à verrouiller*

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## Foreword

ISO (the International Organization for Standardization) is a worldwide federation of national standards bodies (ISO member bodies). The work of preparing International Standards is normally carried out through ISO technical committees. Each member body interested in a subject for which a technical committee has been established has the right to be represented on that committee. International organizations, governmental and non-governmental, in liaison with ISO, also take part in the work. ISO collaborates closely with the International Electrotechnical Commission (IEC) on all matters of electrotechnical standardization.

The procedures used to develop this document and those intended for its further maintenance are described in the ISO/IEC Directives, Part 1. In particular, the different approval criteria needed for the different types of ISO documents should be noted. This document was drafted in accordance with the editorial rules of the ISO/IEC Directives, Part 2 (see [www.iso.org/directives](http://www.iso.org/directives)).

Attention is drawn to the possibility that some of the elements of this document may be the subject of patent rights. ISO shall not be held responsible for identifying any or all such patent rights. Details of any patent rights identified during the development of the document will be in the Introduction and/or on the ISO list of patent declarations received (see [www.iso.org/patents](http://www.iso.org/patents)).

Any trade name used in this document is information given for the convenience of users and does not constitute an endorsement.

For an explanation of the voluntary nature of standards, the meaning of ISO specific terms and expressions related to conformity assessment, as well as information about ISO's adherence to the World Trade Organization (WTO) principles in the Technical Barriers to Trade (TBT), see [www.iso.org/iso/foreword.html](http://www.iso.org/iso/foreword.html).

This document was prepared by Technical Committee ISO/TC 5, *Ferrous metal pipes and metallic fittings*, Subcommittee SC 2, *Cast iron pipes, fittings and their joints*.

Any feedback or questions on this document should be directed to the user's national standards body. A complete listing of these bodies can be found at [www.iso.org/members.html](http://www.iso.org/members.html).

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# Restrained joint systems for ductile iron pipelines — Calculation rules for lengths to be restrained

## 1 Scope

This document specifies a computation method used to determine the length of the ductile iron pipes to be restrained, when used for conveying raw water, drinking water, sewerage under pressure.

This computation method takes into account all common pipeline route changes, including changes in the diameter of the pipeline itself and dead ends at the extremity of the pipeline, the outside diameter of the pipe, the system test pressure (to estimate the thrust), depth of cover, the characteristics of the soil surrounding the pipe and trench backfilling methods for a worldwide usage. The characteristics of the restrained joint are not covered by this document but can also be considered to determine the restraining length using any appropriate method.

The computation method defined in this document is applicable to all types of restrained joint systems, with their operating pressure ratings of ductile iron pipelines complying with ISO 2531, ISO 7186 and ISO 16631.

NOTE 1 ISO 10804 deals with actual design of the joint for various operating pressures of the pipeline.

NOTE 2 National standards or established calculation methods can be used instead of this ISO standard.

## 2 Normative references

The following documents are referred to in the text in such a way that some or all of their content constitutes requirements of this document. For dated references, only the edition cited applies. For undated references, the latest edition of the referenced document (including any amendments) applies.

ISO 2531, *Ductile iron pipes, fittings, accessories and their joints for water applications*

ISO 7186, *Ductile iron products for sewerage applications*

ISO 10804, *Restrained joint systems for ductile iron pipelines — Design rules and type testing*

ISO 16631, *Ductile iron pipes, fittings, accessories and their joints compatible with plastic (PVC or PE) piping systems, for water applications and for plastic pipeline connections, repair and replacement*

## 3 Terms, definitions and symbols

### 3.1 Terms and definitions

For the purposes of this document, the terms and definitions given in ISO 2531, ISO 10804 and the following apply.

ISO and IEC maintain terminological databases for use in standardization at the following addresses:

- ISO Online browsing platform: available at <https://www.iso.org/obp>
- IEC Electropedia: available at <http://www.electropedia.org/>

#### 3.1.1

##### **mechanical flexible joint**

flexible joint in which sealing is obtained by applying pressure to the gasket by mechanical means

### 3.1.2

#### **push-in flexible joint**

flexible joint assembled by pushing the spigot through the gasket into the socket of the mating component

### 3.1.3

#### **restrained joint**

joint in which a means is provided to prevent longitudinal separation of the assembled joint

### 3.1.4

#### **maximum design pressure**

##### **MDP**

$P_{MD}$

maximum operating pressure of the system or of the pressure zone fixed by the designer considering future developments and including surge

Note 1 to entry: It is the maximum pressure considering the design pressure and surge together, where:

- MDP is designated MDPa,  $P_{MDa}$ , fixed allowance for surge (secondary distribution networks);
- MDP is designated MDPC,  $P_{MDc}$ , surge is calculated (pumping & water mains).

[SOURCE: ISO 10802:2020, 3.6]

### 3.1.5

#### **system test pressure**

##### **STP**

$P_{ST}$

pressure to which a pipeline or a pipeline section is subjected for testing purposes

Note 1 to entry:

- $P_{ST} = 1,5 \times P_D$  (when  $P_{MD} \leq 10$  bar), or
- $P_{ST} = P_D + 5$  (when  $P_{MD} > 10$  bar)

where  $P_D$  is the design pressure.

Note 2 to entry: 1 bar is equivalent to 0,1 MPa.

[SOURCE: ISO 10802:2020, 3.7, modified — The original note 2 to entry has been replaced by a new one.]

### 3.1.6

#### **thrust force**

unbalanced hydrostatic force developed at the locations of a pipeline, changing diameter or direction

### 3.1.7

#### **bearing resistance**

passive pressure that is generated as the pipeline attempts to separate and move into the soil

### 3.1.8

#### **frictional resistance**

resisting force resulting from the interaction of the pipeline with the soil encountered on the project site and the pipeline laying conditions

### 3.1.9

#### **passive soil pressure**

maximum pressure that the soil imparts on a structure at the prescribed depth

Note 1 to entry: The passive soil pressure is dependent upon the compaction of the soil.



**3.1.10****restrained length**

minimum length to be restrained in order to balance *thrust forces* (3.1.6) and prevent disassembly or separation of the pipeline

**3.2 Symbols**

$A$	cross-sectional area of pipe, in $\text{m}^2$
$A_p$	surface area of the pipe bearing on the soil, in $\text{m}^2/\text{m}$
$C$	pipe-soil cohesion, equals $f_c C_s$ , in $\text{kN}/\text{m}^2$ ;
$C_s$	soil cohesion, in $\text{kN}/\text{m}^2$ (see Table 2)
$D_e$	outside diameter of pipe spigot, in m (see Annex A)
$f_c$	ratio of pipe-soil cohesion to soil cohesion (see Table 2)
$F_f$	unit frictional resistance, in $\text{kN}/\text{m}$
$F_s$	unit frictional force assuming 1/2 the pipe circumference bears against the soil, in $\text{kN}/\text{m}$
$(F_s)_b$	unit frictional force assuming the entire pipe circumference contacts the soil, in $\text{kN}/\text{m}$
$f_\phi$	ratio of pipe-soil friction angle to soil friction angle (see Table 2)
$h$	thrust block height, in m
$H$	depth of cover to top of pipe, in m
$H_c$	depth of cover to pipe centreline, in m
$K_n$	trench condition modifier (see Table 2)
$L$	minimum required restrained pipe length, in m
$N_\phi$	$= \tan^2 (45^\circ + \phi/2)$
$P$	system test pressure, in $\text{kN}/\text{m}^2$
$P_p$	passive soil pressure, in $\text{kN}/\text{m}^2$
$R_s$	unit bearing resistance, in $\text{kN}/\text{m}$
$T$	resultant thrust force, in kN
$\gamma$	backfill soil density, in $\text{kN}/\text{m}^3$ (see Table 2)
$W$	unit normal force on pipe $= 2 W_e + W_p + W_w$ , in $\text{kN}/\text{m}$
$W_e$	earth prism load $= \gamma H D_e$ , in $\text{kN}/\text{m}$
$W_p$	unit weight of pipe, in $\text{kN}/\text{m}$ (see Annex A)
$W_w$	unit weight of water, $\text{kN}/\text{m}$ (see Annex A)
$\theta$	bend angle, in degrees
$\delta$	pipe-soil friction angle, equals $f_\phi \phi$ , in degrees;

$\varphi$  soil internal friction angle, in degrees (see [Table 2](#))

$S_f$  safety factor (see [4.2](#))

## 4 Thrust restraint principles, calculation rules and general specification

### 4.1 Thrust forces

When underground or above-ground pipelines are in operation, unbalanced hydrostatic or hydrodynamic forces are developed at many locations under the internal pressure of the fluid in the pipeline, this is known as thrust forces. Unless the pipe joints in these areas are restrained against longitudinal movement, joint separation can result. These thrust forces are developed at locations where the pipeline changes either in diameter or in direction. Such locations include horizontal and vertical bends, tees, wyes, reducers, offsets, pipe bifurcations and valves.

At these locations the thrust forces are resisted with thrust blocks at the focus of the thrust force, or by installing a group of restrained joint pipes, in such a way that the unbalanced force is transmitted to the surrounding soil or pedestals (above-ground installation, without overstressing the pipeline wall and without subjecting the pipeline to joint separation).

The present standard studies and provides formulae which enable to balance thrust forces with the adequate quantity of restrained joint pipes.

Proper care shall be taken by the designer when chambers are installed within the restrained length of the pipeline.

The manufacturer's recommendations for selecting the type of restrained joint shall also be taken into account.

### 4.2 Calculation rules and general specification

The following parameters shall be taken into account: the cross-sectional area of the pipe ([Table A.1](#)), the pipeline changes generating the thrust force ([Clause 5](#)), the outside diameter of the pipe ([Table A.1](#)), the depth of cover of the pipe (see [Figure 6](#)), the characteristics of the soil surrounding the pipe and the trench backfilling methods ([Clauses 7](#) and [9](#)), the pipeline external coating system (bituminous, epoxy and acrylic paints or polyethylene encased pipe, PU and other extruded coatings - [Clause 8](#)).

The system test pressure (STP) of the pipeline is calculated from the maximum design pressure (MDP) and shall be used to estimate the thrust forces ([Clause 5](#)); and a safety factor of 2 is recommended.

For each pipeline changes and their combination, a specific formula is provided to calculate the length of pipes to be restrained. The list of common situations is provided in [Table 1](#) together with the subclause number:

**Table 1 — Type of common situation**

Description	Subclause number
Horizontal bends	<a href="#">10.1</a>
Vertical down bends	<a href="#">10.2</a>
Vertical up bends	<a href="#">10.3</a>
Tees	<a href="#">10.4</a>
Reducers	<a href="#">10.5</a>
Dead ends	<a href="#">10.6</a>
Encroaching restrained lengths	<a href="#">10.7</a>
Equal angle vertical offset ( $\theta$ )	<a href="#">10.8</a>

Table 1 (continued)

Description	Subclause number
Combined horizontal equal angle bends ( $\theta$ )	<a href="#">10.9</a>
Combined horizontal unequal angle bends	<a href="#">10.10</a>
Combined vertical equal angle offsets ( $\theta$ )	<a href="#">10.11</a>
Pipeline under obstruction	<a href="#">10.11.1</a>
Pipeline over obstruction	<a href="#">10.11.2</a>

### 4.3 Standard jointing systems offer no longitudinal restraint

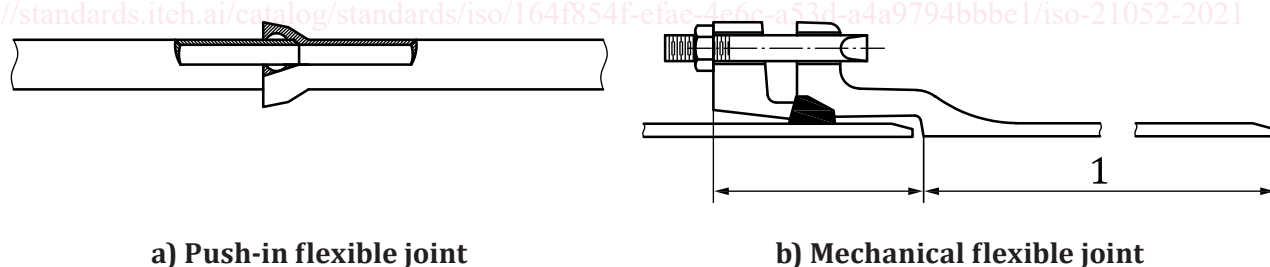
Ductile iron pipes and fittings are most often joined with push-in or mechanical flexible joints ([Figure 1](#)). Neither of these joints provide significant restraint against longitudinal separation other than the friction, between the gasket and the plain end of the pipe or fitting. Tests have shown that this frictional resistance of these joints are unpredictable. Thus, these joints should be considered as offering no longitudinal restraint for design purposes.

### 4.4 Restrained joint systems

The primary objective of the restrained joint system is to design a system to transmit the unbalanced forces to the surrounding soil without overstressing the pipeline wall and without subjecting the pipeline to joint separation. In order to accomplish the transfer of the unbalanced forces, the friction and passive resistance have been relied upon.

### 4.5 Length to be restrained

The length of the pipe, with restrained joints on each side of the focus of a thrust force, is calculated using the sum of the components of the unbalanced forces in the direction of the corresponding leg. The objective of the thrust restraint design using a restrained joint system is to extend the side of the fitting with inseparable joints so that the fitting can transmit the unbalanced forces to the surrounding soil.



#### Key

1 nominal laying length

Figure 1 — Push-in and mechanical flexible joints

### 4.6 Restrained design method

This document shows the method to calculate the quantum of thrust forces for the most common situations and the approaches to the design of restrained joint systems for balancing these forces. The suggested design approaches are conservatively based on accepted principles of soil mechanics.